Harnessing impact-induced cracking via stiffness heterogeneity 3 Ji Lin^{1,2,3}, Yujie Xie¹, Manqi Li², Jin Qian², Haimin Yao^{1,*} ¹Department of Mechanical Engineering, The Hong Kong Polytechnic University, Hung Hom, Kowloon, Hong Kong SAR, China 6 ²Department of Engineering Mechanics, Key Laboratory of Soft Machines and Smart Devices of Zhejiang Province, Zhejiang University, Hangzhou 310027, China ³Piezoelectric Device Laboratory, Faculty of Mechanical Engineering & Mechanics, Ningbo University, Ningbo 315211, China 11 ^{*}To whom correspondence should be addressed, E-Mail: mmhyao@polyu.edu.hk (H. Yao) **Abstract** Mechanical heterogeneity refers to the spatial inhomogeneity of the mechanical properties in materials, which is a common feature of composites consisting of multiple distinct phases. Generally, the effects of mechanical heterogeneity on the overall properties of the composites, such as stiffness and strength, are thought to follow the rule of mixture. Here, we investigate the cracking behavior of composite plates under impact and found that the rule of mixture may break down in describing the cracking resistance of composites with high stiffness heterogeneity. Our results show that the resistance of a composite plate, which consists of two phases of distinct stiffnesses, against dynamic cracking strongly depends on the hybridizing manner of the two phases. When the stiff phase is dispersed in the compliant matrix, the resulting composite plate exhibits superior cracking resistance compared to the monolithic plates made of either phase. In contrast, if the compliant phase is dispersed in the stiff matrix, the resulting composite plate displays reduced cracking resistance and thus higher absorption of the impact energy as compared to the monolithic controls. Our work provides an approach to harnessing the dynamic fracture by controlling the stiffness heterogeneity, which would be of great value to the design and fabrication of the protective armors and energy-absorbing shields. **Keywords:** Mechanical heterogeneity, Ballistic cracking, Crack-inclusion

interactions, Structure-property relationship, Protective armors

1. Introduction

 The mechanical properties and behavior of solid materials are determined not only by their chemical compositions but also by the way of how these compositions are bonded together in space, namely the so-called structure. Revealing the structure- property relations and then applying them to direct the design and manufacturing of new materials is an everlasting topic in materials science and engineering. As a typical structural attribute, mechanical heterogeneity refers to the spatial variation and inhomogeneity of the mechanical properties (*e.g.*, elasticity and plasticity) in materials. It is commonly observed at multiple length scales in materials consisting of constituent phases with distinct properties such as engineering composites [1-4] and natural biological materials [5-9]. Mechanical heterogeneity has been shown to play an important role in determining the overall mechanical properties of the composite materials. Normally, materials with high heterogeneity are believed to have inferior mechanical properties in comparison to the homogeneous counterparts made of similar chemical compositions. This is because the mechanical mismatch between the distinct building phases, upon external loading, tends to cause stress concentration and strain localization near the phase interfaces. Nevertheless, sometimes heterogeneity was also found to benefit the mechanical properties of the materials. For example, the heterogeneous-structured metallic materials achieved superior mechanical properties that are not accessible to conventional homogeneous counterparts [10]; a "brick-and- mortar" structure with high heterogeneity could effectively suppress the crack-induced stress intensification and fortify the flaw tolerance of nacreous composites [11]; a 10- 15% micro- and nanomechanical heterogeneity was proved an optimal scheme to promote the ductile behavior of bones in nano- and microscale [12]. Particularly, the heterogeneity of plasticity at nano length scale in bones was demonstrated to promote energy dissipation during plastic deformation [13]. Moreover, heterogeneity in yield strength was found to benefit the strength-ductility synergy in metals [14].

 In addition to plasticity, heterogeneity in stiffness, which is characterized by the elastic modulus, was found to influence the fracture behavior of a material. Related

 studies can be traced back to the investigations of the effect of an interface between two dissimilar elastic materials on the propagation tendency of a crack. It is found that the interface between dissimilar materials may enlarge or suppress the SIF of a crack in the front of it, depending on the stiffness heterogeneity across the interface. When the crack is situated on the compliant side of the interface and heads to the stiff side, the interface would suppress the SIF at the crack tip; in contrast, if the crack is situated on the stiff side of the interface and heads to the compliant side, the interface would enlarge the SIF at the crack tip [15-17]. This conclusion still holds when a crack embedded in an infinite elastic medium approaches to a circular inclusion with distinct elastic modulus, which is a basic physical picture of fracture in composites with heterogeneous stiffness. Tamate [18] and Atkinson [19] analytically solved the SIF at the crack tips under the influence of inclusion with different moduli and distances away from the crack. For 74 more complicated cases, such as a crack under mixed-mode loading [20], the influence 75 of interfacial strength between the inclusion and matrix [21], finite element method (FEM) was adopted to evaluate the SIF of a stationary crack in the presence of an 77 inclusion. With the aid of the extended finite element method (XFEM), Jiang et al. [22, 23] found a similar effect of inclusions on the SIF of a dynamic crack tip. Tran and Truong [24] improved the smoothness of the stress and strain fields for the crack growth problem in composite material by incorporating XFEM with a twice interpolation 81 method. The effect of inclusion on the SIF of a dynamic crack has been experimentally 82 verified by the photoelasticity technique [25]. As a special inclusion with zero stiffness, a hole was also found to attract a crack propagating nearby [26], which is qualitatively in accord with previous studies. Further extensions were made to the studies on the effects of the heterogeneous interface on crack deflection and kinking [27, 28] and out- of-plane excursions of cracks [29]. Utilizing stiffness heterogeneity, researchers have effectively enhanced the fracture toughness of composites [30-32].

 Although the loading and the geometries of the models in the above works were simplified, the revealed phenomena implied that stiffness heterogeneity might have a similar effect on impact-induced cracking and therefore can be applied to harness

 impact fracture. To verify this hypothesis, in this work we firstly carry out a systematic computational study on the effects of stiffness heterogeneity on the cracking caused by impact. Our study starts from the effect of circular inclusions on the growing tendency of a pre-existing crack under a static load. Then, the discussion is extended to the dynamic cracking in inclusion-matrix composites caused by ballistic impact. Two types of composites with complementary hybridizing schemes are studied: stiff inclusions embedded in a compliant matrix (S@C) and compliant inclusions embedded in a stiff matrix (C@S). Parametric studies on the effects of inclusion size, stiffness, volume fraction, inter-inclusion spacing, and distribution pattern are carried out, followed by the experimental verification of the effect of stiffness heterogeneity on cracking resistance. Finally, the paper is concluded after discussing the synergetic effect of the 102 S@C and C@S composites under ballistic impact. The results obtained in the present study are believed to serve as a general guide for the development of anti-impact materials and protective shields.

2. Interference of stiffness heterogeneity with SIF of a static crack

 To quantify the effect of stiffness heterogeneity on the propagation tendency of a crack, an idealized finite element (FE) model is constructed (ABAQUS/Standard, Dassault Systèmes), in which a square plate (edge length: 2*D*) with pre-existing cracks contains a circular inclusion of radius *R* at the center (see Fig. 1(a)). Periodic boundary conditions are applied on both lateral sides of the model. That is, the model depicts a representative volume element (RVE) of a periodic structure with a period of 2*D*. The thickness and the period of the model are taken as 5 mm and 100 mm, respectively. The inclusion and the matrix are assumed perfectly bonded. Young's modulus and Poisson's 115 ratio of the matrix are prescribed as $E_M = 10$ GPa and $v_M = 0.3$, respectively. Eight- node linear brick elements with reduced integration (C3D8R in ABAQUS) are employed except in the region around the crack tip, where six-node linear triangular prism elements are applied to improve the accuracy in describing the singular stress 119 field with square-root singularity. A uniform tensile stress σ_0 is applied along the 120 direction perpendicular to the surface of the pre-existing cracks. The SIF (K_I) is derived 121 from $K_I = \sqrt{JE_M}$, where *J* is the calculated J-integral around the crack tip [33]. Due to the symmetry of the model, we do not distinguish the two cracks in the model when discussing the SIF in the following.

 Figure 1. (a) The RVE model applied in finite element analysis; (b) Meshing of the model and enlarged view at the crick tip. Computed variation of the stress intensity factor (normalized) at the crack tip with the crack length (normalized) for inclusion- matrix composites with inclusions of (c) different stiffnesses, and (d) different sizes. 129 The inclusion radius in (c) is taken as $R = 0.33D$ and the inclusion modulus in (d) is 130 taken as $E_I = 10 E_M$; (e) Snapshots of the von Mises stress field (normalized by the load 131 σ_0) at three selected moments as indicated in (d). α : $a/(D - R) = 0.48$; β : $a/(D - R)$ 132 R) = 0.75; γ : $a/(D - R) = 0.98$.

Fig. 1(c) shows the calculated SIF as a function of crack length for three cases

135 with inclusions of different stiffnesses: $E_I = 10E_M$, $E_I = E_M$ and $E_I = 0.1E_M$. For 136 the homogeneous case with $E_I = E_M$, the analytical solution to the SIF exists and is given by [34]

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$$
K_{\rm I} = \sqrt{\frac{2D}{\pi a} \tan \frac{\pi a}{2D}} \sigma_0 \sqrt{\pi a}
$$
 (1)

 The consistency between such an analytical solution and our calculated results shown in Fig. 1(c) validates the calculated SIF in other cases. Comparison between the three cases shown in Fig. 1(c) confirms that stiffness heterogeneity affects the SIF, which 142 determines the growing tendency of a crack. Particularly, stiff inclusion $(E_I > E_M)$ suppresses the SIF and therefore resists the crack propagation, while compliant 144 inclusion $(E_I < E_M)$ enhances the SIF and therefore facilitates the crack propagation.

 We further studied the size effect of the inclusions. Fig. 1(d) shows the variation of the calculated SIF with the crack length for cases with inclusions of different sizes and 147 given modulus $E_1 = 10E_M$. In these three cases, the SIF exhibits a similar variation trend with the increasing crack size. That is, it increases initially and then decreases with the increase of the crack size. Such variation of SIF can be further visualized from the stress (von Mises) field near the crack tip at three representative moments, as shown in Fig. 1(e). Comparison between these three studied cases indicates that larger inclusion imposes higher suppression on the SIF.

3. Effect of stiffness heterogeneity on ballistic cracking

3.1 Simulation of ballistic impact

 To extend our study from static cracks to dynamic fracture, we simulated the ballistic impact process of a spherical projectile (radius: 2 mm) on composite plates (96 158 mm \times 96 mm \times 1 mm) composed of a stiff (S) phase and a compliant (C) phase, as shown in Fig. 2(a). The modulus of the stiff phase is taken as 10 times that of the compliant phase. To mask off the possible influence of the factors other than stiffness on the cracking behavior, the other properties of these two phases, such as density, 162 Poisson's ratio, fracture strength, and fracture energy are assumed the same. Both 163 compliant phase and stiff phase are assumed as brittle materials with fracture energy $\Gamma = 10 \text{ J} \cdot \text{m}^{-2}$, which is close to the fracture energy of glass (Fig. S1). As we are concerned about the cracking behavior in the composite plates, the material of the projectile is simply assumed as steel with linear elasticity. The detailed material properties of each phase and the projectile are summarized in Table 1. Two complementary hybridizing schemes for the composite plates are considered. One is to 169 embed stiff inclusions into the compliant matrix (denoted as $S(\widehat{a}|C)$), and the other is to 170 embed compliant inclusions into the stiff matrix (denoted as $C(\partial S)$). For comparison, monolithic plates of the same dimensions composed of compliant phase or stiff phase only are applied as the control cases. Both plates and projectiles are modelled with four- node shell elements (S4R in ABAQUS) with a thickness of 1 mm. Simply-supported boundary conditions are applied on four vertexes of the plate. The initial velocity of the 175 projectile is taken as $8 \text{ m} \cdot \text{s}^{-1}$ perpendicularly towards the plate center. The friction coefficient between the projectile and plate is set as 0.2. The crack initiation and propagation in the plate upon the ballistic impact by the projectile are simulated by the element deletion technique (brittle cracking material model in ABAQUS/Explicit) [35], whereby an element is "deleted" by gradually reducing its stiffness to zero when its maximum principal stress reaches the prescribed fracture strength. During the crack opening, linear stress reduction is introduced to describe the stress variation of the element, which consumes the fracture energy of the material.

183 **Table 1**. Material properties adopted in FE simulations

Properties	Stiff Phase	Compliant Phase	Projectile
Density, ρ (g cm ⁻³)	1.3	1.3	7.8
Young's modulus, $E(GPa)$	2	0.2	210
Poisson's ratio, ν	0.3	0.3	0.3
Fracture strength, $\sigma_f(MPa)$	2	\mathcal{D}	
Fracture energy, Γ (J m ⁻²)	10	10	

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185 Figure 2. (a) Schematic showing the simulation model of a projectile perpendicularly 186 impacting on a plate. (b-e) Snapshots of the calculated cracking process at time $t = 0.5$, 187 1, and 2 ms: (b) monolithic compliant plate, (c) monolithic stiff plate, (d) $S@C$ 188 composite plate, and (e) $C(a)S$ composite plate. Von Mises stress distributions at $t = 2$ 189 ms shown at the bottom row is normalized by the fracture strength (σ_f) of the materials. 190 The radius of the inclusions in (d) and (e) is 4 mm, and the inter-inclusion spacing 191 (center-to-center) is 12 mm.

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193 **3.2 Results and discussions**

194 The snapshots of the cracking process of four plates upon ballistic impact are 195 displayed in Fig. 2. For the two monolithic cases, it can be seen that cracks propagate 196 faster and longer in the pure stiff plate (Fig. $2(c)$) than in the pure compliant plate (Fig. 2(b)), although the fracture strength (σ_f) and fracture energy (Γ) of both materials are taken the same. This can be attributed to the lower toughness $(\frac{1}{2})$ 198 taken the same. This can be attributed to the lower toughness $(\frac{1}{2}\sigma_f^2/E)$ of the stiff 199 material. If the fracture strength of the stiff material is increased to such a value that its 200 toughness would be equal to that of the compliant material, both monolithic plates will 201 exhibit similar cracking configurations under impact (see Fig. $S2(a, b)$). In the $S@C$ 202 composite plate (Fig. 2(d)), the ballistic cracks emitted from the impact point 203 (compliant region) are deflected or blocked by the inclusions (stiff phase), and thereby 204 crack propagation are constrained in a limited region; in contrast, in the $C@S$ plate (Fig. 205 $2(e)$), the ballistic cracks initiating from the impact point (stiff region) penetrate and 206 pass through the inclusions (compliant phase). Consequently, crack propagation is 207 highly exacerbated in the C@S plate as compared to the $S@C$ composite plate and two monolithic controls. Similar result is observed when the toughness $(\frac{1}{2})$ 208 monolithic controls. <mark>Similar result is observed when the toughness $(\frac{1}{2}\sigma_f^2/E)$ of both</mark> 209 phasic materials are taken as the same (see Fig. $S2(c, d)$). The contrast of different plates 210 in resisting dynamic cracking shown in Fig. 2 ($S@C >$ monolithic $>C@S$) also agrees 211 with the effect of the heterogeneous interface on the SIF of a stationary crack as shown 212 $\frac{1}{\ln \text{Fig. 1}}$. In both composite plates, the stress level in the compliant phase is lower than 213 that in the stiff phase, implying that the ballistic cracks are prone to propagate into the 214 regions with lower stress level.

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216 Figure 3. Calculated evolutions of (a) the total crack length, (b) the maximum crack 217 radius, and (c) energy absorption with the time in the impact process. Here the total 218 crack length and maximum crack radius are normalized by the radius of the projectile 219 (R_n) and energy absorption is normalized by the initial kinetic energy of the projectile. 220 This normalization scheme is applied throughout this paper.

 To make a quantitative comparison between the results from different plates, we examined the total crack length and the maximum crack radius in the four simulated cases, as shown in Fig. 3(a) and Fig. 3(b) respectively. Here, the total crack length is calculated by summing up all the cracks developed in the plate, and the maximum crack radius is the length of the longest radial crack generated by the impact. Based on either 226 the total crack length or the maximum crack radius as calculated, the cracking resistance 227 of the four plates can be ranked in the following sequence: $S@C$ composite > 228 monolithic C > monolithic $S > C(\omega)S$ composite. Moreover, we examined the energy 229 absorption, which is defined as the loss of the kinetic energy of the projectile during $\frac{1}{\text{impact}}$ (Fig. 3(c)). These four plates exhibit an opposite sequence in energy absorption: 231 S@C composite \leq monolithic C \leq monolithic S \leq C@S composite. This makes sense because the absorbed energy is proportional to the total length of cracks generated in the plate given that the fracture energy (Γ) of both phases are presumably identical in our simulations.

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236 Figure 4. Contour maps of (a) total crack length, (b) maximum crack radius, and (c) 237 energy absorption as functions of the normalized inclusion radius (R_I/R_p) and the 238 normalized inter-inclusion spacing (D/R_p) . Total crack length and maximum crack 239 radius are normalized by the radius of the projectile (R_n) and the energy absorption is 240 normalized by the initial kinetic energy of the projectile. (d) Post-impact configurations 241 of three selected cases as marked by symbols of a circle, a diamond, and a pentagon in 242 $(a)-(c)$.

243 To further explore the potential of the $S@C$ composite in resisting crack 244 propagation, more FE simulations are conducted on a series of $S@C$ composite plates 245 with different inclusion radius (R_I) and inter-inclusion spacing (D) . The mechanical 246 properties of each phase are kept unchanged, as well as the properties and initial 247 velocity of the projectile. The simulation results including the total crack length,

248 maximum crack radius, and energy absorption by the end of the simulation are plotted 249 in terms of R_I/R_p and D/R_p in Fig. 4. It can be seen that crack propagation can be 250 restrained further by reducing the inter-inclusion spacing (D) or enlarging the inclusion 251 and radius (R_I) . This is because reducing D (with fixed R_I) or increasing R_I (with fixed D) 252 shortens the distance between the impact point and the nearest inclusions, which will 253 effectively enhance the suppression effect of the stiff inclusion on the crack propagation.

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255 Figure 5. Variations of (a) total length and (b) maximum radius with the normalized 256 radius of inclusions (R_I) for the given volume fraction of the inclusion (V_I) . Here, both 257 total length and maximum radius are normalized by the radius of projectile (R_n) . (c) 258 The post-impact cracking configuration of the S@C composite plates with the same 259 volume fraction ($V_I = 19.6\%$) but different R_I/R_p .

260 To visualize the effects of R_1 and *D* on cracking, the post-impact configurations 261 are shown in Fig. 4(d) for three selected cases which are marked with a circle, a 262 diamond, and a pentagon symbols in Figs. 4(a-c). Comparison between them 263 reconfirms that larger inclusions $(R₁)$ or shorter inter-inclusion spacing (D) results 264 in higher resistance to ballistic cracking. Given the ratio of R_I/D , namely the volume 265 fraction of the inclusion phase, the variations of the total crack length and maximum

Figure 6. Variations of the normalized (a) total crack length, (b) maximum crack radius,

 and (c) energy absorption as functions of volume fraction of stiff inclusion in S@C composite plates with different distribution patterns.

 In the above discussion, the stiff inclusions in the S@C composite plates are presumably distributed in a square pattern. To reveal the effect of distribution pattern on the resistance to cracking, we further considered the S@C plates with stiffer inclusions distributed in an equilateral triangular pattern. Figs. 6(a-c) compare the calculated total crack length, maximum crack radius, and energy absorption between the S@C plates with inclusions distributed in different patterns. Given inclusion size, the resistance to ballistic cracking in the S@C plates increases as the volume fraction of inclusions increases. Given both size and volume fraction of inclusion, the triangular pattern results in better resistance to ballistic cracking as compared to the square 292 counterparts. This is mainly due to the smaller distance between the impact point and 293 the nearest stiff inclusions, therefore stronger suppression effect to the cracking, in the 294 triangular pattern as compared to the square pattern with the same inclusion size and 295 volume fraction (Fig. S3).

 Figure 7. (a) Contour map of energy absorption as a function of inclusion radius (*R*I) and inter-inclusion spacing. Here the compliant inclusions are distributed in a pattern. Here energy absorption is normalized by the initial kinetic energy of the projectile. (b)

300 The cracking configurations of the C@S composite plates after impact by a projectile.

- 301 On the other hand, for the complementary C@S composites which exhibit superior 302 energy absorption potential, we investigated the dependence of energy absorption on 303 the size of the compliant inclusions and inter-inclusion spacing, as shown in Fig. 7(a). 304 It can be seen that larger inclusions and smaller inter-inclusion spacing lead to higher 305 energy absorption of C@S composites. The cracking configurations of three selected 306 cases are shown in Fig. 7(b). Similarly, for C@S plates with given volume fraction (*V*I), 307 the one with triangular patterned inclusions exhibits relative higher energy absorption 308 in comparison with the one with inclusions in square pattern (Fig. S4).
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310 **3.3 Experimental verification**

311 To experimentally verify the effect of stiffness heterogeneity on the resistance to 312 impact cracking, we carried out drop weight tests (Dynatup 9250HV, Instron) on the 313 S $@C$ and $C@S$ plate samples synthesized with polymethyl methacrylate(PMMA) and 314 soda-lime glass. Here, PMMA serves as the compliant (C) phase, while glass serves as 315 the stiff (S) phase. The elastic moduli of PMMA and glass at ambient temperature are 316 2-3 GPa and ~60 GPa, respectively. Fig. 8(a) shows the schematics of the experimental 317 setup and Figs. 8(b, c) display the images of two types of the composite after the drop 318 weight impact tests. As expected, severe cracking is observed in the $C(\widehat{\omega})$ S 319 (PMMA@Glass) sample, and some cracks initiated from the impact point even 320 penetrate and pass through the PMMA inclusions (Fig. 8(b)). In contrast, in the $S@C$ 321 (Glass@PMMA) sample, fewer and shorter cracks are generated and no clear crack 322 penetration into the glass inclusions is observed (Fig. $8(c)$), which is consistent with the 323 prediction of the numerical simulations above. Fig. $8(d)$ shows the variations of the 324 reaction force exerted on the projectile and the energy absorption (calculated from the 325 loss of kinetic energy of the projectile) with the time. It can be seen that PMMA@Glass 326 plate imposes higher resistant force to the projectile as compared to the Glass@PMMA 327 plate. However, both plates exhibit similar energy absorption even though the cracking 328 in the PMMA@Glass plate is apparently severer than that in the Glass@PMMA plate 329 (Fig. 8(b, c)). This is basically due to the much higher fracture energies (Γ) of PMMA

330 in comparison to that of the glass, as has been demonstrated by FE simulation (Table

331 **S1**, Fig. **S5**).

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333 Figure 8. (a) Schematic illustration showing the experiment the drop weight impact test. 334 (b-c) Photos of PMMA@Glass and Glass@PMMA composite plates after drop weight 335 tests. The dimensions of the plates are 100 mm \times 100 mm \times 2 mm. The diameter of the 336 inclusions is 10 mm, and the inter-inclusion spacing (center-to-center) is 15 mm. To 337 enhance the interfacial strength between the glass and PMMA, the glass surfaces were 338 functionalized with γ -MPS [30]. The drop weight applied was 17.34 kg and the drop 339 height was 0.5 m corresponding to an impact velocity of 3.13 $\text{m} \cdot \text{s}^{-1}$. The tup used is 340 a stainless steel cylinder (diameter: 5 mm) with a conical end (included angle $\approx 85^{\circ}$). 341 (d) Variations of reaction force applied on the projectile (left axial) and energy 342 absorption (right axial) as functions of time. Solid lines and dash lines stand for the 343 drop weight test on Glass@PMMA plate and PMMA@Glass plate, respectively.

344 **4. Discussion and conclusions**

 In this work, we systematically studied the effects of stiffness heterogeneity on the resistance of composites to impact cracking. It is revealed that impact cracking can be significantly prohibited in S@C composites produced by hybridizing stiff inclusions 348 into a compliant matrix. The performance of $S@C$ composites in prohibiting cracking can be optimized by tuning the size, volume fraction, and distribution pattern of the stiff 350 inclusions. Moreover, the performance of the $S@C$ composites in resisting impact 351 cracking also depends on other practical factors such as the interfacial strength between 352 the S and C phases and the impacting position of the projectile. FE simulations reveal 353 that weak inclusion/matrix bonding would diminish the cracking resistance of the $\frac{S}{Q}C$ 354 composites especially for the cases with smaller inter-inclusion spacing (Fig. S6); similarly, the cracking resistance of S (a)C composites will be weakened to some extent if the projectile impacts on the inclusion phase instead (Fig. S7).

357 On the other hand, for the complementary $C(a)S$ composites, which are the counterparts produced by hybridizing compliant inclusions into a stiff matrix, are found to facilitate impact cracking and therefore exhibit superior competence in energy absorption. Such distinct mechanical behaviors of these two types of composites in response to the impact loading have also been demonstrated in the experiment. The mechanism behind this phenomenon might be attributed to the inverse proportionality of energy release rate, which can be deemed as the driving force of crack propagation, to the stiffness of material under a given loading or stress intensity factor.

365 The distinct behaviors of $S@C$ and $C@S$ composites under impact loading endow them with different functionalities in application. For the materials whose structural integrity is crucial, applying the S@C hybridizing scheme could enhance their resistance to cracking. For the energy-absorption materials, which are often used as 369 disposable shields for protection, applying the $C(a)$ S hybridizing scheme could promote the competence of energy absorption. Moreover, these two types of hybridizing strategies can be used together to exert their synergic effects. For example, we can stack them in tandem to form a double-layer assembly, as shown in Fig. 9. Similar FE simulations under the same ballistic impact as described above show that the cracking configurations of the double-layer assembly depend on the stacking sequence of the 375 plates. If the C@S plate is placed in the front of the S@C plate, the front C@S plate is 376 cracked severely after impact while the rear $S@C$ plate is almost intact (Fig. 9(a)). In 377 contrast, if the S $@C$ plate is placed ahead of the C $@S$ plate, the projectile can penetrate both plates (Fig. 9(b)). A similar phenomenon is also observed in the control double- layer assemblies composed of the monolithic stiff and compliant plates, as shown in Fig. 9(c) and (d). In summary, our work provides a theoretical guideline for harnessing impact cracking by tuning stiffness heterogeneity. This strategy could be applied further to controllably deflect and guide the crack propagation trajectory, and finally constrain the fracture within a limited region [36, 37]. Furthermore, our hybridizing schemes can be applied in combination with other approaches such as applying T-stress [38] to obtain more efficient crack-controlling strategies. These technologies are believed of great value to the design and manufacture of anti-impact materials such as windshields and shields for space stations and satellites [39].

 Figure 9. Simulated cracking configurations of double-layer assemblies composed of (a) a C@S plate backed by an S@C composite plate, (b) an S@C plate backed by a C@S plate, (c) a monolithic compliant plate backed by a monolithic stiff plate, (d) a monolithic stiff plate backed by a monolithic compliant plate. In all cases, the two plates are parallel and separated by 1 cm. The radius and spacing between inclusions, if 394 available, are $R_1 = 4$ mm and $D = 12$ mm, respectively.

Declaration of Competing Interest

The authors declare that they have no known competing financial interests or personal

relationships that could have appeared to influence the work reported in this paper.

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