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1	Modeling a Flexible Ring Net with the Discrete Element	

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24 **Abstract:** Flexible barriers have been proven to be effective measures for mitigating natural hazards, such as rockfalls, gravel flows and debris flows. This paper presents a new numerical ring 25 model based on the discrete element method (DEM) to simulate a flexible ring net. The Edinburgh 26 27 Bonded Particle Model is applied to create internal forces within a ring element. The mechanical behavior of a ring element was analyzed from measurements collected during quasi-static tensile 28 tests. The systematic calibration approach of this ring model is described in detail. Two reduction 29 factors related to the bond Young's modulus and the bond radius are proposed to effectively adjust 30 the bending and axial stiffnesses of the ring element. With calibrated DEM parameters from the 31 tensile tests, the ring model is validated by reproducing these tensile tests under different boundary 32 conditions. Finally, a three-dimensional DEM model is established for modeling the rockfall 33 impact on a flexible ring net. A comparison between the existing test data and simulation results 34 reveals that the new ring model can accurately reproduce the response of a flexible ring net under 35 both static and dynamic conditions. 36

Author keywords: Flexible barrier; Discrete element method; Bond model; Tensile test;
Rockfall.

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#### 40 Introduction

41 Flexible barriers are widely adopted in Hong Kong and many other regions and countries as 42 protection systems against natural hazards. A flexible barrier protection system for mitigating 43 rockfalls and debris flows consists of nets, steel ropes, brake elements and steel posts to transfer loads to foundations. In comparison with rigid barriers and other protective structures, flexible 44 45 barrier protection systems are lightweight, have less of an influence on the environment, provide a considerable reduction in construction time and are also easily maintained in remote areas. A 46 47 substantial amount of impact energy is dissipated through the sliding of nodes and large 48 elastoplastic deformation of ring nets, one of the most distinguishing features of flexible barriers. In 2012, the Geotechnical Engineering Office (GEO) of The Hong Kong Special Administrative 49 50 Region Government (HKSARG) produced a Discussion Note titled "Suggestions on Design 51 Approaches for Flexible Debris-resisting Barriers" (Kwan and Cheung 2012). However, the design methods and selection of design parameters in the Discussion Note have not yet been validated. It 52 53 is necessary to produce a comprehensive and recognized design standard for flexible barrier protection systems. 54

55 Over the past few decades, due to their low cost and reproducibility, numerical approaches have been developed as powerful tools for modeling flexible barriers under impact loadings of 56 57 either rockfalls or debris flows. Different ring models have been developed for applications in 58 flexible barriers (Nicot et al. 2001b; Coulibaly et al. 2017). Any type of parametric study is allowed by numerical methods; this flexibility helps to optimize a typical design. Two major approaches 59 have been developed for modeling flexible barriers. One is the classical finite element method 60 61 (FEM), which has been extensively utilized by engineers and researchers to mathematically model and numerically solve complicated structural, fluid, mechanical and electrical problems. It is 62

essentially a mathematical method dedicated to solving partial differential equations and is well 63 suited for modeling continuous materials. A commercial finite element code (Abaqus/Explicit), 64 featuring nonlinear geometrical, mechanical and contact behavior in the structural dynamic range, 65 was used to model the impact of falling rocks against common steel wire flexible meshes (Cazzani 66 et al. 2002). Gentilini et al. (2012) proposed a highly nonlinear, dynamic, three-dimensional model 67 68 that meets the requirements of major parameters in European guidelines (2008). Later, a few modifications were suggested in the model of barrier 3000 to enhance its cost-effectiveness and 69 on-site performance (Gentilini et al. 2013). Volkvein (2004) applied newly developed discrete 70 71 elements to the finite element software FARO to simulate ropes and net rings under the influence 72 of long distance slides and included friction. Moreover, Escallón et al. (2015) developed a model that accounts for contact interactions in flexible chain-link wire nets with loose connections. 73 Mentani et al. (2016) investigated the performance of a low-energy rockfall barrier in relation to 74 the bullet effect based on a finite element model. 75

76 On the other hand, a more recently proposed numerical method, the discrete element method (DEM), which was initially proposed by Cundall and Strack (1979), is also capable of modeling 77 78 the mechanical response of debris-resisting systems. It is the simplest discrete medium model; it 79 adopts a particle system composed of a large number of spherical particles, meeting Newton's second law of motion and employs a spring-damper model to describe the interaction between 80 81 particles. Bertrand et al. (2008, 2012) introduced a remote interaction model to simulate the 82 double-twisted hexagonal mesh and rockfall protection fences based on the DEM under European guidelines (2008). Li and Zhao (2018) used this remote interaction model to simulate the same 83 hexagonal-shaped wire mesh coupled with a debris mixture by using computational fluid dynamics 84 85 (CFD). Thoeni et al. (2013) further improved the remote interaction model by considering

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distortion of the wires and hexagons. Albaba et al. (2017) established a DEM model of granular flows impacting an elementary mesh by adopting a cylindrical element (Bourrier et al. 2013).

88 In summary, the FEM is based on continuum mechanics theory; thus, it is appropriate to use 89 this approach for modeling a continuous medium. However, specific algorithms are required to describe potential cracking or local failure, which is time consuming to implement. In contrast, 90 91 compared to the FEM, the DEM is particularly well suited to model mechanical systems in which 92 large relative displacements may occur during a loading process, owing to its inherent advantage 93 in describing the granular material with particle interactions. Large displacements and failures can 94 be easily simulated. Consequently, the DEM is suitable for modeling both flexible barriers and granular flowing materials as well as their interactions. In accordance with different materials, 95 96 geometries and connection types, flexible net systems vary. However, only few bond models have been employed to simulate a simple wire mesh. A circular ring net, as a typical protection structure, 97 has seldom been considered with the DEM due to its complex structure and special mechanical 98 99 response. There is a lack of a systematic calibration process to reproduce the ring behavior both under static and dynamic conditions. 100

This paper presents a newly developed ring model that uses the DEM to simulate the behavior 101 102 of a circular wire ring. This new ring model is developed from the Edinburgh Bonded Particle 103 Model (EBPM) (Brown et al. 2014) based on Timoshenko beam theory with limited simplifications and assumptions. The mechanical behavior of a wire ring in a quasi-static tensile 104 test is investigated using the new ring model. To determine the rigidity and bond strength of a wire 105 106 ring, a novel calibration approach on bond parameters is elaborated based on a parametric study 107 of tensile tests and experimental data from the literature. The model reliability is assessed by reproducing tensile tests carried out on a steel ring under different boundary conditions. Finally, 108

the capability of the new ring model is further tested to replicate the tests results of dynamic rockfall impact on a squared ring net. This approach enables the accurate modeling of both a circular ring net and granular materials as well as their interactions in the same computational framework.

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#### 114 Model Description

115 Normally, a typical flexible barrier protection system consists of various components, such as net, supporting cables, brake elements, posts, and anchors, among which the metal wire net (Fig. 1) 116 plays a vital role either in the mechanical behavior or in the failure mode of the entire system. A 117 118 metal wire net is composed of interconnected wire rings. Hence, the behavior of a wire ring, called 119 a ring element, under loading becomes a basic and fundamental issue. In this study, the DEM is 120 used to simulate a ring element in EDEM software. The EBPM (Brown et al. 2014) is applied to 121 the particles in contact to resist not only tensile and shear forces but also compressive forces and bending and twisting moments through the EDEM Application Programming Interface (API). 122

The EBPM, proposed by Brown at el. (2014), is based on Timoshenko beam theory that, unlike
Euler-Bernoulli beam theory, accounts for the effects of transverse shear deformation. The EBPM
provides a more accurate representation of true bending in a beam. Beams that are either short or
have large expected deflections are more accurately modeled by considering Timoshenko beam
theory.

To bond particles that are not physically in contact when using EBPM, it is necessary to set a contact radius larger than the physical radius of a particle, thus forming an overlap between two particles. Once the simulation time exceeds the bond time, the bond initialization procedure will be triggered. Particles will be bonded by a Timoshenko beam element, which links the centers of
the two particles, as illustrated in Fig. 2(a). With the increase in time step, the bond forces and
moments governed by the Timoshenko beam theory are updated:

(1)

134  $\Delta \mathbf{F} = \mathbf{K} \cdot \Delta \mathbf{u}$ 

where  $\Delta \mathbf{F}$  is the incremental force vector, which contains 12 force and moment increments at the two ends of the bond [Fig. 2(b)];  $\Delta \mathbf{u}$  is the displacement (rotation) vector, which contains 12 displacement and rotation increments at the two ends of the bond; and  $\mathbf{K}$  is a 12×12 tangential stiffness matrix, which is derived from the differential equations for beam deformation using the unit displacement theory for a Timoshenko beam. By adding all the increments since the first step of the computation, the total internal forces and moments  $\mathbf{F}$  can be obtained:

141 
$$\mathbf{F} = \sum \Delta \mathbf{F}$$
(2)

There are three strength criteria that determine the failure of bonds: the compressive  $\sigma_c$ , tensile  $\sigma_T$  and shear  $\tau$  strength criteria. If any of the maximum stresses exceed the limit values of strength, the bond will break. The maximum compressive stress  $\sigma_{Cmax}$ , tensile stress  $\sigma_{Tmax}$  and shear stress  $\tau_{max}$  are obtained by beam theory.

146 
$$\sigma_{C} < \sigma_{C\max} = -\min\left(\frac{F_{qx}}{A_{b}} - \frac{r_{b}\sqrt{M_{py}^{2} + M_{pz}^{2}}}{I_{b}}, \frac{F_{qx}}{A_{b}} - \frac{r_{b}\sqrt{M_{qy}^{2} + M_{qz}^{2}}}{I_{b}}\right)$$
(3)

147 
$$\sigma_T < \sigma_{T\max} = \max\left(\frac{F_{qx}}{A_b} + \frac{r_b\sqrt{M_{py}^2 + M_{pz}^2}}{I_b}, \frac{F_{qx}}{A_b} + \frac{r_b\sqrt{M_{qy}^2 + M_{qz}^2}}{I_b}\right)$$
(4)

148 
$$\tau < \tau_{\max} = \frac{M_{px}r_b}{2I_b} + \frac{4\sqrt{F_{py}^2 + F_{pz}^2}}{3A_b}$$
(5)

149 where  $A_b = \pi r_b^2$ ;  $I_b = \frac{\pi r_b^4}{4}$ .  $r_b$  is the disc radius of the bond.  $A_b$  is the cross-sectional area of the 150 bond.  $I_b$  is the second moment of area of the bond. p and q represent the two ends of the bond. 151 Although the bonds behave in a linear elastic brittle manner, with accurately calibrated rigidity 152 and strength, the new ring model is capable of exhibiting large deformation consistent with the 153 elastoplastic behavior that a metal wire ring typically experiences.

The bond elements describe the bond behavior within a ring element. For the nonbonded contact, such as the contact between the granular material and flexible ring net, the forcedisplacement relationship is determined by the Hertz-Mindlin contact law (Johnson 1987), which follows a spring-dashpot configuration in two directions, namely, normal and tangential (Fig. 3). The contact force is calculated by the sum of the normal force and the tangential force. Both forces include spring parts ( $F_{ns}$ ;  $F_{ts}$ ) and damping parts ( $F_{nd}$ ;  $F_{td}$ ).

160 The normal spring force  $F_{ns}$  is a function of normal overlap  $\delta_n$  between the particles in 161 contact and is expressed as

162 
$$F_{ns} = \frac{4}{3} E^* \sqrt{R^*} \delta_n^{\frac{3}{2}}$$
(6)

163 where  $E^* = \left(\frac{1-v_i^2}{E_i} + \frac{1-v_j^2}{E_j}\right)^{-1}$  and  $R^* = \frac{R_i R_j}{R_i + R_j}$ .  $E^*$  is the equivalent Young's modulus.  $R^*$  is the

164 equivalent particle radius.  $E_i$ ,  $R_i$ ,  $v_i$  and  $E_j$ ,  $R_j$ ,  $v_j$  are the Young's moduli, Radii and Poisson's 165 ratios of the two particles in contact. The normal damping force is given by

166 
$$F_{nd} = -2\sqrt{\frac{5}{6}}\beta\sqrt{S_n m^* v_n^{rel}}$$
(7)

167 where 
$$\beta = \frac{\ln e}{\sqrt{\ln^2 e + \pi^2}}$$
,  $S_n = 2E^* \sqrt{R^* \delta_n}$ , and  $m^* = \frac{m_i m_j}{m_i + m_j}$ .  $\beta$  is the damping ratio governed by the

168 coefficient of restitution  $e \cdot S_n$  is the normal stiffness.  $m^*$  is the equivalent mass.  $v_n^{\overline{rel}}$  is the normal 169 component of the relative velocity.  $m_i$  and  $m_j$  are the masses of each contact particle. The 170 tangential spring force  $F_{ts}$  is based on the tangential overlap  $\delta_t$  and is limited by the Coulomb 171 friction  $\mu$ . It is expressed as

$$F_{ts} = -S_t \delta_t \tag{8}$$

173 where 
$$S_t = 8G^* \sqrt{R^* \delta_n}$$
 and  $G^* = \left(\frac{1 - v_i^2}{G_i} + \frac{1 - v_j^2}{G_j}\right)^{-1}$ .  $S_t$  is the tangential stiffness.  $G^*$  is the equivalent

shear modulus. The tangential damping force is given by

175 
$$F_{td} = -2\sqrt{\frac{5}{6}}\beta\sqrt{S_t m^*} v_t^{\overline{rel}}$$
(9)

176 where  $v_t^{\overline{rel}}$  is the tangential component of the relative velocity.

177

### 178 Model Calibration

The quasi-static tensile test is one of the most effective ways to investigate the mechanical characteristics of a material. The bearing and deformation capacity of a ring element can be determined from such tests. Model calibration work is an essential procedure when applying the EBPM to a flexible ring net. Once the material and deformation parameters of a ring element aredetermined by the test results, full-scale impact tests on flexible ring nets can proceed.

# 184 **DEM Modeling of a Tensile Test**

A high-grade steel wire ring net is adopted in this paper. The tested wire ring was fabricated with Al/Zn coated wires to improve corrosion resistance to extend the service life. Each ring was obtained by wrapping a single 3 mm diameter steel wire 12 times. The tested ring had a 300 mm internal diameter and was held in place by three metal clips.

189 In engineering practice, each ring element in a flexible ring net connects to four neighboring 190 ring elements. Thus, there are four contact points on every ring element. A series of laboratory 191 tests were carried out on a servo-hydraulic universal testing machine at the Swiss Federal Institute 192 of Technology Zurich (ETHZ) (Grassl 2002). The tested ring element was mounted in a specially designed steel double plate device that can create different boundary conditions. The bolts were 193 194 fixed on the holes through bushings, and the ring element was positioned in the middle of the double plates. The top double plates moved at a velocity of between 0.2 mm/s and 1 mm/s, 195 depending on the experimental setup (Fig. 4). 196

The DEM modeling of a tensile test conducted in this paper is shown in Fig. 5 and is based on the same setup as that of the laboratory test in Fig. 4(a). The circular ring is discretized by bonding 100 particles to form a closed loop ring with the EBPM. Since a bond has the same circular crosssection as a wire ring, changes need to be made to the radius of the bonded particle and the bond to meet the requirements for different steel wire diameters. The cross-sectional area of a ring can be determined by the sum of the areas of the wires, but this approach will overestimate the bending stiffness of the ring. Thus, a reduction coefficient  $\chi$  was introduced to calculate the equivalent cross-sectional area ( $A_{eq}$ ) of a ring (Escallón and Wendeler 2013):

$$A_{eq} = \chi A_{eq} = \chi \pi r_w^2 n_w \tag{10}$$

where  $r_w$  is the radius of a wire and  $n_w$  is the number of windings. In this study, an equivalent 206 207 particle radius (4.5 mm) is calculated as a reference case regardless of the value of  $\chi$ . The axial and bending stiffnesses are adjusted by the bond Young's modulus and bond disc radius, which 208 will be discussed in the following sections. The number of particles is a significant issue in the 209 DEM simulation. As the bond is straight, as more particles are generated, the results of the ring 210 simulation will become more accurate. The deformation of the DEM model with 100 bonded 211 particles will be used to represent the curvature of a ring. The computational efficiency would 212 213 decrease if more particles are used. Furthermore, the coefficient of bond strength variation is set to 0 to ensure that all the bonds in a ring have the same bond strength. 214

In addition to the bond parameters that are directly related to the tensile failure of the ring 215 216 element, other values of the numerical and nonbonded contact parameters for the reference case are listed in Table 1. The global damping coefficient in the EBPM is defined as the damping 217 applied to the particles through the equations of motion so that energy is dissipated in every particle 218 in the system. A damping coefficient of 0.95 defines the test as a quasi-static problem, which 219 220 compensates for the instability of the model induced by a relatively large loading rate (5 mm/s). Hence, a larger time step (7.87e-07) can be employed to improve the computational efficiency, 221 and the impact of the dynamic effect can be minimized (Cho et al. 2007). 222

Fig. 6 depicts the four stages of the quasi-static tensile test of a 300 mm diameter ring elementfixed at 4 vertices of contact:

(a) Bonds are generated between particles at the bond time. Then, the steel plates move at a certain
velocity, and forces act on the ring through the four bolts.

(b) The ring deforms nonlinearly due to the change in geometry, which represents the bendingbehavior of the ring.

(c) The ring is stretched into a rectangular shape, and geometric nonlinearity no longer occurs. The
 forces increase faster in this stage than they did in the second stage. The bearing capacity is
 determined mainly by the normal force in the ring.

(d) The deformation of the ring increases with loading until a bond is broken and the ring fails.

# 233 Parametric Study on the Bond Parameters of the Tensile Test Simulations

For the purpose of simulating a specific ring element, the relationship between the DEM parameters and failure criteria should be studied. The influences of the three most important bond parameters, bond Young's modulus, mean bond tensile strength and bond radius multiplier, are investigated by analyzing the force-deformation characteristics, breaking load and maximum deformation of a ring element. Other nonbonded contact parameters are not considered to have a significant influence on the failure mode of a ring. The ranges of the values used in the parametric study are shown in Table 2.

### 241 Effects of the Bond Young's Modulus

Fig. 7(a) plots the effects of the bond Young's modulus ( $E_b$ ) on the load-displacement relationship of the ring. The range of the bond Young's modulus is from 40 GPa to 200 GPa. With the increase in the bond Young's modulus, the bonds become stiffer. Thus, to produce the same deformation of the ring, a greater applied loading force is needed to resist the induced bond forces. For a ring with a 200 GPa bond Young's modulus, the ring breaks during the shape-changing stage. Fig. 7(b
and c) show the respective relations between the bond Young's modulus and the breaking load and
maximum elongation at failure. Because the value of the bond Young's modulus modifies the
stiffness of the bond, considerable elongation occurs when the bond Young's modulus decreases
to 40 GPa, which is more than three times that when the bond Young's modulus is 200 GPa. A
significant increase in loading is observed accordingly.

### 252 Effects of the Mean Bond Tensile Strength

253 Fig. 8 shows the effects of the mean bond tensile strength  $(T_{e})$  on the load-displacement 254 relationship, breaking load and maximum elongation of a ring. The tested mean bond tensile strengths range from 4 GPa to 20 GPa and the bond Young's modulus is 120 GPa. The load-255 displacement curves of the 5 tests nearly overlap due to the identical stiffness of the rings. 256 Generally, the breaking load and maximum elongation increase with the mean bond tensile 257 strength. Linear relations are observed after the ring is stretched into a rectangular shape. During 258 this stage, the bond force is mainly affected by the properties of the material itself, rather than the 259 geometrical deformation. 260

#### 261 Effects of the Bond Radius Multiplier

262 The bond radius multiplier  $\lambda$  is a reduction coefficient of the bond radius that can influence the 263 bond stiffness and strength:

$$r_b = \lambda \min(r_A, r_B) \tag{11}$$

where  $r_A$  and  $r_B$  are the radii of particles A and B, respectively. Fig. 9 demonstrates the effects of the bond radius multiplier on the load-displacement relationship, breaking load and maximum elongation of a ring. The range of the bond radius multiplier is from 0.5 to 0.9. The bond radius multiplier has a significant influence on the breaking load because it changes the cross-sectional area of the bond. For a given mean bond tensile strength, the bond force and stiffness decrease with the decreasing cross-sectional area. Therefore, the bond becomes more ductile as the bond radius multiplier decreases. The maximum elongation will slightly increase. However, the effects of the bond radius multiplier on the maximum elongation are less significant in contrast with the results of varying the bond Young's modulus.

### 274 Rigidity and Bond Strength Determination of a Ring Element

The reference case of the quasi-static tensile test reveals that the bending and axial stiffnesses of the ring dominate the ring element behavior in different stages. Notably, the equivalent particle radius ( $r_{eq}$ ) of 4.5 mm was determined based on the actual cross-section of a circular ring and the particle number of a ring element in the simulation. The equivalent particle radius did not equal the equivalent bond radius ( $R_{eq}$ ), which affects the stiffness of a ring element. The bending stiffness can be expressed as

281 
$$k_{bending} = EI = E \frac{\pi R_{eq}^4}{4}$$
(12)

282 where  $R_{eq} = \sqrt{\frac{A_{eq}}{\pi}} = \sqrt{r_w^2 n_w} = \lambda r_{eq}$ . *E* is the steel Young's modulus (200 GPa). The axial stiffness

283 can be expressed as

$$k_{axial} = EA = E\pi R_{ea}^2 \tag{13}$$

Grassl (2002) found that the theoretical axial stiffness is five times the rigidity in the tensile
test for wire rings with 7, 12 and 19 turns. This should be explained by the different utilization of

the individual wires because the axial stiffness k decreases with the increased loading for a single wire tensile test. The overall axial stiffness of a wire ring is the sum of the individual wires connected in parallel. Therefore, the sum of the axial stiffness should be less than  $n_w k$  and the axial stiffness can be rewritten as

291 
$$k_{axial} = \frac{1}{5} E \pi r_w^2 n_w \tag{14}$$

As mentioned in the proceeding sections, the bending stiffness should be reduced to eliminate the overestimation of the area moment of inertia by decreasing the equivalent bond radius. However, according to Eq. (13), the axial stiffness will decrease with the bond radius; this result is consistent with the parametric study. The reduction of the bond radius cannot change the bending and axial stiffnesses independently. Based on the equations and findings from the parametric study, the bond Young's modulus can likewise influence the bending and axial stiffnesses; thus, two coefficients  $\gamma$  and  $\eta$ , are inserted into Eq. (12) and Eq. (13):

299 
$$k_{bending} = \eta E \frac{\pi (\gamma R_{eq})^4}{4}$$
(15)

$$k_{axial} = \eta E \pi (\gamma R_{eq})^2 \tag{16}$$

301 where  $\gamma R_{eq} = \lambda r_{eq}$ . Substitution of Eq. (14) into Eq. (16) gives the reduction factor of the axial 302 stiffness:

$$\eta \gamma^2 = \frac{1}{5} \tag{17}$$

1

304  $\gamma$  and  $\eta$  can be easily obtained by Eq. (17) once the reduction factor of bending stiffness 305  $(\eta\gamma^4)$  is known.  $\eta\gamma^4$  can be determined by assuming  $\eta = 1$  and testing different values of  $\lambda$  until 306 reaching a good agreement with the experimental bending response.

The experiment (Grassl 2002) related to the ring with 4 vertices of contact is employed for 307 model calibration. Finally, the bond Young's modulus, bond tensile strength and bond radius 308 multiplier were calibrated by fitting the experimental and DEM results (Fig. 10). There is a slight 309 310 difference between these sets of results in the final stage of the test. The fluctuation in the physical test may be explained by the incompatible deformation of the individual wires and the slip between 311 the wire and clips, whereas the DEM model cannot account for these effects. The results are also 312 313 compared to the FEM predictions by Grassl (2002). The FEM ring element was developed and implemented in the FARO finite element program. The resistance due to bending and traction for 314 any boundary conditions are detected with 8 bar and 8 spring elements. Our new ring model 315 exhibits a better fit in the tensile regime than that of the FEM result. The values of the DEM 316 parameters of the ring tensile test are listed in Table 3. 317

318

#### 319 Model Validation

### 320 Modeling of Tensile Tests with Different Boundary Conditions

To validate the calibrated parameters of the ring model, two additional DEM tensile tests of the same 300 mm diameter ring element with 2 and 3 vertices of contact were conducted. The obtained load-displacement relationship is compared to the corresponding experimental data and FEM results (Grassl 2002), as shown in Fig. 11(a and b). The bending stiffness is slightly underestimated for both cases, mainly attributable to the constant change in the cross-sectional area of the ring in the bending regime. Different area moments of inertia shall lead to different bending stiffnesses.
In addition, because the ring model is not able to consider plastic deformation in the bending
regime, the ring will enter the tensile regime slightly earlier than in the experiment.

Furthermore, a more complex loading condition with a group of 8 interconnected wire rings [Fig. 4(d)] was considered to account for the ring net response. The DEM predictions agree with the experimental results, matching the net response in both the bending and tensile regimes [Fig. 11(c)]. The ring model exhibits a better performance in describing the axial resistance of the group wire rings than that of the FEM model. Sudden drops in loading occur in the tensile regime due to the sliding motion between the wire rings in contact.

Owing to the proper calibration of parameters and the inherent advantage of the ring model in dealing with geometrical nonlinearity, the model response is quite agreeable to the results of the experiment in various loading configurations. Therefore, further study of the entire flexible ring net can be carried out on the basis of the new ring model.

# 339 Modeling of Rockfall Impact Tests on a Flexible Ring Net

Based on the tensile test results of the ring element, the overall performance and dynamic response 340 341 of the flexible ring net is evaluated by a falling rock impact test. The experimental test was 342 conducted at the Swiss Federal Rockfall Test Site in Walenstadt (SG), Switzerland (Grassl 2002). The test ring net employed was composed of the same 300 mm diameter rings as the tensile test in 343 344 the above section. Each ring is linked with four others. A total of 180 ring elements constitute the net with dimensions of  $3.9 \text{ m} \times 3.9 \text{ m}$ . The net was fixed to the top of a rigid steel frame by 345 346 shackles. A boulder of mass 825 kg was lifted to 3 m above the net and dropped with 24.3 kJ of kinetic energy [Fig. 12(a)]. In parallel to the experiment, an FEM model (Grassl 2002) was built 347

to simulate the rockfall tests. The aforementioned FEM ring element was used to describe the wire
net behavior. For each ring, 8 nodes are used to connect the bar elements and detect the contact
points of adjacent rings and rockfall material.

351 By making use of the experimental data, a full-scale three-dimensional DEM model was established to validate the test and obtain a better understanding of the interaction between the 352 353 rockfall and flexible ring net. For a real flexible ring net, the connection points between the rings 354 is moveable, which allows the ring elements to slide during the impact. To accurately simulate the 355 behavior of the ring net, 180 ring elements formed by 18000 particles were generated in three-356 dimensional space through the EDEM API [Fig. 12(b)]. In the stage of particle generation, each ring was rotated to an appropriate degree to avoid contact with the other rings. Bonds were created 357 between only adjacent particles within each ring. In addition, boundary conditions represented by 358 359 rigid steel rings were added to the outermost rings, allowing rotation and sliding of the rings. A spherical boulder was released from the same height to the net as that in the physical test. Moreover, 360 361 since the ring net is influenced by gravitational acceleration, it is necessary to consider the initial sag of the net by allowing the net to fall to a free state before boulder impact. 362

Fig. 13 displays the three-dimensional lateral view of the net deformation developed during 363 the impact test at different time points. Fig. 14(a) depicts the evolution with time of the rockfall 364 acceleration, in comparison with the test data and FEM results. Time 0 corresponds to the first 365 contact of the boulder with the net. This result indicates a good correlation between the 366 experimental results and the DEM simulation. The acceleration of the boulder in the DEM model 367 368 increases slightly faster than that in the experiment from time 0.1 s to time 0.15 s. The distinct 369 initial states of the interconnected wire rings between the model and experiment may lead to this difference, although the initial sag of the net was taken into account. In accordance with the 370

calibration procedures, impact tests were repeatedly performed for different sizes of wire rings with 7 and 19 turns to further evaluate the reliability of the ring model [Fig. 14(b and c)]. It is observed that the maximum acceleration of the boulder decreases with an increasing number of turns. In contrast to the FEM results, the developed DEM ring model with less oscillations is more accurate for predicting the dynamic response of a flexible ring net under rockfall impact.

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# 377 Summary and Findings

378 In this paper, the Edinburgh Bonded Particle Model (EBPM) is employed to establish a new 379 numerical ring model for a steel wire ring. The effects of the bond parameters of this ring element are investigated by means of a series of parametric studies in modeling quasi-static tensile tests. 380 381 The rigidity and bond strength of the ring model are carefully calibrated. In addition, the ring model is validated by comparing computed values with test data. A three-dimensional discrete 382 383 element method (DEM) model for a flexible ring net is developed and applied for simulating the 384 deformation of a flexible ring net under the impact of a falling rock. Based on these works, the following findings are obtained: 385

1. The new numerical ring model for a steel wire ring (a ring element) based on the EBPM issuitable for modeling a steel wire ring.

2. The calibration of bond parameters in this new ring model can be easily performed by comparingthe modeling values with available test data.

390 3. The three-dimensional DEM flexible ring net based on our new ring model can reproduce the
behavior of a ring net barrier under rockfall impact, allowing engineers to employ this model as
a design tool of this typical structure. Furthermore, the model also has great potential in the

study of the interaction between flexible barriers and other natural hazards, including gravel
flows and debris flows in the same framework due to the inherent merits in describing particle
interactions by the DEM. The proposed DEM model will enable parametric studies on the
interaction between flexible barriers and natural hazards under more complicated impact loading
conditions.

4. Compared with finite element modeling of a ring net, any simplified fine-mesh modes composed
of truss or spring elements are not necessarily established in a discrete element model. It is more
accurate to describe the behavior of both a single ring element and an entire flexible ring net
under impact loadings with the DEM than the FEM. Other deformable components, such as
steel cables and brake elements, can be added into our DEM model to evaluate the performance
of a whole flexible ring net barrier system under impact loadings.

It is recommended that large-scale physical model impact tests related to rockfalls and debris flows be executed to compare and validate the proposed numerical DEM model with new ring elements in the future.

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DEM Parameters	Values
Bond	
Bond Young's modulus (GPa)	120
Bond Poisson's ratio	0.3
Compressive strength (GPa)	16
Tensile strength (GPa)	16
Shear strength (GPa)	10
Coefficient of variation of the strength	0
Bond radius multiplier	0.8
Non-bond	
Particle radius (mm)	4.5
Particle contact radius (mm)	8
Particle density (kg/m <sup>3</sup> )	7800
Particle Young's modulus (GPa)	200
Particle Poisson's ratio	0.3
Coefficient of static friction	1
Coefficient of rolling friction	0
Coefficient of restitution	0.5
Numerical	
Loading rate v (m/s)	0.005
Global damping coefficient	0.95
Time step (s)	7.87e-07

 Table 1. Values of DEM Parameters for the Reference Case of a Quasi-Static Tensile Test

Parameters	Refence	Min	Max
Bond Young's modulus (GPa)	120	40	200
Mean bond tensile strength (GPa)	16	4	20
Bond radius multiplier	0.8	0.5	0.9

**Table 2.** Parametric Study on Bond Parameters of a Quasi-Static Tensile Test

Parameters	Values
Bond Young's modulus (GPa)	292
Bond Poisson's ratio	0.3
Tensile strength (GPa)	41
Shear strength (GPa)	12
Coefficient of variation of the strength	0
Bond radius multiplier	0.43

**Table 3.** Calibrated Bond Parameters for a 300mm Flexible Ring Element







































(a)

(b)

 $\geq$ 

0







# **Figure Caption List**

Fig. 1. Flexible metallic wire ring net in Hong Kong

Fig. 2. Two particles bonded by a Timoshenko beam element

Fig. 3. Schematic of the Hertz-Mindlin contact model

**Fig. 4.** Quasi-static tensile tests with 300 mm diameter steel ring elements: (a) 4 vertices of contact; (b) 3 vertices of contact; (c) 2 vertices of contact; (d) a group of 8 wire rings

Fig. 5. The DEM model of a quasi-static tensile test

Fig. 6. Four stages of the reference case tensile test

**Fig. 7.** Effects of the bond Young's modulus on (a) load-displacement relationship; (b) breaking load; (c) maximum elongation

**Fig. 8.** Effects of the mean bond tensile strength on (a) load-displacement relationship; (b) breaking load; (c) maximum elongation

**Fig. 9.** Effects of the bond radius multiplier on (a) load-displacement relationship; (b) breaking load; (c) maximum elongation

**Fig. 10.** Comparison of DEM simulation values with FEM and experimental results of a tensile test with a 300 mm diameter steel ring element (4 vertices of contact)

**Fig. 11.** Comparison of DEM simulation values with FEM and experimental results of a tensile test with a 300 mm diameter steel ring element: (a) 3 vertices of contact; (b) 2 vertices of contact; (c) a group of 8 wire rings

**Fig. 12.** The rockfall impact test on a flexible ring net: (a) setup of physical model test; (b) the DEM model

Fig. 13. The three-dimensional and lateral view of net deformation for the impact test

**Fig. 14.** Comparison of DEM simulation values with FEM and experimental results of the flexible ring net under the rockfall impact: the evolution with time of the rockfall acceleration: (a) 12 turns per wire ring; (b) 7 turns per wire ring; (c) 19 turns per wire ring