

## **Axial behaviour of steel pipelines buried in sand: effects of surface roughness and hardness**

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1 **ABSTRACT**

2 Surface roughness and coating hardness of underground pipelines are expected to play decisive  
3 roles in their axial pullout behaviour, which is an important aspect of pipeline design. Existing  
4 guidelines and previous studies underestimated or ignored these effects, resulting in potentially  
5 unsafe design. To address this problem, the current study conducted nine large-scale physical  
6 modelling tests on pipes in dry and dense sand. Five steel pipes with varying normalised  
7 roughness (0.04-1.01) and coating hardness (32.6-59.0 HRA) were used and instrumented with  
8 a novel type of film-like piezoresistive sensors for measuring soil-pipe contact pressure. The  
9 measured pullout resistance of rough pipes is 2.70-2.85 times of smooth pipes, significantly  
10 greater than the value specified in current design guidelines (i.e., 1.17 times). This substantial  
11 increase stems from an increase in interface friction coefficient (accounting for 72-79%) and a  
12 contact pressure increase induced by constrained dilation and soil arching (contributing the  
13 remaining 21-28%). Regarding coating hardness, a critical hardness was observed (around 35  
14 HRA). Due to equivalent roughness from particle embedding, pipes with hardness below this  
15 value exhibited similar behaviour to rough pipes. Finally, a new and simple method was  
16 proposed for calculating the pullout resistance with consideration of the effects of roughness  
17 and dilatancy.

18 **Keywords**

19 Pipes & pipelines; Buried structures; Model tests; Soil/structure interaction

## 20 **Introduction**

21 In pipeline engineering, surface treatment techniques such as coating, wrapping, and sand-  
22 blasting are commonly employed on underground pipelines to prevent corrosion, adjust  
23 resistance and provide thermal insulation ([ISO 8501-1, 2007](#); [CEDD, 2020](#)). These treatments  
24 can greatly impact the pipe surface roughness and hardness, which are expected to affect the  
25 soil-pipeline interaction that may happen during landslides, earthquakes, and thermal  
26 expansion/contraction. So far, some researchers have investigated the influence of surface  
27 roughness and hardness on soil-pipe interaction in the lateral and vertical directions. Physical  
28 model tests by [Trautmann & O'Rourke \(1985\)](#) and [Trautmann et al. \(1985\)](#) have demonstrated  
29 that variations in lateral resistance and uplift vertical resistance for pipes with different surface  
30 roughness and coating materials are approximately 10% and 10-30%, respectively. These  
31 results suggest that the impact of surface roughness and hardness on the lateral and vertical  
32 interaction is not substantial, as they are primarily governed by the surrounding soil behaviour.  
33 In contrast, the behaviour of axial soil-pipeline interaction (ASPI) is likely influenced by the  
34 interface characteristics between the soil and pipe. Consequently, roughness and hardness,  
35 which significantly affect interface shear strength and dilatancy, are crucial determinants of  
36 axial resistance, as indicated by research from [Dove & Frost \(1999\)](#), [Han et al. \(2018\)](#) and  
37 [Ghanadizadeh et al. \(2022\)](#). Neglecting their influences on the axial load evolution may lead  
38 to unsafe or uneconomical designs of pipeline systems ([O'Rourke et al., 1990](#); [Klar & Marshall,](#)  
39 [2008](#)). For instance, underestimating the axial force imposed by potentially moving soils could  
40 lead to unsafe pipeline design.

41 As a widely used design guideline, [ASCE \(1984\)](#) calculates the pullout resistance (i.e., the  
42 maximum axial force per meter length) of pipes buried in sands,  $T_{\max}$ , using the Mohr-Coulomb  
43 theory. It is assumed that the average contact pressure (namely, the average normal stress) on  
44 pipes is equal to the mean of nominal vertical pressure at the pipe crown,  $\sigma_c' (= \gamma'H_c)$ , and its

45 lateral earth pressure at rest  $K_0 \sigma_c'$ . Hence,  $T_{\max}$  can be calculated as follows:

$$46 \quad T_{\max} = \pi D \frac{1+K_0}{2} \sigma_c' \tan \delta \quad (1)$$

47 where  $D$  is the pipe outer diameter;  $\gamma'$  is the effective unit weight of the backfill sand;  $H_c$  is the  
48 buried depth of the pipe crown;  $\tan \delta$  is the soil-interface friction coefficient;  $\delta$  is the interface  
49 friction angle. [ALA \(2001\)](#) and [PRCI \(2009\)](#) further specify that  $\delta = f \varphi'$ , where  $f$  is the soil-  
50 pipe interface friction factor and  $\varphi'$  is the internal friction angle of sand. The value of  $f$  is  
51 recommended based on the surface roughness and coating materials (related to surface  
52 hardness), as summarised in Table 1. The classification of surface conditions is qualitative but  
53 not quantitative. More importantly, this equation only considers the roughness and hardness  
54 effects on  $\delta$  but ignores their impacts on the contact pressure between soil and pipe during  
55 ASPI ([Lings & Dietz, 2005](#); [Abuel-Naga et al., 2018](#)). The evolution of contact pressure could  
56 be complex due to various factors, especially the constrained dilatancy and soil arching. For  
57 instance, [Wijewickreme et al. \(2009\)](#) conducted three physical modelling tests in dense soil  
58 with one rough steel pipe. The experimental results and corresponding numerical simulations  
59 show that the constrained dilation increased the contact pressure on the pipe surface and, thus,  
60 the pullout resistance. In addition, constrained dilatancy may interact with other factors, such  
61 as soil stiffness and pipe diameter ([Yin et al., 2012](#); [Wijewickreme & Weerasekara, 2015](#)).  
62 [Sheil et al. \(2018\)](#) tested one heavy steel pipe coated by fusion-bonded epoxy (FBE). The soil  
63 arching was found to affect the contact pressure and the pipe pullout behaviour.

64 As far as the authors are aware, no physical model test has been reported in the literature for  
65 investigating the effects of surface roughness on ASPI. Regarding the effects of coating on the  
66 pullout behaviour of pipelines, only [Scarpelli et al. \(2003\)](#) conducted three valuable but simple  
67 field tests using sands with steel pipes coated by polyethene and coal tar. The pullout  
68 resistances of pipes with harder surfaces were smaller than those of softer pipes. The role of  
69 soil arching and constrained dilatancy was not explored. Clearly, more experimental results are

70 needed to understand and quantify the effects of roughness and hardness on the ASPI  
71 mechanism and pullout resistance.

72 In this research, a large-scale experimental system was developed to study ASPI. Nine  
73 pullout tests were conducted using five pipes with different surface treatments, including three  
74 surface roughness and two coating materials. The main objectives are: (a) to investigate the  
75 extent to which surface roughness and hardness affect the pullout resistance, (b) to examine  
76 the mechanisms of surface condition effects, and (c) to propose a new method of calculating  
77 pipe pullout resistance.

### 78 **A new and large-scale experimental system**

79 An experimental apparatus for testing ASPI was modified from the soil nail testing system of  
80 [Yin & Su \(2006\)](#), as shown in Fig. 1. It consists of a steel box, pipes, an axial actuation  
81 subsystem, and a vertical loading subsystem. The steel box with strengthening beams was used  
82 to simulate the pipeline trench conditions. It has internal dimensions of 1.0 m in length, 0.8 m  
83 in height, and 0.6 m in width. This width meets the minimum requirements of the trench for  
84 pipes with a diameter of less than 150 mm, as specified in the guideline of [USBR \(1996\)](#). Two  
85 holes are provided on the front and rear walls. The pipe can pass through these holes to ensure  
86 a constant soil-pipe interaction length during the pullout process.

87 The front and rear walls of the box would cause non-uniform shear stress distribution along  
88 the pipe. This problem was observed in physical model tests by [Al-Khazaali & Vanapalli  
89 \(2019\)](#) and DEM simulation by [Meidani et al. \(2017\)](#). Therefore, this study introduced two pipe  
90 sleeves with an inner diameter of 132 mm to avoid the soil-pipe interaction in the zone close  
91 to the front and rear walls. Their efficiency in minimizing boundary effects is discussed later.  
92 The sleeves inside the box are 0.15 m long, resulting in a soil-pipe interaction length of 0.7 m.

93 The axial actuation subsystem, including an electric linear actuator, a servo control panel,  
94 and a reaction frame, was used to impose axial displacement. A connector (see Fig. 1) was

95 introduced to allow a free relative vertical movement between the pipe and the actuator. Rubber  
96 sleeve membranes with a nominal inner diameter of 100 mm and a nominal thickness of 1.5  
97 mm were inserted between the pipe and sleeves to prevent soil leakage. The pipe and sleeves  
98 were aligned coaxially, creating a gap of approximately 8.5 mm ( $22 d_{50}$  of the sand used in this  
99 study) between the pipe and sleeves. This gap provides sufficient space for pipe settlement (no  
100 more than 0.5 mm in this study) and the development of interface shear zone ( $2-10 d_{50}$ , [DeJong](#)  
101 [& Westgate \(2009\)](#)).

102 A flexible pneumatic bag (see Fig. 1) was installed between the soil and the top cap. The  
103 length and width of the pneumatic bag match the dimensions of the box's interior. By adjusting  
104 the air pressure  $P$  inside the bag, the nominal pressure at the crown,  $\sigma_c'$ , can be controlled by  
105  $\sigma_c' = \gamma H_{c0} + P$ , where  $H_{c0}$  = real buried depth from the ground surface to the crown of the pipe.  
106

#### 107 **Pipes with different surface roughness and hardness**

108 Hot-rolled seamless steel pipes, commonly used in Hong Kong, with a nominal outer diameter  
109 of 102 mm and a nominal thickness of 4 mm, were used ([BSI EN 10220:2002, 2002](#); [HKIUS,](#)  
110 [2011](#)). Further details can be found in Table 2. The effects of fluid weight on the pipe pullout  
111 mechanism might be insignificant in most cases, and they are not considered here. Smooth,  
112 intermediate, and rough pipes were prepared using three non-coated raw steel. Their surface  
113 roughness was quantified with the normalised surface roughness ( $R_n$ ) using the equation by  
114 [Kishida & Uesugi \(1987\)](#):  $R_n = R_{\max}/d_{50}$ , where  $d_{50}$  is the average particle size, and  $R_{\max}$  is the  
115 maximum height on the surface profile over a travel length of  $d_{50}$ . It should be noted that  
116 different classification methods for rough, intermediate and smooth pipes are available in the  
117 literature. One method is based on the variation of interface friction angles with the normalised  
118 roughness. The interface friction angles increase with the normalised roughness for  
119 intermediate interfaces and normally keep constant for smooth and rough interfaces. The

120 critical values of normalised roughness that differentiate smooth-intermediate and  
121 intermediate-rough interfaces are 0.02 and 0.5, respectively, as suggested by [Paikowsky et al.](#)  
122 [\(1995\)](#). Alternatively, some researchers use dilation/contraction during shearing ([Lings &](#)  
123 [Dietz, 2005](#); [Farhadi & Lashkari, 2017](#)): the critical value of normalised roughness transiting  
124 the intermediate to rough ranges is normally 0.1-0.2 in this framework. The first approach is  
125 adopted in the current study. Without any treatment, the raw seamless steel pipe has a  
126 roughness value of 0.04, as measured by a surface roughness tester (SJ-210, Mitutoyo).  
127 Although slightly larger, this value is very close to the critical value of 0.02, so the raw pipe is  
128 referred to as a smooth pipe in this study. The surfaces of the rough and intermediate pipes  
129 were treated using the turning method. Turning depths of 0.38 mm with an interval of 0.9 and  
130 0.08 mm with an interval of 0.4 mm were applied to achieve normalised roughness values of  
131 1.01 and 0.21, respectively.

132 Two commonly used coatings, epoxy asphalt (EA) and fusion bonded epoxy (FBE), coated  
133 two raw steel pipes to evaluate the hardness effects. Surface hardness tests were conducted  
134 using the Rockwell hardness scale of HRA by a hardness tester (OMAG 206) ([ASTM E18,](#)  
135 [2022](#); [ASTM D785, 2023](#)). The measured hardness values for EA, FBE, and raw steel are 32.6,  
136 44.2, and 59.0, respectively. The measured values of  $R_n$  of the two coated pipes are both 0.01.

### 137 **Instrumentation**

138 A novel Force Sensing Resistor (FSR), namely FSR 402 from Interlink Electronics, was used  
139 to measure earth pressure and soil-pipe contact pressure ([Interlink, 2023](#)). It is a piezoresistive  
140 sensor whose electric resistance is a function of pressure. Its thinness and flexibility can  
141 effectively eliminate the arching effects on sensors and conform well to curved surfaces ([Liu](#)  
142 [et al., 2021](#); [Kootahi & Leung, 2022](#)). To measure earth pressure (Application I in this study),  
143 each FSR was affixed to a 1 mm-thick steel slice and coated with epoxy adhesive, as shown in  
144 Fig. 2. Five FSRs for Application I were utilised to measure the vertical soil stress at various

145 depths, as depicted in the side view of Fig. 1. As for measuring the soil-pipe contact pressures  
146 (Application II), FSRs were installed on the pipe surface (see Figs. 1 and 2). To facilitate wire  
147 routing, small holes were created in the pipe, and the sensors were secured and protected using  
148 epoxy adhesive while the holes were sealed. Eighteen FSRs were deployed across three cross-  
149 sections with a spacing of 0.2 m between sections. Each cross-section included six FSR sensors  
150 mounted at the crown, one shoulder, two springlines, one haunch, and the invert (see Fig. 1).

151 FSR's electrical conductance (the reciprocal of electrical resistance) was measured using a  
152 datalogger of dataTaker DT85G. The pressure-conductance relationship is found to be  
153 dependent on the specific installation condition of each sensor, so post-installation calibration  
154 is necessary. The sensors in Application I were put into a sealed tank. As for Application II,  
155 pipe repair clamps, consisting of a rubber sleeve with thickened edges and a steel split sleeve  
156 with screws, were used to form a sealed space for the sensors. A hole was punched in the middle  
157 of the repair clamp and connected hermetically to a pneumatic connector. By adjusting the air  
158 pressure in the sealed tank/space, the relationship between pressure and output can be  
159 determined. A typical pressure-conductance relationship with a loading rate of 4 kPa/min from  
160 0 to 200 kPa is shown in Fig. 2. The mean of non-linearity error within a measurement range  
161 of 0-200 kPa from calibration results of eighteen FSR sensors is approximately 5.8%. To  
162 improve the accuracy, a nonlinear signal processing method was employed: the calibration  
163 results were fitted using the smoothing spline algorithm in MATLAB (see the line in Fig. 2).  
164 In addition, the influence of shear stress on the FSR reading was assessed through two direct  
165 shear tests. One FSR sensor was installed on a steel interface with an  $R_n$  of 1.01. Under effective  
166 normal pressures of 55 and 110 kPa, the shear stresses were increased from 0 to the interface  
167 strengths of 47 and 73 kPa, respectively. During shearing, the differences between the  
168 measured and applied effective normal pressures were all below 10%.

169 An optic fibre was utilised for the optical frequency domain reflectometry (OFDR) method

170 to measure the pipe axial strain distribution to verify the feasibility of sleeves and rubber  
171 membranes. This fibre was attached to the pipe's inner surface at two shoulders and two  
172 haunches.

173 Two linear variable differential transformers (LVDTs) were utilised to measure the pipe's  
174 vertical displacement. Additionally, a load cell with a measurement range of  $\pm 10,000$  N and  
175 an LVDT with a measurement range of 50 mm were adopted to monitor the axial force and  
176 axial displacement, respectively. The locations of these sensors are shown in Fig. 1.

### 177 **Test materials**

178 The testing soil is standard medium sand sourced from Fujian Province, China, with a particle  
179 size ranging from 0.25 to 0.5 mm. Its properties are summarized in Table 2. The target relative  
180 density is 85%. A series of direct shear tests were conducted to measure the internal friction  
181 angle of pure sand and the interface friction angle between sand and interfaces (treated using  
182 the same methods on pipes). The shear box and test method are similar to those of [Cui et al.](#)  
183 [\(2024\)](#). The sizes of specimens in direct shear tests and interface direct shear tests were  $60 \times$   
184  $60 \times 40$  mm and  $60 \times 60 \times 20$  mm, respectively. The shear rate was 0.02 mm/s, consistent with  
185 the pipe pullout speed in the following physical modelling. The effective normal stresses were  
186 set as 17, 34, 50, and 100 kPa, corresponding to the target nominal pressure at the pipe crown  
187 in the physical modelling. The results are shown in Table 3.

### 188 **Testing programme**

189 Two series of pullout tests were conducted, as summarised in Table 4. Series I consisted of  
190 seven tests to study the effects of roughness and stress. Smooth and rough pipes were buried at  
191 different nominal pressures  $\sigma_c'$ . The target nominal pressures  $\sigma_c'$  include 17, 34, and 50 kPa,  
192 corresponding to 1, 2, and 3 m in burial depths. An intermediate pipe was also tested under a  
193 nominal pressure  $\sigma_c'$  of 34 kPa. Series II involved the pullout testing of FBE-coated and EA-  
194 coated pipes under a nominal pressure  $\sigma_c'$  of 34 kPa. These coated pipes were compared with

195 the smooth pipe to analyse the effects of coating hardness.

## 196 **Model preparation and test procedures**

197 The sand pluviation method (Fretti et al., 1995) was introduced to prepare dense and uniform  
198 samples and simulate the dumping technique in the engineering practice of compaction. The  
199 models were prepared in layers. Each layer had a thickness of approximately 25 mm. A laser  
200 level magnetically attached to the box's inner wall and six rulers affixed to the side walls were  
201 used to get a flat surface for each layer. During preparing the sample, sleeves were installed  
202 once the depth reached 0.28 m. At an elevation of 0.33 m, the pipe with rubber membranes was  
203 put into the box. The soil around the ends of the sleeves was temporarily moved, and after  
204 placing the rubber membranes and adjusting the pipe to the desired position, the removed soil  
205 was backfilled and compacted to its original state using a square wooden mallet. At this stage,  
206 the pipe was buried to a depth equalling 30% of its diameter. Subsequently, sand was  
207 continuously added until the elevation reached 0.76 m.

208 After preparing the sample, the pneumatic bag and top cap were put on the soil. The air  
209 pressure was controlled, and this condition was maintained for approximately 30 minutes until  
210 sensor readings, such as vertical displacement and earth pressure, stabilised. Finally, the pullout  
211 speed of the pipe was set as 0.02 mm/s, and the target displacement was 20 mm, both of which  
212 are common values in previous studies (Sheil et al., 2018; Reza & Dhar, 2021).

## 213 **Evaluation of boundary effects**

214 Two boundary effects were addressed in this study. The first one is the non-uniform distribution  
215 of shear stress caused by the presence of front and rear walls. As mentioned above, two sleeves  
216 with rubber membranes were introduced to address this problem. Fig. 3 illustrates the axial  
217 strain distribution measured by optic fibre with various axial displacements using the rough  
218 pipe under a surcharge of 50 kPa. The square points represent the axial strain at the side of the  
219 actuator ( $d = 700$  mm) calculated by the axial force measured by the load cell, Young's

220 modulus of steel and pipe cross-sectional area. These measurements align with the micro  
221 strains obtained through the fibre at the pipe end, confirming the accuracy of the strain  
222 measurement. Throughout the pullout process, the distance  $d$  and the axial strain show a good  
223 linear relationship ( $R$ -squared  $> 0.99$ ). It indicates that the non-uniformity of the shear stress  
224 distribution was minimised successfully by sleeves and membranes and that the size of the gap  
225 between pipe and sleeves is applicable to guarantee sufficient lateral support for the  
226 surrounding soils by rubber membranes.

227 The second boundary effect pertains to the roughness of the side walls. Two tests were  
228 conducted using the smooth pipe under a surcharge of 34 kPa. The side walls of the first test  
229 consisted of smooth steel plates ( $R_n = 0.01$ ), while for the second test, two pieces of sandpaper  
230 with a grit size of P40 ( $R_n \approx 1.13$ ) were attached to the steel plates to simulate rough trench  
231 conditions. The measured axial resistance-displacement relationships for the two tests exhibit  
232 remarkable consistency, suggesting that the roughness of the side walls has negligible effects.

## 233 **Experimental results and discussion**

### 234 *Roughness effects on pullout force-displacement relationship and pullout resistance*

235 Fig. 4 illustrates the relationship between axial displacement and mobilised axial resistance in  
236 Series I. The axial pullout behaviour is significantly influenced by roughness. The results show  
237 that rough and intermediate pipes exhibit obvious strain-softening behaviour, while strain-  
238 softening is negligible for smooth pipes. This finding aligns with the results of soil-interface  
239 direct shear tests, where rough interfaces exhibit significant dilatant behaviour, while smooth  
240 interfaces show limited dilation. Under the same nominal pressure, the pullout resistance of  
241 rough pipes is 1.70-1.85 times higher than smooth pipes. This value is much larger than 0.17  
242 ( $\tan(0.8\varphi')/\tan(0.7\varphi')-1$ ) suggested by guidelines (see Table 1). Such a significant increment is  
243 due to different mechanisms. The increase in friction coefficient from soil-pipe smooth to  
244 rough soil-pipe interfaces (see Table 3) is 1.34 times ( $\tan(37.9^\circ)/\tan(18.4^\circ)-1$ ). It means that

245 72-79% of the roughness-caused resistance increase is attributed to the increase in interface  
246 friction coefficient. The remaining 21-28% is mainly attributed to the increment of soil-pipe  
247 contact pressure that is closely related to the constrained dilation, as discussed later.

248 Fig. 5 demonstrates the relationship between the pullout resistance and the nominal vertical  
249 pressure at the crown for rough and smooth pipes. For both pipe types, the intercept of the trend  
250 line is higher than zero if assuming a linear envelope. This phenomenon is similar to the  
251 nonlinear failure envelope in interface shear. The soil's dilatancy becomes more pronounced  
252 at lower pressure, leading to this non-linearity.

253 Fig. 6 (a) compares the interface friction coefficients measured in the interface direct shear  
254 tests ( $\mu_{dir}$ ) (see Table 2) and predicted by the guidelines ( $\mu_{ALA}$ ) (see Table 1) to  $\mu_{phy}$  (=  $T_{max}/[\pi D(1/2)(1+K_0)\sigma_c]$ ). The value of  $\mu_{dir}$  ranges from 0.49 to 0.78, aligning with the findings  
255 of previous shear studies on dense sand-structure interfaces (Lings & Dietz, 2005; Farhadi &  
256 Lashkari, 2017; Han et al., 2018).  $\mu_{ALA}$  is calculated from  $\tan\phi'$ . The guidelines (ALA, 2001;  
257 PRCI, 2009) do not define "rough" and "smooth"; thus, a range is depicted in Fig. 6 (a).  
258 According to Table 1, the values of  $\mu_{ALA}$  for smooth and rough steel pipe are  $\tan 0.7\phi'$  (= 0.53)  
259 and  $\tan 0.8\phi'$  (= 0.62), respectively. Finally,  $\mu_{phy}$  is the back-calculated value based on pullout  
260 resistances from physical modelling and Eq (1), representing the ratio of the pullout resistance  
261 to the assumed soil-pipe contact force per unit length. For the sake of simplicity in expression,  
262 it is denoted as  $\mu_{phy}$ , although it differs from the concept of the interface friction coefficient  
263 given changes in soil-pipe contact pressure during the pullout process. The value of  $\mu_{phy}$  ranges  
264 from 0.45 to 1.52 and increases with the normalised roughness, showing an approximate linear  
265 relationship with the logarithmic value of roughness. If the average contact pressure is constant  
266 (CNL conditions) during pullout,  $\mu_{phy}$  should be equal to  $\mu_{dir}$ . However,  $\mu_{dir}$  consistently  
267 remains lower than  $\mu_{phy}$  in Fig. 6 (a). It implies that the average contact pressure on the pipe  
268 surface increased during the pullout process. The pipe pullout behaviour aligns more closely  
269

270 with the constant normal stiffness (CNS) condition rather than the CNL condition due to the  
271 constrained dilation behaviour (DeJong & Westgate, 2009; Pra-ai & Boulon, 2016; Ng et al.,  
272 2020; Zhou et al., 2020): the presence of sands far away from the interface constrains the  
273 dilation trend at the interface, leading to an increase in contact pressures during shearing. The  
274 dilatancy of the soil-pipe interface is positively correlated with the interface roughness (Lings  
275 & Dietz, 2005; Zhang et al., 2011; Farhadi & Lashkari, 2017). Therefore, although  $\mu_{dir}$  also  
276 increases as the roughness increases, its growth rate with roughness increase is not as steep as  
277 that of  $\mu_{phy}$  and their difference,  $\mu_{phy}-\mu_{dir}$ , widens with increasing roughness.

278 The values of  $\mu_{ALA}$  only encompass a small range of the measured values of  $\mu_{phy}$ . The  
279 guidelines slightly overestimate smooth pipes' pullout resistance and significantly  
280 underestimate that of rough pipes. Both underestimation and overestimation can pose safety  
281 risks to the pipeline system. Underestimation implies a lower design pipeline strength. When  
282 there is any relative axial displacement between the pipe and the soil, the stress experienced by  
283 the pipe may exceed the pipeline strength, potentially causing damage to the pipe. On the other  
284 hand, overestimating the pullout resistance results in higher expectations for the soil's ability  
285 to limit deformation in the pipeline system. This can potentially lead to issues like upheaval  
286 buckling due to thermal expansion, bending due to nearby tunnel construction (Marshall et al.,  
287 2010; Wang et al., 2011), or excessive loads concentrated on weak points, such as valve stations  
288 or cracks in the piping system, as discussed by PRCI (2009). Moreover, the wide range of  $\mu_{phy}$   
289 underscores the importance of quantifying roughness in pipeline design rather than relying on  
290 the qualitative definition provided by guidelines (ALA, 2001; PRCI, 2009).

### 291 ***Roughness effects on contact pressures distribution and evolution***

292 Fig. 7 illustrates the distribution and evolution of contact pressures for smooth and rough pipes  
293 subjected to a nominal vertical pressure of 34 kPa at the crown. The difference in the contact  
294 pressures measured by FSR sensors is generally within  $\pm 10\%$  of the average value at the

295 corresponding positions on the three cross-sections. The contact pressure difference between  
296 the left and right springlines is kept under 2 kPa, indicating good symmetry. Therefore, the  
297 average values at the crown, shoulders, springlines, haunches, and invert are directly displayed.  
298 The contact pressure assumed by the guidelines is  $\sigma_c'$  (34 kPa) at the crown and invert,  $K_0\sigma_c'$   
299 (12.3 kPa,  $K_0$  takes as  $1 - \sin \varphi'$ ) at the springlines,  $(1+K_0)\sigma_c'/2$  (23.2 kPa) at the shoulders and  
300 haunches, respectively.

301 Before the pullout, the roughness did not affect the initial contact pressure distribution. It is  
302 distributed with the shape of a “Norman Shield”, as shown in Fig. 7. This shape is consistent  
303 with the testing results of [Wijewickreme et al. \(2009\)](#) and [Sheil et al. \(2018\)](#). The contact  
304 pressures at the invert and shoulders are always larger than the assumed values, while those at  
305 the crown, springlines, and haunches are normally less than the expected value. However, the  
306 differences between measured values and expected values are typically not more than 25% of  
307 expected values. The mean values of initial average contact pressure with  $\sigma_c'$  of 17, 34, and 50  
308 kPa are 12.6, 22.5, and 37.5 kPa, respectively. These average values closely align with the  
309 assumed values by  $(1+K_0)\sigma_c'/2$  (11.6, 23.2, and 34.1, respectively) because the increase and  
310 decrease in contact pressure at different locations offset each other. Calculating the initial  
311 average contact pressure using  $(1+K_0)\sigma_c'/2$  is acceptable.

312 During the pullout process, the contact pressures on different pipes exhibit an overall  
313 increasing trend, with the magnitude of the increase being positively correlated with the  
314 roughness (see Fig. 7). This finding supports the explanation provided earlier regarding why  
315  $\mu_{\text{phy}}$  consistently exceeds  $\mu_{\text{dir}}$  and why  $\mu_{\text{phy}} - \mu_{\text{dir}}$  is influenced by the roughness and hardness of  
316 the pipe surface.

317 Different evolution processes of contact pressures reflect on pipes with different roughness.  
318 For the rough pipe (Fig. 7 (b)), the contact pressure increase is persistent and notable. This  
319 increase predominantly occurs at the crown and the invert, where the pressures can reach

320 around 55 and 70 kPa, respectively. The increase at the springlines (around 8 kPa) is  
321 insignificant. Moving to the smooth pipe (Fig. 7 (a)), most of the contact pressures experience  
322 an initial increase before reaching a displacement of 3 mm, a subsequent decrease after the  
323 peak, and finally, a slight increase. This fluctuation trend is similar to the results of  
324 [Wijewickreme et al. \(2009\)](#).

### 325 *Effects of soil arching and constrained dilatancy on pipe pullout mechanism*

326 The distribution and evolution of soil-pipe contact pressure in Fig. 7 are closely related to the  
327 soil arching effects ([Sheil et al., 2018](#); [Meguid, 2019](#)), and they are explained using the  
328 schematic diagram in Fig. 8. During the backfilling process, the soil above the pipe is expected  
329 to experience a smaller settlement than the soil at the pipe sides, owing to the stiffening effects  
330 of pipe. The relative movement results in internal friction in the soil, concentrating more  
331 pressure on the pipe, especially at the crown, shoulders and the invert. This pressure  
332 concentration also causes the pipe to settle. In this study, the measured pipe settlements during  
333 this process are 0.1-0.5 mm, increasing pressure at the invert but reducing the pressure at the  
334 crown towards  $\sigma_c'$ . In addition, the concentration of overburden pressure on the pipe reduces  
335 the vertical pressure on the sides, thus lowering the contact pressure on the springlines (lateral  
336 earth pressure).

337 During the pullout process, the evolution of soil-pipe contract pressure is likely affected by  
338 two different mechanisms. Firstly, shearing at the soil-pipe interface disturbs the original stress  
339 equilibrium, weakening the soil arching effects developed during the backfilling process and  
340 tending to reduce the contact pressure. Secondly, the constrained dilation at the soil-pipe  
341 interface tends to increase the contact pressure and could cause additional soil arching due to  
342 further relative vertical displacement between the soils at pipe sides and above the pipe,  
343 resulting in a more significant increase of contact pressures at the crown, shoulders, and invert.  
344 The overall change in contact pressure depends on the relative importance of these mechanisms.

345 For the smooth pipe, the contact pressure at most points increases slightly due to the second  
346 mechanism, while contact pressure at the springlines increases more significantly due to the  
347 first mechanism, as shown in Fig. 7 (a). As for the rough pipe, the second mechanism likely  
348 plays a dominant role (Martinez & Frost, 2017). The average contact pressure increases sharply  
349 due to its significantly constrained interface dilation behaviour. The additional soil arching  
350 makes this increase concentrate on the crown and invert and the vertical component of the  
351 contact pressures of shoulders and haunches. This additional soil arching also weakens the  
352 overburden vertical pressure on the soil of the pipe sides, reducing the corresponding lateral  
353 earth pressure (contact pressure at the springlines). It offsets part of the contact pressure  
354 increase caused by constrained dilation, resulting in limited pressure change at the springlines  
355 (as shown in Fig. 7 (b)).

356 The evolution of earth pressure in Fig. 9 further supports the above explanation. Before  
357 pullout, the earth pressures above and below the pipe are consistently greater than the earth  
358 pressure on the pipe sides with the same level due to the pressure concentration on the pipe.  
359 During the pullout process, the earth pressures around the smooth pipe remain unchanged or  
360 increase slightly, indicating that the impact of dilation behaviour is limited. In contrast, the  
361 earth pressures above and below the rough pipe experience significant growth, while the  
362 vertical pressure on the pipe sides slightly decreases. This is consistent with the evolution of  
363 contact pressures shown in Fig. 7 (b).

#### 364 ***Roughness effects on stress path***

365 Fig. 10 illustrates the stress paths at the soil-pipe interfaces in physical modelling. The failure  
366 envelopes of three interfaces are determined through CNL interface direct shear tests and  
367 included as references. The shape of the stress path remains unaffected by the overburden  
368 pressure but is influenced by the roughness. For the smooth pipe, the stress path generally  
369 includes four stages. In the first stage, the stress path rises vertically, meaning the friction angle

370 mobilisation is essentially elastic during this process. Then, the path turns to the second stage  
371 by moving towards the top right direction. It illustrates an increase in average contact pressure  
372 caused by constrained dilatancy, as explained above. Subsequently, the stress path turns to the  
373 upper left in the third stage due to the average contact pressure decrease caused by the  
374 disturbance of soil arching. During the ultimate stage, the path moves slightly to the bottom  
375 right after touching the peak failure envelope. It indicates a slow decrease in the mobilised  
376 friction angle and slight constrained dilation behaviour.

377 As for the stress paths of rough and intermediate pipes, they generally include three stages  
378 resembling the typical stress path observed in constant normal stiffness (CNS) tests on the  
379 rough interface (Ooi & Carter, 1987; Pra-ai & Boulon, 2016). The stress paths of rough and  
380 intermediate pipes during the first and second stages are similar to those of smooth pipes, which  
381 rise vertically and then turn to the top right. The major difference is that, before touching the  
382 peak failure envelope, the stress path keeps the trend moving towards the top left rather than  
383 undergoing the smooth pipes' third stage due to its significant constrained dilation.  
384 Subsequently, the stress paths turn to the bottom right and tend to touch the critical state  
385 envelope in the third stage of rough and intermediate pipes. During this process, the  
386 degradation of the mobilised friction angle is much more evident than the further increase in  
387 contact pressure, corresponding to the obvious softening behaviour in Fig. 4.

### 388 *Coating hardness effect on pullout behaviour*

389 Moving on to the coating hardness effects in Series II, the pullout force-displacement curves  
390 of the EA-coated and FBE-coated pipes in Fig. 4 resemble those of rough and smooth pipes,  
391 respectively, although the values of normalised roughness are both small for these two coated  
392 surfaces. Additionally, the resistance of the EA-coated and FBE-coated pipes is close to that  
393 of rough and smooth pipes, respectively (see Fig. 5). These observations are mainly related to  
394 particle embedment. After tests, numerous particles were found embedded in the surface of the

395 EA-coated pipe, while no embedment or scratches were observed on the FBE-coated pipe (see  
396 Series II in Fig. 4). The pipes coated with softer materials may be considered a specialised form  
397 of rough pipe due to the increase in equivalent roughness caused by embedment. Conversely,  
398 the pipes coated with harder materials do not experience this increase in equivalent roughness.

399 The above results are further supported by Fig. 6 (b). On the one hand,  $\mu_{\text{phy}}$  for EA-coated,  
400 FBE-coated, and smooth steel pipes are 1.21, 0.45, and 0.45-0.54, respectively, larger than the  
401 corresponding  $\mu_{\text{dir}}$  values of 0.75, 0.35, and 0.33. The  $\mu_{\text{phy}}-\mu_{\text{dir}}$  of EA-coated and FBE-coated  
402 pipes also closely resemble those of rough and smooth pipes, respectively. On the other hand,  
403 similar to rough pipes, the resistance of pipes with soft material coatings (such as coal tar and  
404 EA) is notably underestimated by the guidelines, as shown in Fig. 6 (b). This underestimation  
405 can be attributed to the dilation behaviour. Additionally,  $\mu_{\text{phy}}$  for EA-coated is slightly smaller  
406 than that of the rough steel pipe. The reason might be that the stiffness of the coating layer is  
407 much smaller than steel. A volume change in the coating layer would happen, though not large,  
408 due to the limited coating thickness, reducing the contact pressure increase caused by  
409 constrained dilation. Hence, the axial resistance of the EA-coated pipe is slightly lower.  
410 However,  $\mu_{\text{phy}}$  of FBE-coated pipe aligns well with the guidelines' predictions because there is  
411 limited dilatancy at the hard and smooth interface, making its axial pullout behaviour resemble  
412 the assumption of the guidelines.

413 There appears to be a critical hardness of around 35 HRA. The resistance of pipes with  
414 hardness less than this critical value is susceptible to surface hardness, whereas, for pipes with  
415 surface hardness greater than this value, the effect of hardness on ASPI is minimal. This  
416 concept of critical hardness has also been used by [Abuel-Naga et al. \(2018\)](#). It is highlighted  
417 that the above value of critical hardness cannot be generalised because it is also dependent on  
418 other important factors, such as normal pressure, particle angularity, size, and grain size  
419 distribution based on the experimental results on soil-geomembranes by [Frost & Han \(1999\)](#)

420 and Dove & Frost (1999). Particles overall morphology of this sand measured by Liang et al.  
 421 (2021) is given in Table 2 for information.

### 422 *Development of a new method for calculating pipe pullout resistance*

423 Fig. 11 compares the measured values of pullout resistance with the ALA predictions (Eq. (1)  
 424 ) using the experimental results in Wijewickreme et al. (2009), Sheil et al. (2018), and this  
 425 study. The interface friction angle,  $\delta$ , uses the value measured by direct shear testing. Eq. (1)  
 426 cannot predict the axial resistance well, especially for rough pipes buried in dense sands. As  
 427 discussed earlier, this equation neglects the contact pressure increase caused by constrained  
 428 dilation. Additionally, it does not account for the effects of pipe self-weight, as Sheil et al.  
 429 (2018) highlighted. A new equation is proposed to address these limitations as

$$430 \quad T_{\max} = \left[ \pi D (\gamma' H_c \frac{1+K_0}{2} + \Delta\sigma_D') + \Delta W \right] \tan(f_R \phi') \quad (2)$$

431 where three new terms are introduced:  $\Delta\sigma_D'$  is the average increase in contact pressure caused  
 432 by constrained dilation;  $\Delta W$  is the normal force increase per unit length caused by the pipe  
 433 weight; and  $f_R$  is the roughness-dependent soil-pipe interface friction factor. The determination  
 434 of these three terms is explained below:

435 Firstly,  $\Delta\sigma_D'$  can be determined by the linear elastic expanding cylinder theory, as given by  
 436 Houlsby & Italiana (1991):

$$437 \quad \Delta\sigma_D' = 4G \frac{u_r}{D} \quad (3)$$

438 where  $u_r$  is the radial expanding displacement of the shear band due to interface dilation and  $G$   
 439 is the soil shear modulus. This model was further developed to predict the pullout resistance of  
 440 soil nails and pipes in subsequent studies (Luo et al., 2000; Yin et al., 2012; Wijewickreme &  
 441 Weerasekara, 2015).  $u_r$  can be determined as follows:

$$442 \quad u_r = \int_0^{u_c} \frac{du_r}{du} du = \int_0^{u_c} \frac{du_r / S}{du / S} du = \int_0^{u_c} \frac{d\varepsilon_v}{d\gamma} du = \int_0^{u_c} \tan \psi du \quad (4)$$

443 where  $u_c$  is the critical axial displacement where the mobilised dilation angle reaches its peak  
 444 value;  $S$  is the thickness of the shear band;  $d\varepsilon_v$  is the volumetric strain increment, equaling  $du/S$   
 445 for the soil in the shear band;  $d\gamma$  is the shear strain increment, equaling  $du/S$ ;  $\tan \psi (= d\varepsilon_v/d\gamma)$  is  
 446 the dilatancy. Luo et al. (2000) and Yin et al. (2012) assumed that  $\tan \psi$  linearly increases with  
 447 the axial displacement until reaching the maximum dilation angle ( $\psi_{\max}$ ). Under this  
 448 assumption,  $u_r$  can be further calculated by

$$449 \quad u_r = \int_0^{u_c} \tan \psi du = \int_0^{u_c} \frac{u}{u_c} \tan \psi_{\max} du = \frac{1}{2} u_c \tan \psi_{\max} \quad (5)$$

450 Assuming that the mobilised dilation angle and the mobilised pullout resistance reach their  
 451 peak values at the same critical axial displacement  $u_c$ , Eqs. (3) and (5) suggest that

$$452 \quad \Delta \sigma_D' = 2G \frac{u_c}{D} \tan \psi_{\max} \quad (6)$$

453 where  $u_c$  is affected by surface roughness and coating hardness. The method of Audibert &  
 454 Nyman (1977) is employed to determine  $u_c$  from the results in Fig. 4. This method consists of  
 455 a horizontal line through the maximum resistance and a secant line passing through the origin  
 456 and the point of 70% of the maximum pullout resistance. The intersection of these two lines  
 457 determines  $u_c$ . Table 5 summarises the values of  $u_c$  for different pipes in this study and  
 458 Wijewickreme et al. (2009).  $u_c$  for non-coated steel pipes ranges from 0.77 to 2.85 mm,  
 459 showing a positive correlation with the normalised roughness.  $u_c$  for FBE-coated and EA-  
 460 coated pipes are 1.85 and 3.76 mm, relatively larger than those of the non-coated steel pipes.

461 On the other hand,  $\psi_{\max}$  in Eq. (6) can be determined using the method of Bolton (1986),  
 462 which can be simplified as

$$463 \quad \psi_{\max} = A_\psi / a_\psi \left[ I_D (Q - \ln \sigma_f') - R \right] \quad (7)$$

464 where  $I_D$  is relative density;  $\sigma_f'$  is the effective stress at failure;  $a_\psi$ ,  $A_\psi$ ,  $Q$  and  $R$  are empirical  
 465 coefficients. For dense sand at low pressure ( $I_D(Q - \ln \sigma_f') - R > 4$ , common in soil-pipe interaction),  
 466 Chakraborty & Salgado (2010) obtained values of these four coefficients as 0.62, 3.8, 7.1 +

467  $0.75 \ln \sigma'_i$ , and 1, respectively.  $\sigma'_i$  is the initial effective stress, taken as  $\gamma'H_c(1+K_0)/2$  in this  
 468 study. It should be noted that  $\sigma'_f$  is the effective normal stress at the failure, but it is hard to  
 469 determine the effective stress at failure  $\sigma'_f$  in Eq. (7) for engineers. Hence,  $\sigma'_f$  is also  
 470 approximately taken as the average initial effective stress,  $\gamma'H_c(1+K_0)/2$ , in this study. The  
 471 potential error of this approximation on  $\psi_{\max}$  is as small as not more than 4% based on the value  
 472 of  $Q$  and  $R$ .

473 Moreover,  $G$  can be determined by the model of [Oztoprak & Bolton \(2013\)](#) as

$$474 \quad G = \frac{A(\gamma) \cdot p_a}{(1+e)^3} \cdot \left( \frac{\sigma'_i}{p_a} \right)^{m(\gamma)} \quad (8)$$

475 where  $e$  is the void ratio;  $p_a$  is a reference pressure of 100 kPa.  $A(\gamma)$  and  $m(\gamma)$  are empirical  
 476 parameters dependent on the shear strain  $\gamma (= u/S)$ . Based on the values of  $u_c$  in Table 5 and the  
 477 empirical data in interface shear band thickness ([DeJong & Westgate, 2009](#)), the critical  
 478 interface shear strains at which the mobilised pullout resistance reaches their peak values are  
 479 determined to be no less than 10%. It is practical to adopt the empirical values of  $A(\gamma) = 126$   
 480 and  $m(\gamma) = 1$  for  $\gamma = 10\%$ , as suggested by [Oztoprak & Bolton \(2013\)](#).

481 Secondly,  $\Delta W$  can be calculated by the product of the pipe's volume per meter  $V$  and the  
 482 unit weight difference ( $\Delta\gamma$ ) between the pipe and backfilled soil as

$$483 \quad \Delta W = V \Delta\gamma = \frac{\pi D^2}{4} (\bar{\gamma}_{pipe} - \gamma_b) \quad (9)$$

484 In this equation,  $\bar{\gamma}_{pipe}$  is the apparent unit weight of the pipe. It is the ratio between the actual  
 485 pipe weight (the total weight of pipe and fluid contained in the pipe under operating conditions)  
 486 and the volume occupied by the pipe in the soil.  $\gamma_b$  is the bulk unit weight of the soil. While  
 487  $\Delta W$  has a minimal impact on lightweight pipes (such as pipes used in [Wijewickreme et al.](#)  
 488 [\(2009\)](#) and this study), it can increase the normal force by up to 50% for heavy pipes used by  
 489 [Sheil et al. \(2018\)](#).

490 Thirdly,  $f_R$  can be evaluated by the normalised roughness. As mentioned above, the critical  
 491 normalised roughness among smooth, intermediate, and rough interfaces are 0.02 and 0.5,  
 492 respectively (the framework of Paikowsky et al. (1995)). Experimental results by Paikowsky  
 493 et al. (1995) and Lings & Dietz (2005) indicate that the interface friction angles increase  
 494 linearly with the natural logarithm of the normalised roughness for intermediate interfaces and  
 495 remain constant for smooth and rough interfaces. Therefore,  $f_R$  can be calculated as follows:

$$496 \quad f_R = \begin{cases} \alpha + \beta \ln 0.02 & R_n \leq 0.02 \\ \alpha + \beta \ln R_n & 0.02 < R_n < 0.5 \\ \alpha + \beta \ln 0.5 & R_n \geq 0.5 \end{cases} \quad (10)$$

497 where  $\alpha$  and  $\beta$  are the empirical coefficients, whose values are dependent on the specific soil  
 498 and interface in this study and affected by many other factors, such as void ratio, particle  
 499 angularity, and grain size distribution. For the testing material in this study,  $\alpha$  and  $\beta$  are taken  
 500 as 1.13 and 0.20, respectively.

501 Substituting Eqs. (6) and (9) to (2), the new equation can be expressed as

$$502 \quad T_{\max} = \pi D (\gamma' H \frac{1+K_0}{2} + 2G \frac{u_c}{D} \tan \psi_{\max} + \frac{D}{4} \Delta \gamma) \tan (f_R \phi') \quad (11)$$

503 Table 5 summarises the parameters for the new equation, incorporating data from  
 504 Wijewickreme et al. (2009), Sheil et al. (2018) and the present study. Fig. 11 displays the newly  
 505 predicted values based on these parameters. The Root Mean Square Error (RMSE) of the new  
 506 model stands at 4.61, markedly lower than the 7.74 observed in the ALA predictions.

## 507 **Conclusions**

508 A new large-scale experimental system without boundary effects was developed to investigate  
 509 the effects of surface roughness and coating hardness on the ASPI behaviour. The experimental  
 510 setup involved testing five pipes with varying roughness and coating hardness in dense sand.  
 511 Modified FSR sensors were used to measure the distribution and evolution of contact pressures  
 512 on the soil-pipe interface and the earth pressures in the surrounding soils. Based on the

513 experiments, the following conclusions were drawn:

514 The surface roughness significantly affects the pipe pullout resistance. The pullout  
515 resistance of the rough pipe is 1.70-1.85 times higher than that of the smooth pipe under the  
516 same buried condition. This increment is much greater than the value specified in current  
517 design guidelines (i.e., 0.17). Moreover, this increase stems from different mechanisms of  
518 roughness effects. The increase in interface friction coefficient accounts for 72-79%, while the  
519 remaining 21-28% is attributed to the contact pressure increase induced by the interrelated  
520 constrained dilation and soil arching.

521 Soil arching greatly affects the contact pressure between soil and pipes during both  
522 backfilling and pullout processes. During the backfilling, the contact pressures are concentrated  
523 at the shoulders and invert of the pipe due to the initial soil arching. During the pullout process,  
524 an additional soil arching due to the soil-interface dilation and a disturbance due to the loading  
525 determine contact pressure evolution behaviour. For the rough pipe, the former plays a  
526 dominant role. The thick shear band induces further relative displacement, leading to a  
527 noticeable increase in contact pressure at the pipe crown and invert. For the smooth pipe, the  
528 limited soil-interface dilation does not reinforce the soil arching. Instead, the pullout process  
529 disturbs the original equilibrium of the surrounding soil, releasing part of the internal friction  
530 of the initial soil arching. Consequently, the increase in contact pressure is only observed at the  
531 springlines of the smooth pipe.

532 For the test materials in this study, a critical coating hardness of approximately 35 HRA was  
533 identified based on experimental results. When the hardness is below this critical value (e.g.,  
534 EA-coated pipes in this study), the pipes behave similarly to rough pipes due to an equivalent  
535 roughness caused by particle embedment. When the coating hardness exceeds this critical value  
536 (e.g., FBE-coated and raw steel pipes in this study), the influence of hardness seems limited.

537 A new method for predicting pullout resistance was proposed. This method considers the

538 increase in normal force due to constrained dilatant and pipe weight, and it provides a  
539 quantitative estimation of the interface friction angle based on surface roughness. The new  
540 equation demonstrated good model capabilities, as verified using test results from previous  
541 studies and this study.

#### 542 **DATA AVAILABILITY STATEMENT**

543 All data, models, and code generated or used during the study appear in the submitted article.

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548 NOTATION

$A(\gamma), m(\gamma)$	strain-dependent parameters related to $G$
$a_\psi, A_\psi, Q, R$	empirical parameters related to $\psi_{\max}$
$D$	pipe outer diameter
$d$	distance from one end of the pipe test section
$d_{50}$	average particle size
$d\varepsilon_v$	volumetric strain increment
$e, e_{\max}, e_{\min}$	void ratio, maximum void ratio and minimum void ratio
$f, f_R$	soil-pipe interface friction factor and roughness-dependent soil-pipe interface friction factor
$G$	soil shear modulus
$H_c, H_{c0}$	buried depth of the pipe crown and its real value in the experiment
$I_D$	relative density
$K_0$	coefficient of lateral earth pressure at rest
$P$	air pressure inside the flexible pneumatic bag
$p_a$	reference pressure of 100 kPa
$R_{\max}, R_a$	maximum height and average height deviations on the surface profile over a travel length of $d_{50}$
$R_n$	normalised surface roughness based on $R_{\max}$
$S$	thickness of the shear band
$T_{\max}$	pullout resistance, the maximum axial force per meter length
$u, u_c$	axial displacement and critical axial displacement
$u_r$	radial expanding displacement of the shear band due to interface dilation
$\alpha, \beta$	empirical parameters related to $f_R$
$\gamma, d\gamma$	shear strain and shear strain increment
$\gamma', \gamma_b, \bar{\gamma}_{\text{pipe}}, \Delta\gamma$	soil effective unit weight, soil bulk unit weight, apparent unit weight of the pipe and unit weight difference between the pipe and soil
$\Delta\sigma_D'$	the average increase in contact pressure caused by constrained dilation
$\Delta W$	normal force increase per unit length caused by the pipe weight
$\delta, \tan \delta$	soil-interface friction angle and soil-interface friction coefficient
$\mu_{\text{ALA}}, \mu_{\text{dir}}$	soil-pipe interface friction coefficient from the guidelines' prediction and interface direct shear tests
$\mu_{\text{phy}}$	the ratio of the pullout resistance to the assumed contact force per unit length
$\sigma_c'$	nominal vertical pressure at the pipe crown
$\sigma_i', \sigma_f'$	initial effective stress and the effective stress at failure
$\varphi'$	sand internal friction angle
$\psi, \psi_{\max}$	dilation angle and maximum dilation angle

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550

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**Table 1 Interface friction angle factor (ALA, 2001; PRCI, 2009)**

Pipe Coating	$f$
Concrete	1.0
Coal tar	0.9
Rough steel	0.8
Smooth steel	0.7
Fusion-bonded epoxy	0.6
Polyethylene	0.6

**Table 2 Sand and pipe properties**

Soil Properties	
Specific gravity	2.68
Particle size: mm	0.25-0.5
Median particle size, $d_{50}$ : mm	0.375
Maximum void ratio, $e_{\max}$	0.797
Minimum void ratio, $e_{\min}$	0.526
Target relative density, $I_D$ : %	85
Dry unit weight at target relative density, $\gamma'$ : kN/m <sup>3</sup>	17.0
Peak friction angle*: °	39.6
Critical state friction angle*: °	32.8
Particle sphericity†	0.92
Particle roundness†	0.728
Particle roughness†	1.021
Pipe Properties	
Diameter, $D$ : mm	102
Thickness: mm	4
Total length: m	1.25
Testing length: m	0.7
Steel density: kg/m <sup>3</sup>	7930
Young's modulus: MPa	204
Poisson's ratio	0.3

\*Measured by direct shear tests with applied effective normal stresses of 17, 34, 50, and 100 kPa.

†Data from Liang et al. (2021).

**Table 3 Pipe surface conditions and soil-pipe interface properties**

Pipe type	Surface treatment	$R_{\max}$ : mm	$R_n^*$	$R_a$ : mm	Surface material hardness: HRA <sup>†</sup>	Peak friction angle <sup>‡</sup> : °	Critical state friction angle <sup>‡</sup> : °
Smooth	Raw seamless steel pipe	0.015	0.04	0.0024	59.0	18.4	17.3
Intermediate	Turning surface	0.08	0.21	0.0313	59.0	34.7	29.4
Rough	Turning surface	0.38	1.01	0.0944	59.0	37.9	33.5
FBE	Fusion bonded epoxy coated	0.004	0.01	0.0006	44.3	19.2	19.0
EA	Epoxy asphalt coated	0.003	0.01	0.0004	32.6	36.8	33.8

\* $R_n$ : Normalised surface roughness using  $R_{\max}/d_{50}$ .

<sup>†</sup>HRA: Rockwell hardness based on ASTM E18 and ASTM D785.

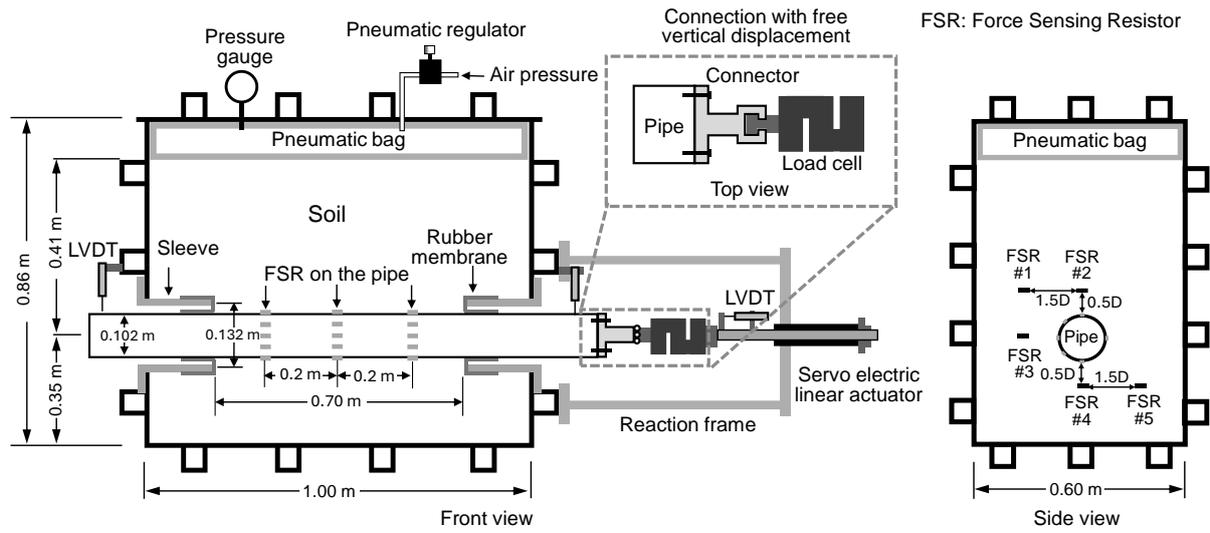
<sup>‡</sup>Measured by interface direct shear tests with applied effective normal stresses of 17, 34, 50, and 100 kPa.

**Table 4. Testing program**

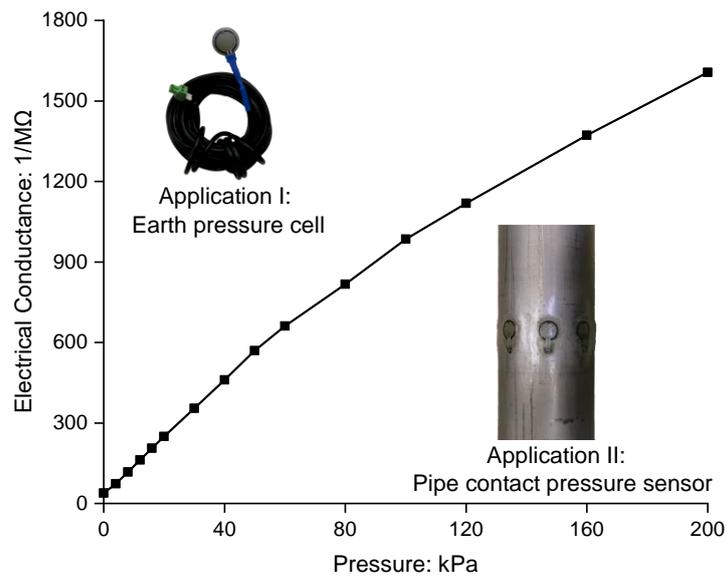
	Reference	Roughness or coating	Nominal pressure at the pipe crown: kPa
<b>Series I</b> Roughness and stress effects	Smooth-17	Smooth steel	17
	Smooth-34	Smooth steel	34
	Smooth-50	Smooth steel	50
	Rough-17	Rough steel	17
	Rough-34	Rough steel	34
	Rough-50	Rough steel	50
	Intermediate-34	Intermediate steel	34
<b>Series II</b> Coating hardness effects	Smooth-34	Smooth steel	34
	EA-34	EA-coated steel	34
	FBE-34	FBE-coated steel	34

**Table 5. Parameters for new equation verification**

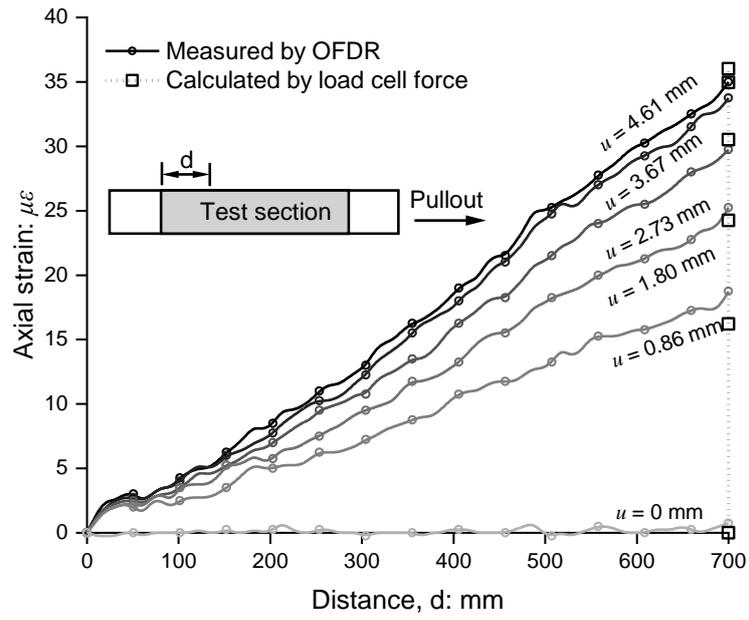
Parameters	This study					Wijewickreme et al. (2009)		Sheil, B. B. et al. (2018)
	Smooth	Intermediate	Rough	FBE	EA	AB-3, 4, 6	AB-5	H1-4
Pipe diameter, $D$ : mm	102	102	102	102	102	457	457	350
Normalised roughness, $R_n$	0.04	0.21	1.01	0.01	0.01	0.84	0.84	0.11
Parameter, $\alpha$	1.13	1.13	1.13	-	-	-	-	-
Parameter, $\beta$	0.2	0.2	0.2	-	-	-	-	-
Interface friction factor, $f$	Eq. (10)	Eq. (10)	Eq. (10)	0.49	0.93	0.81	0.74	0.79
Unit weight difference, $\Delta\gamma$ : kN/m <sup>3</sup>	4.53	4.53	4.53	4.53	4.53	2.43	2.43	27.34
Nominal pressure at the pipe crown, $\sigma_c'$ : kPa	17-50	34	17-50	34	34	17.91	17.53	6.9-50
Internal friction angle, $\phi'$ : °	39.6	39.6	39.6	39.6	39.6	44.5	44.5	37.9
Relative density, $I_D$ : %	85	85	85	85	85	75	20	12
Void ratio, $e$	0.57	0.57	0.57	0.57	0.57	0.7	0.88	0.94
$A(\gamma)$	126	126	126	126	126	126	126	126
$m(\gamma)$	1	1	1	1	1	1	1	1
Critical displacement, $u_c$ : mm	0.77	1.7	2.85	1.85	3.76	2.6	-	-



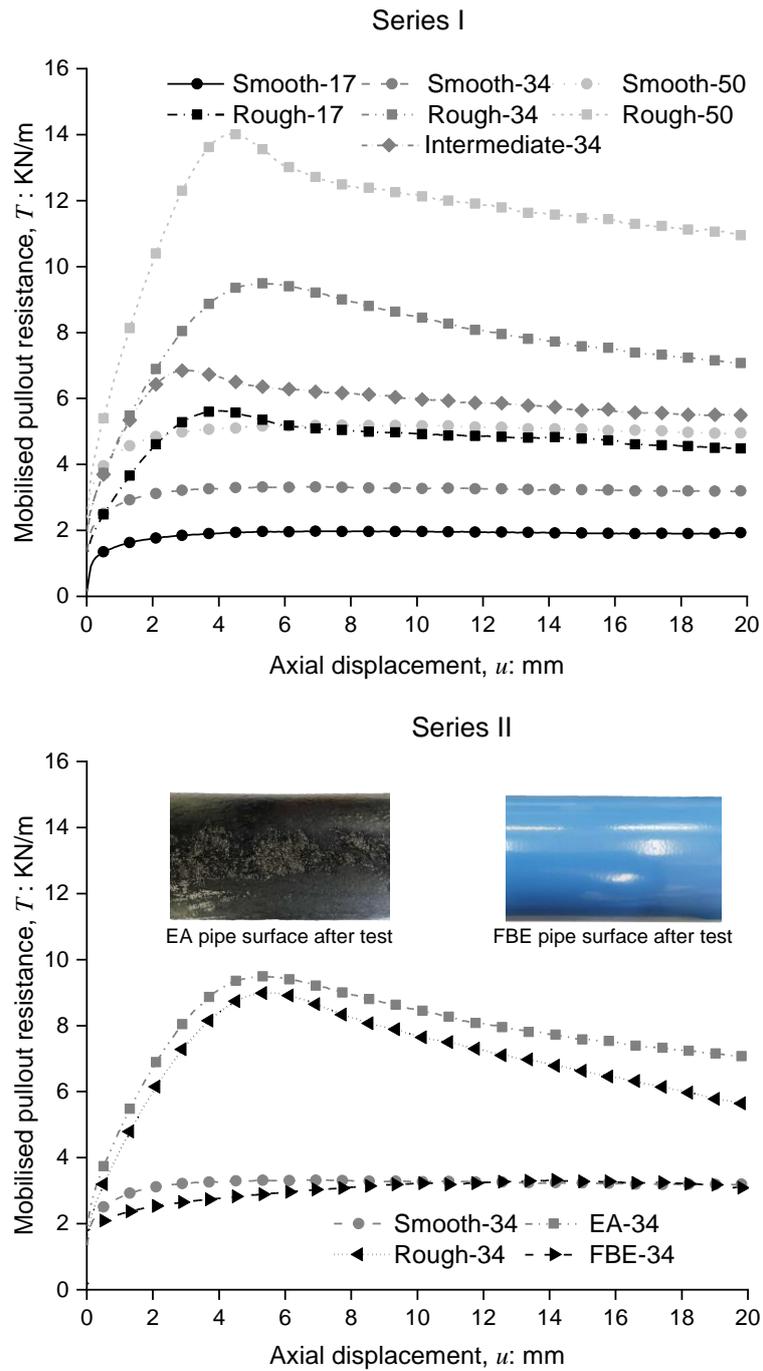
**Fig. 1. Schematic diagram of the experimental system**



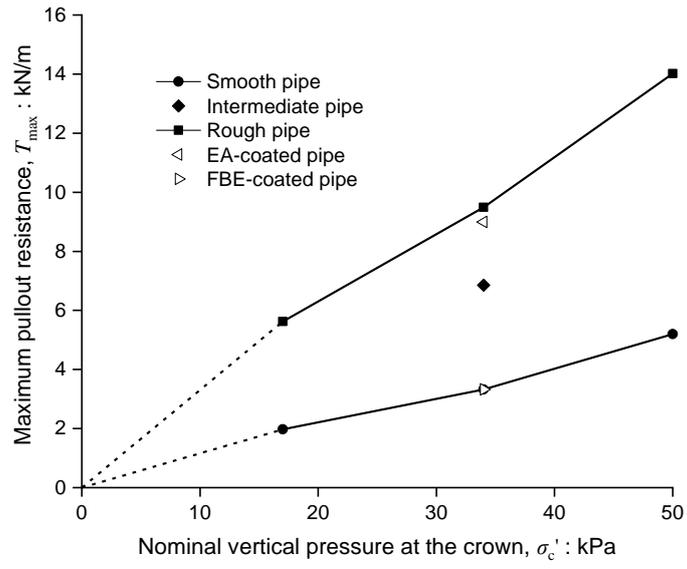
**Fig. 2. Typical calibrated electrical conductance-pressure relationship of FSR sensors**



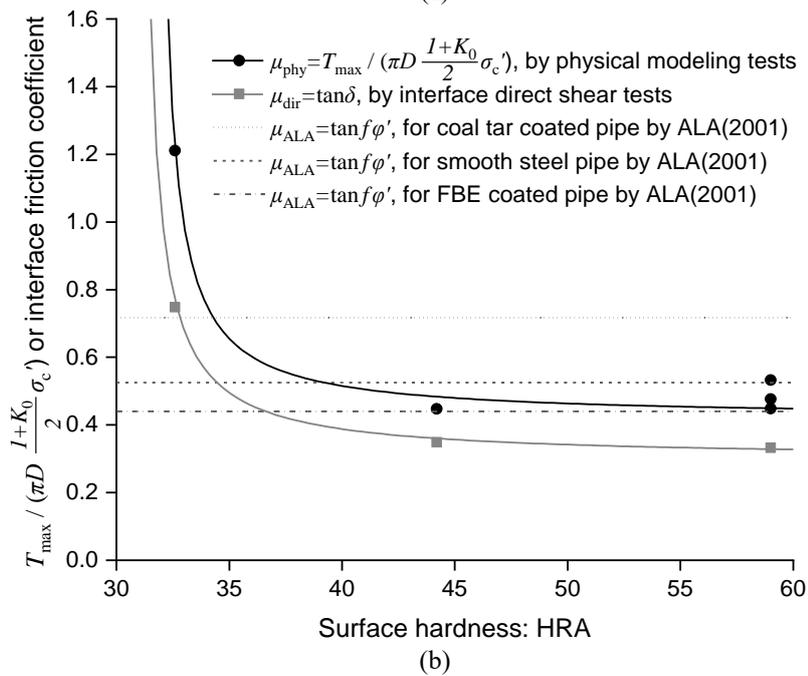
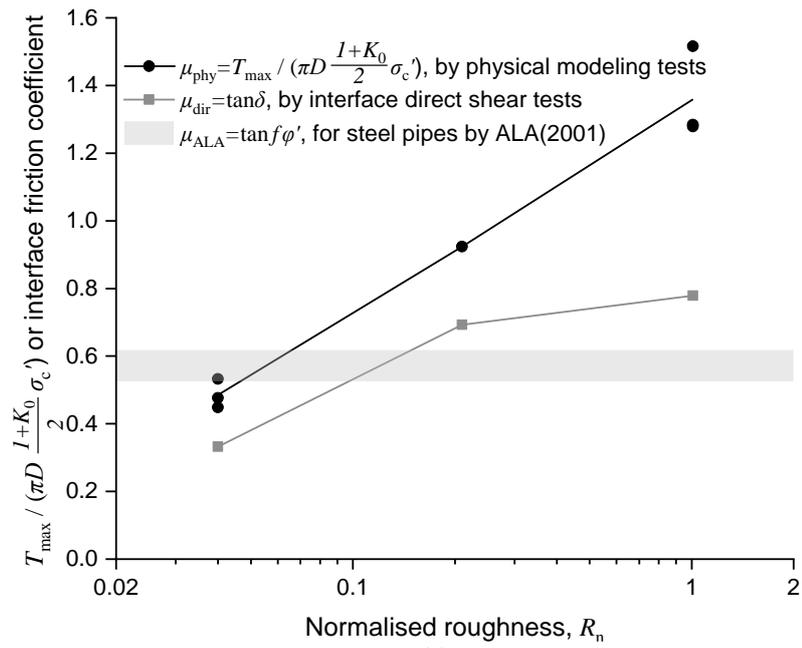
**Fig. 3. Axial strain distribution in Rough-50 measured by OFDR (optical frequency domain reflectometry) with different axial displacements ( $u$ )**



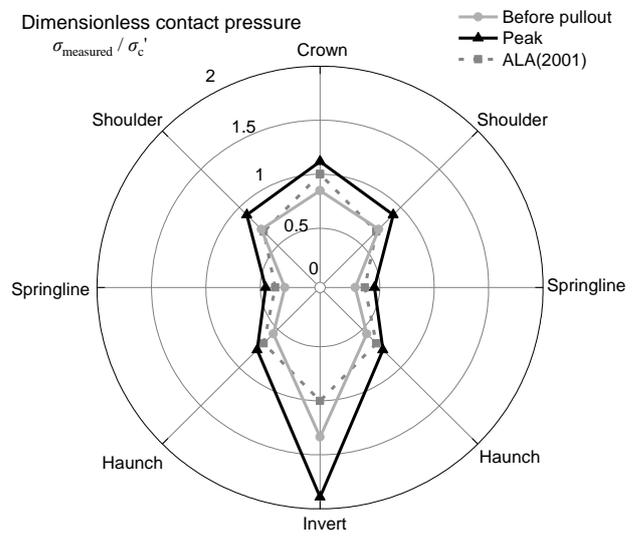
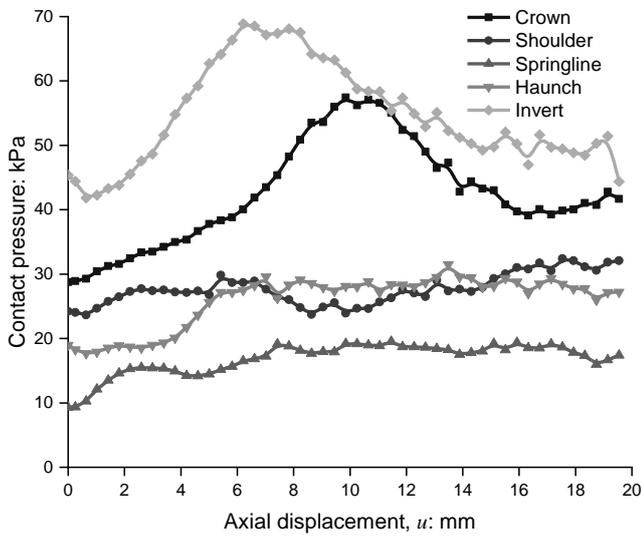
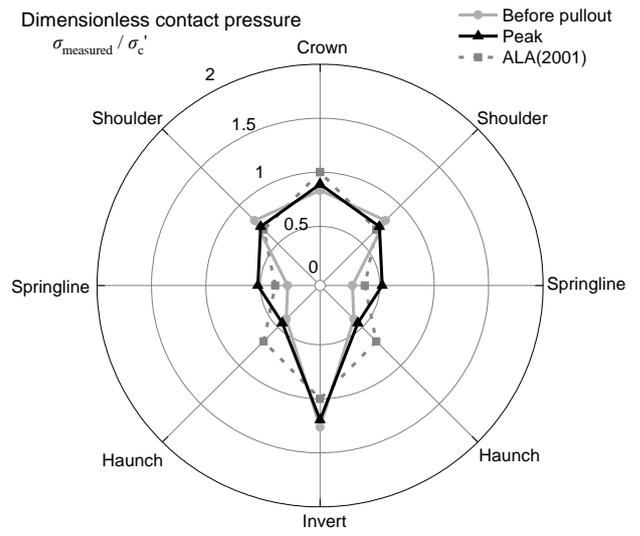
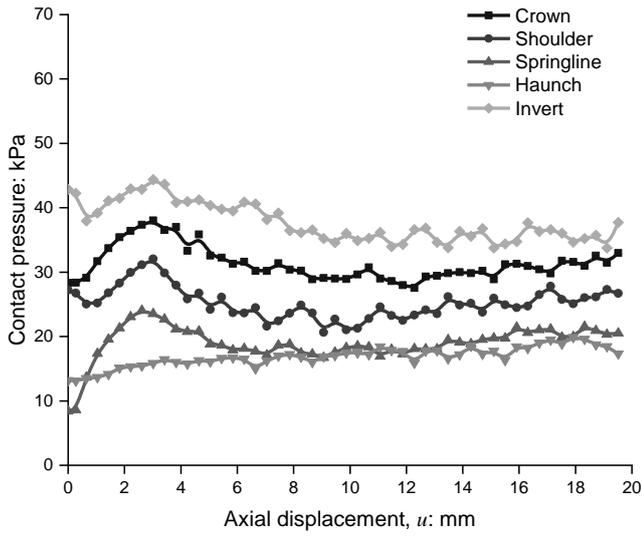
**Fig. 4. Axial force-displacement relationship**



**Fig. 5. Nominal vertical pressure at the crown against the pullout resistance**



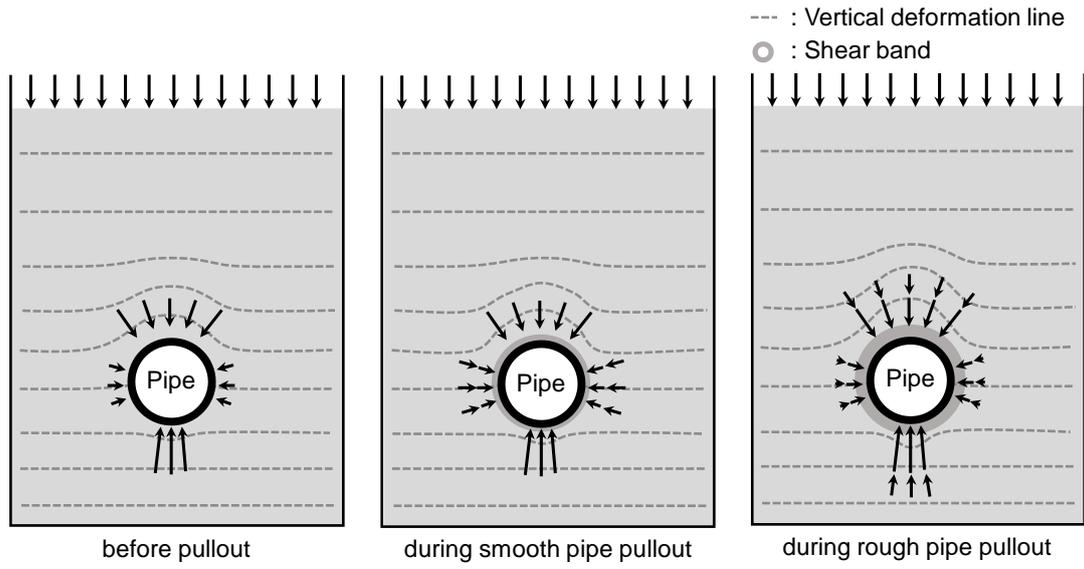
**Fig. 6. Comparison between  $\mu_{\text{phy}}$  and interface friction coefficient: (a) normalised roughness; (b) surface hardness**



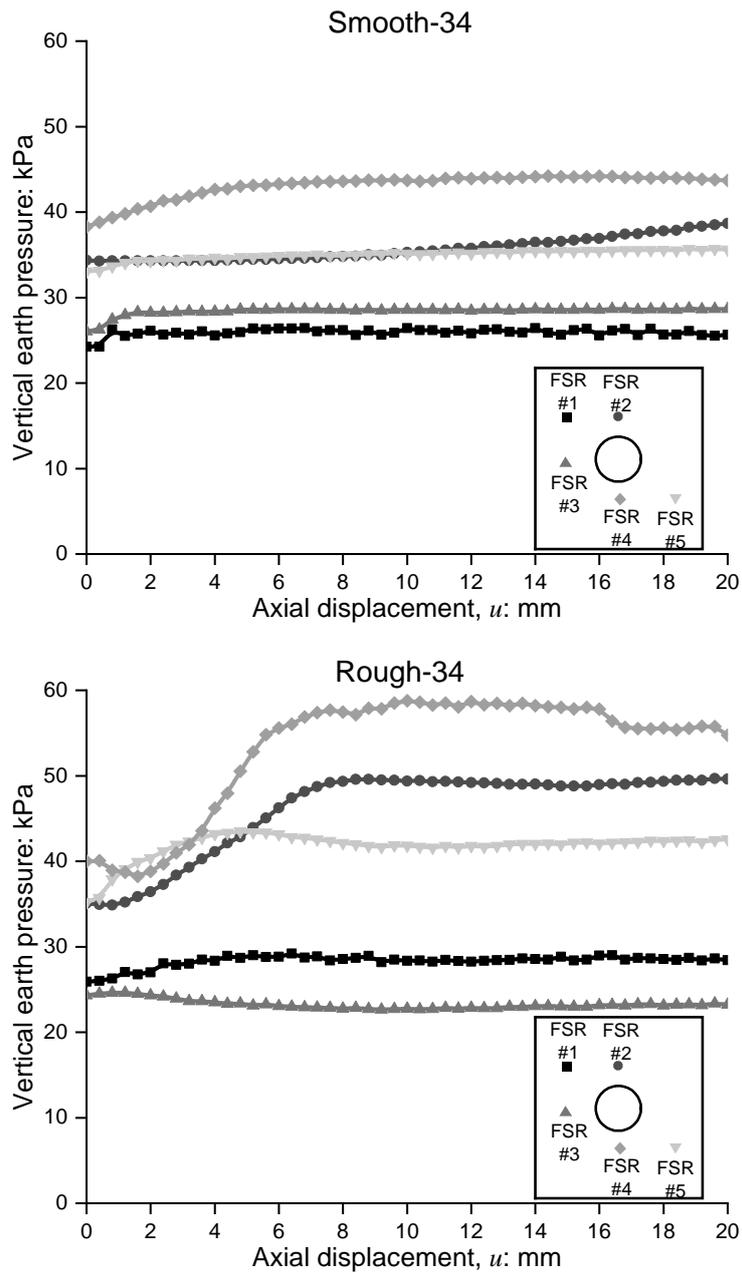
(a)

(b)

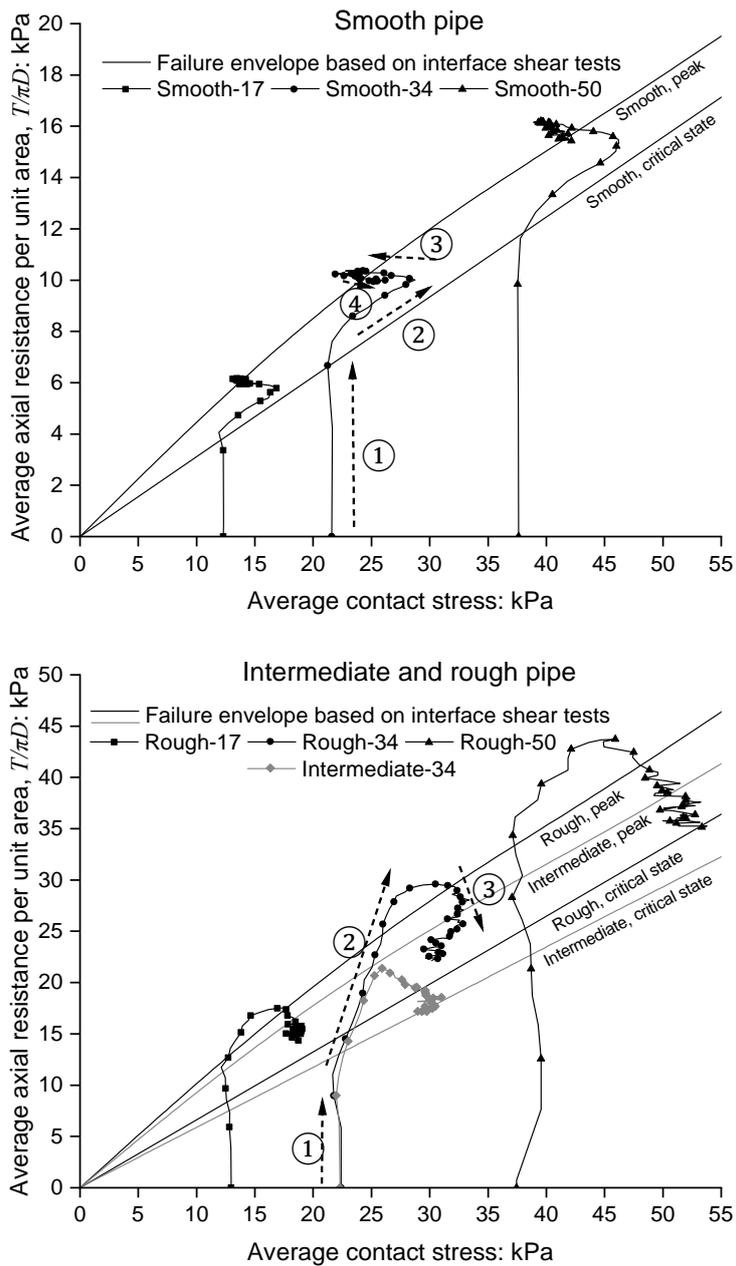
**Fig. 7. Contact pressure development and distribution: (a) Smooth-34; (b) Rough-34**



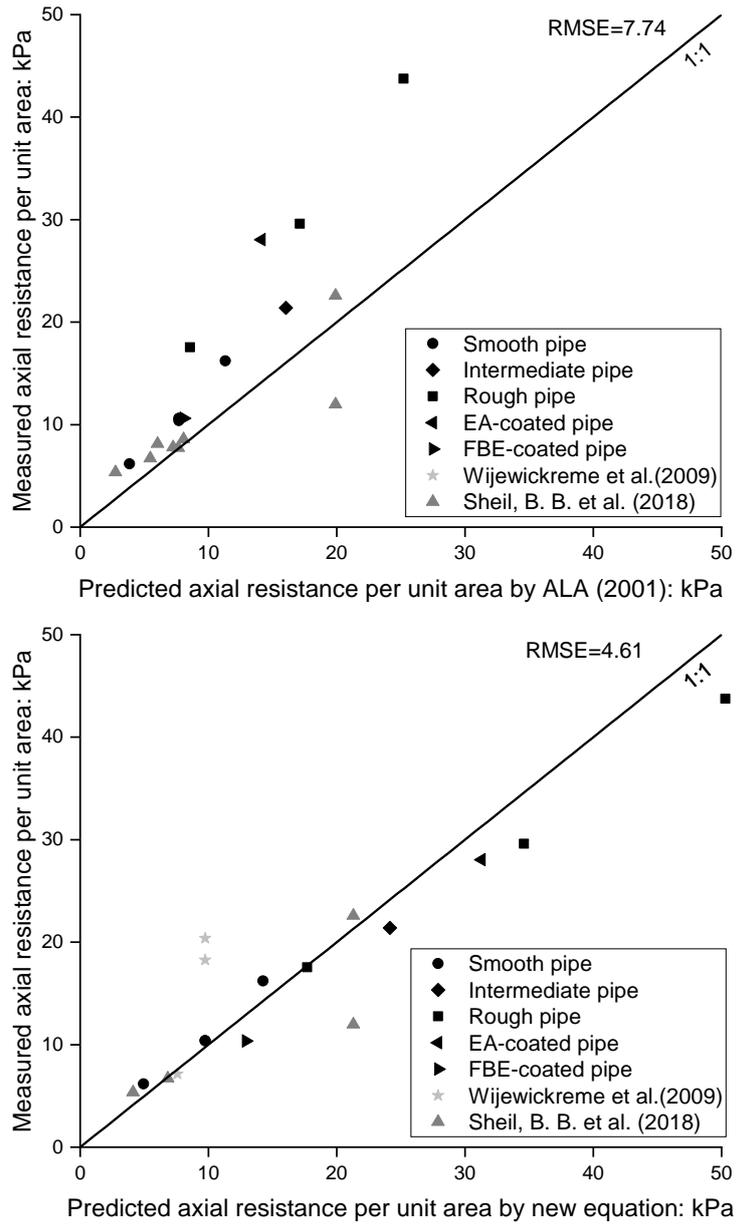
**Fig. 8. Schematic diagram of soil arching effect on pipes**



**Fig. 9. Vertical earth pressure development**



**Fig. 10. Stress path based on average contact pressure measured by FSR**



**Fig. 11. Prediction of pullout resistance**