

1 Energy Conservation through Flexible HVAC Management in Large Spaces: 2 An IPS-based Demand-driven Control (IDC) System

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13 **ABSTRACT:**

14 Buildings consume substantial amounts of energy and require sophisticated control strategies to
15 fulfill occupants' comfort requirements. In large spaces, various occupancy patterns result in
16 uneven load distributions, requiring high-resolution occupancy information for sufficient system
17 control. In recent years, the development of indoor positioning systems (IPS) enabled the
18 possibility of more scientific and precise occupancy detection systems, leading to better operation
19 of buildings' HVAC systems. This paper proposes a demand-driven control system for air
20 conditioner control in large spaces based on IPS. The proposed system focuses on optimizing the
21 ventilation rate based on number of occupants and their spatial distribution in an experimental
22 space. A dual-network (Wi-Fi network and BLE network) indoor positioning system is installed to
23 collect the occupancy data and guide the operation of Variable-Air-Volume (VAV) boxes. The
24 energy-saving potential of the proposed system is examined with a computational fluid dynamics
25 (CFD) model in terms of temperature distribution and energy consumption. This study also
26 explores the interrelationship between cooling load variation and occupancy pattern under
27 different control mechanisms. The final results show the proposed system has significant energy-
28 saving potential by avoiding over-cooling in unevenly distributed occupancy conditions.

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30 **Key words:** Energy efficiency, Demand-driven control, Occupancy, Indoor positioning system,
31 Energy conservation

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33

34 **Nomenclature**

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Nomenclature

$RSSI$	The received signal strength indicator
α	A constant value when determining $RSSI$
d	The distance between the transmitter and the receiver
R	The Gaussian random error with zero mean
$RSSR_{si}$	The received signal strength of reference points
s	The index of the signal source
i	The index of the reference point
p	The order of distance
θ	The shape factor of the distribution
r	The rank of the reference point
P_i	The probability of the occupant being located close to reference point i
t, τ	The index of time
$w(t)$	The time development of the probability density function after several time steps
x_t, x_τ	The vertical zone index at time t, τ
y_t, y_τ	The horizontal zone index at time t, τ
x_i, y_i	The vertical and horizontal zone index of reference point i
$x_{\tau,i}, y_{\tau,i}$	The vertical and horizontal zone index of reference point i at time τ
$W_\tau(x_\tau, y_\tau)$	The transition function of random process
η	The constant value of probability that occupant remains at the same zone
T_S	The temperature of supplied air from VAV box
T_{OUT}	The ambient air temperature
T_{IN}	The indoor temperature in a zone
T_V	The air temperature of ventilation
m_I	The mass flow rate of infiltration air in a zone
m_V	The mass flow rate of ventilation air in a zone
M_k	The mass flow rate of air provided by VAV box k
Q	The total heat flux in a room
z	The index of zone
Q_z	The total heat flux in a zone z
Q_I	The heat gain to zone due to infiltration
Q_V	The heat gain to zone due to ventilation
Q_G	The internal heat gain to zone by occupant, equipment, etc.
Q_{ADJ}	The heat gain to zone due to air flow from adjacent zone, etc.
Q_S	The heat gain to zone from all surfaces
N_O	The number of occupants in the zone
N_C	The number of computers in the zone
N_L	The number of lamps in the zone
W_O	The total fixed heat power of one occupant per hour
W_C	The total fixed heat power of one computer per hour
W_L	The total fixed heat power of one lamp per hour
W_z	The total fixed heat power in a zone z
n_1	The ballast consumption coefficient
n_2	The illumination shade insulation coefficient
C_{Air}	The specific heat capacity of the air

36 1. INTRODUCTION

37 In recent years, energy conservation has become a major objective in every country's sustainability
38 efforts. Buildings have been reported as the largest energy consumers in cities [1], such that the
39 building sector accounts for about 40% of energy use in Europe [2], 28% in China [3], and about
40 39% in the UK [4]. Among commercial buildings, HVAC systems consume the largest portion of
41 energy from the grid and account for 48%, 55%, and 52% of energy usage in the US, the UK, and
42 Spain, respectively [4]. The US Department of Energy (DOE) also highlights HVAC systems as
43 the major target of building energy efficiency measurements [5]. Therefore, the energy
44 performance of HVAC systems is the key to improving efficiency in buildings [6]. In commercial
45 buildings, it is common to find large spaces partially occupied or unoccupied with the HVAC
46 system running at full capacity for significant periods during the course of a typical business day
47 [7]. It is frequently observed that some parts of a building's space are densely occupied, while
48 other parts are empty. Because of the uneven distribution of occupants, it is common to find that
49 some zones are over-cooled, while other zones are insufficiently cooled, causing occupants thermal
50 discomfort. Under these conditions, energy can be significantly wasted, especially within
51 unoccupied zones. Therefore, it is necessary to implement a more efficient and smarter operating
52 mode for HVAC systems [8]. Recent studies suggest a lack of accurate occupancy information is
53 the major source of uncertainty hindering effective HVAC control [9]. Many existing positioning
54 technologies, such as Radio Frequency Identification (RFID) [10], Ultra-wideband (UWB) [11].
55 and Inertia Measurement Units (IMUs) [12,13], count and track occupants inside buildings for
56 higher positioning accuracy, but these technologies require large initial capital investments, new
57 infrastructures, and suitable control mechanisms [14]. Normally, the detected occupants' location
58 cannot be directly coordinated in HVAC systems due to thermal zone-based control designs [15].
59 Therefore, it is necessary to investigate a proper control mechanism for a suitable occupancy
60 detection system that can identify uneven occupancy in large spaces. Such a control mechanism
61 would result in sufficient thermal comfort, eliminate energy waste, and promote HVAC operational
62 efficiency. To bridge such a research gap, this paper proposes a demand-driven control method
63 based on the spatial distribution of occupants. The spatial distribution of occupants is captured
64 with a novel IPS that reports the occupied meshes in a continuous large space through coupled Wi-
65 Fi and Bluetooth Low Energy (BLE) networks. With a demand-based control mechanism, the
66 Variable Air Volume (VAV) HVAC system can adjust its energy consumption based on real demand

67 and avoid unnecessary energy waste.

68

69 **2. BACKGROUND**

70 **2.1 Variable-Air-Volume HVAC System and Demand-Driven Control**

71 Different from conventional Constant Air Volume (CAV) systems, which supply a fixed airflow
72 rate for given temperatures, a VAV system has multiple VAV boxes that can supply varied airflow
73 rates at a constant temperature. One major advantage of the VAV system is its precise temperature
74 control to meet load demand and avoid waste with more flexible air supply amounts. Because of
75 this, most office buildings are equipped with VAV systems. A VAV terminal unit, also called a VAV
76 box, is the zone-level flow-control device equipped with a calibrated air damper and an automatic
77 actuator. The VAV terminal unit normally connects to a local or a centrally controlled system.
78 Traditionally, the air temperature in a return air duct works as a control signal to adjust air flow
79 into the room by controlling VAV boxes based on the differential between the measured
80 temperature at the return duct and the room temperature set point. For large spaces, such as lecture
81 halls, movie theaters or conference rooms, they normally install multiple, centrally-controlled VAV
82 terminal units. It is very common to find these large-scale spaces partially occupied or even
83 unoccupied with the HVAC system at full capacity during a typical business day. Therefore, the
84 reasonable operation and coordination between VAV terminal units is extremely important for large
85 spaces due to their high ceilings, spacious floor areas, and large number of occupants.

86 In order to enhance and optimize HVAC performance and control, many publications propose and
87 discuss several control strategies without considering occupancy distribution information
88 [16][8][17][10], which may or may not avoid a redundant cooling/heating supply or ignore places
89 in demand by bypassing actual occupancy patterns. Demand-driven control is a demand-side
90 management tool to enable the proper operation of HVAC systems [18]. Some simulation-based
91 research studies show potential energy savings from demand-driven HVAC operations vary from
92 10% to 60% [19]. This control mechanism aims to use the actual energy load information to
93 improve control accuracy and eliminate unnecessary waste. For example, the demand-driven
94 control system determines the volume of conditioned air in a thermal zone based on occupancy
95 information (such as the number of occupants) and environmental information (such as
96 temperature and humidity), rather than fixed operating schedules. Therefore, applying demand-

97 driven control systems to multi-zone VAV controls in large-scale rooms shows great potential for
98 optimizing HVAC operation. Lin and Claridge proposed a temperature-based Days Exceeding
99 Threshold-Toa (DET-Toa) method to detect persistent small increases or decreases in the normal
100 building energy consumption [20]. Zeng et al. developed a predictive model of HVAC energy
101 consumption and a data-driven approach to optimize the temperature and air static pressure setting
102 point [21]. In a more recent study, Zhou et al. proposed a supervisory demand-based temperature
103 control system. In this research, the primary VAV box and secondary VAV box differentiate the
104 occupied and unoccupied zones based on the temperature measured at the breathing level [17].

105 The conventional demand-control systems include occupancy-driven demand control [22],
106 temperature-based demand control [17][20], and CO₂-based demand control (mostly in DCV
107 systems) [23]. Each control mechanism is subject to limitations. A temperature-based demand
108 control mechanism adjusts the supplied airflow rate based on nominal temperature or the
109 temperature difference between the supply air and setting values. However, occupancy derived
110 from temperature fluctuations may not be accurate because temperature fluctuations are not
111 necessarily caused by occupants; therefore, such a system yields huge potential for errors in
112 unoccupied zones. CO₂-based demand control compares CO₂ concentrations between supply and
113 return ducts to determine the number of occupants in a space. Many demand-driven strategies are
114 based on coarse occupancy detection by balancing the CO₂ concentrations of supply air, return air,
115 and outdoor air, although it is costly and difficult to accurately estimate the actual occupancy
116 pattern [23][24]. However, similar to temperature-based methodology, CO₂ concentration is an
117 indirect reflection of occupancy. Due to the time delay of CO₂ reaching its equilibrium and a
118 nonlinear relationship with the number of occupants, CO₂ is a dubious occupancy indicator.
119 Therefore, this research develops a fine-grained occupancy detection approach based on an Indoor
120 Positioning System (IPS) to collect occupants' spatial distribution to enable the proposed IPS-
121 based demand-driven control (IDC) approach and study its energy-saving potential. The IDC
122 HVAC control mechanism is built on the new occupancy measurement framework that represents
123 large spaces as small, occupied patches based on data collected from the IPS. The operation of
124 VAV boxes is determined by the distribution of occupied patches. Temperature sensors are also
125 installed at the breathing level, instead of the return ducts, to measure the temperature around
126 occupants. Only occupied zones will receive cooling air until they reach a sufficient thermal
127 comfort level. Therefore, accurate occupancy detection is the premise of proper demand-driven

128 system control. The ASHRAE Guide 14 suggests ideal occupancy identification for a building
129 model should have an hourly Mean Bias Error fall within $\pm 10\%$ and an hourly Cumulative
130 Variation of Root Mean Square Error fall below 30% [25,26]. In a CO₂-based occupancy
131 prediction model, a 3 or 4-tolerance (10%-13% for about 30 occupants) away from actual
132 occupancy could be acceptable if over 80% of accuracy for occupancy detection guaranteed [27].

133

134 **2.2 Building Occupancy and Indoor Positioning Systems**

135 Occupancy information can serve as inputs for energy simulations and assist facility managers in
136 optimizing HVAC system operation to provide sufficient thermal comfort. Therefore, accurate
137 building occupancy detection is the premise of efficient HVAC system control and design [28].
138 Building occupancy detection requires retrieving information at various resolution levels.
139 Christensen et al. defined "occupancy resolution" as a three-dimensional system (i.e., temporal
140 resolution, spatial resolution, and occupant resolution) [29]. Other researchers have summarized
141 occupancy resolution using different scales, such as present, count, location, track, identity, and
142 behavior [30]. Higher data resolution and accuracy enables more sophisticated building energy
143 management strategies.

144 Many researchers have proposed studies on schemes to determine occupancy profiles, and most
145 studies still obtain building occupancy schedules based on assumptions or experience with
146 occupancy models. The American Society of Heating, Refrigerating, and Air-Conditioning
147 Engineers (ASHRAE) standard 90.1-2007 [31] recommends approximate occupancy diversity
148 factors for different building types or zones by hourly occupant distribution within different day
149 types in order to standardize building simulation and analysis when actual data is unknown. It has
150 been observed that the difference between actual occupancy and standardized occupancy schedules
151 recommended in the ASHRAE can be as high as 40% when comparing occupancy schedules in
152 terms of days, weeks, months, and holidays [32]. This discrepancy might cause mismatching
153 between actual energy costs and simulated energy costs in buildings. Therefore, researchers have
154 proposed many approaches to collect reliable occupancy information. These approaches fall into
155 two categories: (1) simulation-based occupancy models and (2) direct monitoring through sensors.
156 Simulation-based occupancy models, such as the agent-based model and Markov models, estimate
157 or predict occupancy based on historical data and analytical analysis; however, their accuracy

158 relies significantly on abundant historical data collections and data mining processes and normally
159 provides low-resolution information, such as the number of occupants and a rough estimate of the
160 duration of the occupants' stay in certain spaces [33][34][35]. Although these model-based and
161 data-mining process methodologies provide insight to facilitate analysis and prediction of
162 occupancy profiles, their outcomes are seldom used for building facility operations. On the other
163 hand, many researchers utilized ambient sensor systems to directly monitor occupancy, such as
164 CO₂-based detection systems [36][37], infrared sensors [38], and RFID [10]. Among those
165 systems, occupancy sensors are widely used for detection in lighting and HVAC systems because
166 they are suitable in cases when only the on/off (occupied or unoccupied) status needs to be detected,
167 potentially leading to inaccessibility of the number of occupants with a timestamp. Another
168 popular methodology is CO₂-based detection systems, which are widely applied in building
169 ventilation control systems and show good results in occupant number prediction for whole
170 buildings. CO₂-based detection determines the ventilation demand using a balance equation
171 between CO₂ concentration of supply air and return air duct [19,23,39]. A study conducted by
172 Jiang et al. [27] suggests the estimation accuracy of CO₂ concentration is often less than 50%.
173 This approach is bound with limitations like time delays, high costs, and inaccuracy due to indirect
174 detection [40]. Also, several researchers use lighting sensors or light switch on/off actions to report
175 the duration of time an occupant spends in one room, or they use building envelope actions
176 (window shades) and building electricity use variance to reversely ratiocinate occupancy [41],
177 possibly causing very low-resolution results [42]. Several researchers combined CO₂ sensors with
178 temperature, humidity, lighting, and sound sensors and reported accuracy ranging from 75 to 84.5%
179 [43,44]. Infrared sensors and motion sensors are often utilized to detect the events of "occupied to
180 vacant" or "vacant to occupied" in single-person office rooms without detecting the number of
181 occupants at an accuracy of 46% [38,45]. RFID technologies were reported with a higher detection
182 accuracy of 88% for stationary occupants and 62% for mobile occupants [10].

183 A fully functional IPS can retrieve real-time locations and identify objects in indoor areas. Many
184 researchers have introduced similar systems for the study of built environments [10][46][47] and
185 focus on the building occupancy presence, number of occupants, and building energy variances.
186 However, it is extremely challenging to capture occupancy distribution in real time and identify
187 unique identities of occupants. In previous research, Wi-Fi and BLE positioning networks show
188 great potential for commercial application [18][48][49] at low cost. Given their popularity, Wi-Fi

189 infrastructures installed in most buildings become the most effective existing signal network with
190 the promise of minimum cost [14]. However, Wi-Fi signals have significant issues in terms of
191 stability when close to obstacles, metals, and building separations. It is impractical to merely rely
192 on a Wi-Fi network for reliable positioning data. Therefore, we introduced another layer in the
193 form of a Bluetooth Low Energy (BLE) network to cross reference the signal fingerprints of Wi-
194 Fi networks. BLE technology is a cheap (less than 20 USD per beacon station and a cell phone can
195 be the tag), portable, and controllable signal network applicable in indoor location acquisition and
196 information broadcasting with location tags. This research intends to improve the operation of
197 HVAC systems based on a new IPS-based occupancy acquisition approach. Therefore, we chose
198 both Wi-Fi and BLE networks to construct the indoor positioning coordination and draw the
199 occupancy distribution. Since most HVAC systems are controlled using thermal zones and VAV
200 box affect areas, it is not necessary to have high positioning precision to enable demand-driven
201 control based on occupants' spatial distribution. More specifically, we utilized the dual network
202 IPS to locate occupants in space meshes and use these meshes to quantify the spatial distribution
203 of occupants. With such information, we can optimize the HVAC operation and minimize energy
204 waste.

205

206 **3. METHODOLOGY**

207 **3.1 Positioning System and Space Meshing**

208 In a large space with Wi-Fi and BLE networks installed, each location inside the space has a unique
209 combination of received signal patterns from the Wi-Fi and BLE networks. Given this property,
210 the k-nearest neighbors (kNN) algorithm is adopted to estimate the most likely location based on
211 signal strength. Since the movements of occupants are continuous, the future location of an
212 occupant is based on his or her current position. Therefore, a random walk algorithm is also
213 embedded in the proposed positioning system. The system assumes occupants can only move to
214 an adjacent zone close to the current location. Figure 1 shows the proposed positioning system that
215 integrates both the kNN and random walk algorithms.

216

217

Insert Figure 1 about here

218

219

220 (1) *Signal Measurement of kNN algorithm*

221 A signal receiver can sense radio signals in the surrounding environment. Since the source of
222 signals could vary in frequency or voltage, received signal strength indicators (RSSIs) are typically
223 used as a quantitative representation of signal strength. *RSSI* is an integer value measurement of a
224 received radio signal that complies with the IEEE 802.11 standard. Based on this protocol, BLE
225 and Wi-Fi networks can adopt the distance estimation metric proposed by Texas Instruments of
226 2.4 GHz radio signal as

227

$$RSSI = \alpha - 10 \cdot n \cdot \log_{10}(d) + R \quad (1)$$

228

229 where power level is measured by *RSSI* in dBm; *d* is the distance between the transmitter and the
230 receiver; the term *R* denotes a Gaussian random error with a zero mean caused by shadowing; α
231 is a constant that depends on several factors, such as averaged fast and slow fading as well as
232 transmitted power (α can often be determined beforehand); and *n* is the signal propagation
233 constant reported in the device manual.

234 The k-Nearest Neighbors (kNN) algorithm is a non-parametric pattern recognition approach for
235 classification purposes. The algorithm compares the “distances” between received signal strengths
236 and all reference points to determine the closest reference point as the rough location of the receiver.
237 There are many measurement metrics that can represent the “distance” between the reference
238 points and receivers. To generalize the estimation, this research adopts the Minkowski metric (or
239 *p* norm) as the measurement tool. The Minkowski distance can be calculated using the following
240 equation:

241

$$Dis(RSSI, RSSR_{si}) = \sqrt[p]{\sum_{s=1}^S (RSSI - RSSR_{si})^p} = \|RSSI - RSSR_{si}\|_p \quad (2)$$

242

243 where $RSSI$ is the received signal strength of the receiver, $RSSR_{si}$ is the received signal strength
244 of reference points, s is the index of the source, i is index of reference points, and p is the order of
245 distance.

246 Once all the distances are calculated, the kNN algorithm ranks the distances in increasing order.
247 Shorter distances, or a higher rank, mean the receivers are closer to the given reference point. In
248 our model, $k = 10$, which means the top ten reference points with the shortest distances will be
249 selected as potential location candidates. Then, the probability of a receiver at the location of
250 certain reference points is assigned based on the kNN rank, which follows a geometric distribution,
251 where r is the rank of that reference point and θ is the shape factor of the distribution in our
252 algorithm $\theta = 0.5$.

253

$$P_i(r) = (1 - \theta)^r \cdot \theta \quad (3)$$

254

255 P_i is the probability of the occupant being located close to reference point i . Then $P_r(x, y)$, the
256 probability of an occupant located in a zone, is calculated by averaging all P_i 's in that zone. The
257 row and column indices of a zone are x and y . For example, the probability of an occupant's
258 location being in zone 2, or $P_r(1,2)$, is averaged from the P_i 's of reference points R2, R3, R9, R15,
259 and R16.

260 (2) Space Meshing and Random Walk

261 In commercial buildings, large rooms are normally divided into multiple thermal zones for the ease
262 of HVAC system operation and interference. Each thermal zone has multiple independent VAV
263 boxes or other air conditioning devices to enable flexible operations. The separation of thermal
264 zones and corresponding subsystems allows facility managers to provide different levels of thermal
265 comfort in regards to space occupancy status. This study takes the conditioned air supply in a large
266 office room as an example to explore the possibility of implementing a demand-driven control
267 mechanism based on high-resolution occupant distribution. This type of occupancy information is
268 collected from the proposed IPS and reflects uneven spatial cooling demands. Therefore, instead
269 of coordination, the proposed IPSs are specially designed for space patches/meshes. All inner space
270 is meshed into small space patches for three major benefits: (1) ease in HVAC operation, (2) high

271 detection accuracy tolerance, and (3) a simpler positioning algorithm. Once the spaces are meshed,
 272 an occupant's movement is modeled as a random walk on a 2D lattice.

273 In any enclosed space, occupants enter from specific entrances and walk toward their targets.
 274 Although their destinations are unclear, their movements are continuous, and the next location of
 275 their movements must be close to their current location. This fact enables us to exclude the
 276 possibility of zones far away from the current zone when predicting in which zone an occupant
 277 will be located at the next time step. Therefore, in addition to the kNN algorithm, the proposed
 278 algorithm also introduces a possibility estimation approach to predict the occupant's location based
 279 on the random walk theory. For each time step, the occupant will move to another zone or remain
 280 in the same zone. The time development of the probability density function $w(t)$ after several time
 281 steps from 0 is given by

282

$$w_t(\tau) = W(x_t \rightarrow x_\tau, y_t \rightarrow y_\tau) \cdot w_t(t) \quad (4)$$

283 where $W_\tau(x_\tau, y_\tau)$ is the transition function of the random process.

284 The equation represents the discretized time-development of one step as

285

$$w_t(\tau) = W_\tau(x_\tau, y_\tau) \cdot w_t(\tau - 1) \quad (5).$$

286

287 This random process has a transition functions as

288

$$W_\tau(x_\tau, y_\tau) = \begin{cases} \frac{1 - \eta}{\sum_i (|x_{\tau,i} - x_i| + |y_{\tau,i} - y_i|)}, & \text{if } |x_{\tau,i} - x_i| - |y_{\tau,i} - y_i| = 1 \\ \eta, & \text{if } x_i = x_{\tau,i} \cap y_i = y_{\tau,i} \\ 0, & \text{else} \end{cases} \quad (6)$$

289

290 where x_τ and y_τ are the vertical and horizontal zone indices at time τ , and η is a constant that
 291 suggests the probability that the occupant remains at the same zone at next time step $\tau + 1$. The
 292 final location for the time step τ is determined by maximizing the product of both the kNN

293 probability and the transitional probability.

294

$$\arg \max_{x \in Z} P_r(x) \cdot W_r(x), \quad \forall x \quad (7)$$

295

296 **3.2 Field Tests and Tracking Accuracy**

297 To investigate whether the proposed occupancy detection approach could effectively obtain highly
298 accurate occupancy data, the research team conducted a field experiment in a signal-covered space
299 inside an institutional building. The selected testbed is an open space without any separations so
300 the possible interference caused by internal walls can be mitigated. The space is part of a public
301 lobby in the AC3 building of City University of Hong Kong. There is no wall or separation inside
302 the space that may cause sudden signal strength depreciation. The testing space was marked and
303 divided into a 6 by 6 grid with 36 zone patches (1 meter by 1 meter) and 85 location nodes
304 (presented as Rxx). Figure 2 shows a picture of the experimental space and the network settings.

305

306

Insert Figure 2 about here

307

308

309 In the space, the signals for three access points were detected without knowing their locations.
310 Four iBeacons were installed at nodes R25, R22, R61, and R64 to generate Bluetooth Low Energy
311 (BLE) networks. To construct positioning coordinates, the signal strength of all three access points
312 and the four iBeacons was collected for each node on the grid. These nodes served as reference
313 points (RF) to locate occupants in the future. There was a column at the location of a reference
314 point (R53) in the experiment space, so no signal was collected for R53, R47, and R60. The signal
315 strength of these three reference points was derived by averaging the signal strength of surrounding
316 reference points. Each reference point was measured three times to minimize random error. Three
317 research assistants participated in the preliminary experiment, each equipped with a data logger
318 that recorded the signals from all seven signal emitters. For the accuracy test, all research assistants
319 walked through the space and covered all zones. Each assistant randomly selected a location inside

320 each zone and recorded the signal strength of the APs and iBeacons.

321 After the preliminary experiment, two groups of data were archived for detection system
322 construction and validation. The first data group (construction group) included the signal strength
323 at the extract location of each reference point. This data was used to train the positioning algorithm
324 and construct a reference grid with coordinates (i.e. the signal “fingerprints”). The second data
325 group (validation group) included experimental data collected by the research assistants. Instead
326 of reference nodes, each assistant collected a set of data at a random location inside each zone. The
327 signal was collected in a manner allowing the determination of the location of the assistant and
328 validation of the positioning accuracy. Figure 3 shows a comparison between the walking path of
329 a sample test and its predicted probability spectrum based on Hamming distance. The top two
330 images of Figure 3 are ground truth of a trail movement; the bottom two images of Figure 3 show
331 the predicted position with probabilities (lighter color suggests higher probability). Our proposed
332 algorithm will select the zone with the highest probability as the predicted location.

333 -----

334 Insert Figure 3 about here

335 -----

336

337 To determine the accuracy performance of the proposed IPS system, we summarized its accuracy
338 in Table 1. The accuracy is calculated by the total number of correct predictions in each timestamp
339 divided by the total number of timestamps.

340 -----

341 Insert Table 1 about here

342 -----

343 Table 1 compares the performance of different distance metrics for the kNN algorithm and shows
344 the chronic accuracy of the model. From the results, the City Block distance and the third order
345 Minkowski distance show the highest accuracy, and the City Block distance is selected for system
346 control algorithm development. At the same time, chronic accuracy shows the error development
347 is constrained to some degree, and the accuracy is converged/stable. These results suggest there is

348 no significant error accumulation in the model, and the algorithm is robust and reliable.

349

350 **3.3 Control Algorithm**

351 The flow chart in Figure 4 shows the supervised demand-driven control algorithm proposed by
352 this research. The supervised control algorithm executes through three control phases.

353 (1) In phase I, the indoor positioning system will identify whether the zones or patches are
354 occupied. Once the system finds occupied patches, it will estimate the occupancy distribution for
355 each thermal zone and calculate the necessary air supply amount to be distributed to that zone.

356 (2) In phase II, the algorithm will determine which VAV boxes should be turned on or off and the
357 flow rate the VAV boxes need to provide to the zones. To quantify the thermal comfort of each
358 zone, the temperature difference $e_{z,i}$ (tracking error) between the average temperature of
359 temperature sensors in each zone (T_z) and the room temperature set (T_{SET}) will be used to
360 determine the airflow rate of each VAV box. If the $e_{z,i}$ fails to meet the condition ($e_{z,i} =$
361 $|T_z - T_{SET}| \leq 0.5^\circ\text{C}$), a temperature-based PID control and feedback mechanism will adjust the
362 airflow rate to reach the setting point. It should be noted that the supply airflow rate of each
363 operating VAV box is adjusted independently for each zone during this phase. Also, the supply
364 airflow rate should be adjusted to create a temperature distribution to match the occupancy
365 distribution. The flow rate of each VAV box can then be estimated or simulated with fluid dynamic
366 theories [50]. In this paper, the flow rates were calculated based on CFD simulation; to simplify
367 the calculation, the flow rate can also be determined by ASHRAE standards. Combined with the
368 mechanical information of VAV boxes, the cooling load and energy consumption can be estimated
369 through integrated efficiency over time. When the temperature tracking error in a corresponding
370 zone is minimized and the corresponding zone is conditioned to the setting point, meaning it meets
371 the thermal comfort level, the VAV box will work at that specific airflow rate. When the VAV box
372 of a target zone operates at full capacity for more than a specified time length (e.g., 5 minutes) and
373 the zone still cannot be cooled to reach the corresponding temperature setting point, the system
374 will proceed to the next execution phase.

375

376

Insert Figure 4 about here

377

378

379 (3) In phase III, the adjacent VAV boxes will work together with one of the target zones to
380 compensate the cooling load when one target zone in phase II cannot reach the expected thermal
381 comfort within 30 minutes. In the control algorithm, occupancy distribution mapped into occupied
382 patches can be acquired by IPS and only occupied areas in zones would be conditioned by VAV
383 boxes. When the target zone's air supply amount provided by a VAV box is not sufficient, the
384 adjacent zone will provide additional cooling power at its full capacity. Figure 5 illustrates how
385 zones are defined. The operation efficiency of a VAV box is determined by the occupancy level of
386 the zone it covers and its adjacent zones. Once the adjacent zone is defined, the target zone and
387 the adjacent zone would be combined as an integral zone. Phase II would be repeated so that the
388 flow rate provided by VAV boxes in two zones is adjusted based on the load in the integral zone
389 and the temperature captured by the sensors located at the thermal comfort level in the two zones.
390 Once occupancy information in the former zone has varied and cooling load estimation is lower
391 than the maximum cooling load, the corresponding zones would be conditioned individually, and
392 control phase III would transition back to phase II.

393

394

Insert Figure 5 about here

395

396

397 Compared to a conventional control system, the proposed algorithm treats the HVAC system in a
398 large space as a loosely connected system. Through switching between the zone levels to deduce
399 a room level control basis, the proposed algorithm provides a customized flow rate based on the
400 demand of each zone to avoid waste. The energy-saving potential is realized by matching the
401 demand to capacity and avoiding unnecessary cooling or heating activities in unoccupied zones.
402 Therefore, this proposed system is highly suitable for indoor spaces with uneven load distribution.

403

404 **3.4 Energy Simulation Model**

405 Since the preliminary experiment only has an IPS installed, to validate the efficiency of the
406 proposed control algorithm, we developed a building simulation model and conducted a
407 computational fluid dynamic (CFD) simulation for a sample office space. Simulation is an
408 economic approach to explore the feasibility of a control algorithm at low cost before the practical
409 system implementation. The outcome of the simulation could provide guidance for future system
410 design. More specifically, the outcome of our CFD simulation includes the temperature
411 distribution and the stable flowrate to maintain that temperature distribution. The flowrate
412 indicates how much energy the system will consume to supply the cooling air amount and maintain
413 the power of the VAV boxes. Therefore, the occupants' thermal comfort is achieved by matching
414 the temperature distribution and occupancy distribution.

415 In this study, Fluent Airpak was employed to simulate temperature distribution of the airflow
416 pattern under different control strategies. Fluent Airpak is one of the most popular commercial
417 software programs used in the HVAC field to simulate airflow, air quality, and contaminates. It can
418 construct realistic boundary conditions and predict the air spread and penetration in a confined
419 room. The physical test bed is an office room 10 meters in length, 10 meters in width, and 3 meters
420 in height. There are 21 occupants living in the room, and each occupant has one work desk, one
421 computer, and one monitor. To simplify the model, the mixing ventilation type is chosen. Hence,
422 the air in this room is a steady and incompressible Newtonian fluid with the buoyancy effect of
423 the body face neglected. The default ambient temperature outside the room is set as 35°C, which
424 is a typical summer temperature for most subtropical cities [17], while the indoor air temperature
425 is set to 25°C, also a typical indoor temperature setting point in most commercial buildings. The
426 specifications and physical conditions of the test room are illustrated in Figure 6.

427 -----
428 Insert Figure 6 about here
429 -----

430
431 The space also has wireless temperature sensors installed at the human breathing level, which is a
432 height of 1.1m and at least 0.5m away from the nearest occupant [17]. The walls of the room are

433 adiabatic, and the heat flux of the walls is neglected. The turbulence in the room is modeled with
434 two standard $k - \varepsilon$ equations to represent the airflow of the mixing ventilation. More detailed
435 room specifications and CFD model settings are summarized in Table 2. The total cooling load is
436 about 5.3kW when the room temperature is set at 25°C.

437 -----
438 Insert Table 2 about here
439 -----

440
441 The transfer function method (TFM) is used to estimate cooling load and identify uneven cooling
442 demand. The TFM can relate an output function at one specific time to the value of one or more
443 input functions at that time or to previous values of output functions [51]. Based on TFM, heat
444 flux to the zone can be expressed using the formula below:

$$Q_i = Q_I + Q_V + Q_G + Q_{ADJ} + Q_S \quad (8)$$

445
446 where Q_{ADJ} , Q_S in equation (8) represents the heat gain due to air flow from the adjacent zone or
447 boundary condition and the heat gain to the zone from all surfaces, respectively. The model
448 constructed in this study represents an inner space not in direct contact with the building envelope
449 or surface, and in this case, the heat transfer between two adjacent zones is neglected.

450 While the heat gains to a zone due to infiltration Q_I and ventilation Q_V are

$$Q_I = m_I \cdot C_{Air} \cdot (T_{OUT} - T_{IN}) \quad (9)$$

451
452 and

$$Q_V = m_V \cdot C_{Air} \cdot (T_V - T_{IN}) \quad (10)$$

453
454
455 ASHRAE standards require a minimum ventilation rate of fresh air for occupants in rooms or

456 whole buildings [52]. That is, heat gains to a zone due to ventilation can be estimated with
 457 occupancy information. The heat production, Q_G , usually includes the process of equipment heat
 458 dissipation (such as computers in our model, shown in Figure 6), lamp heat dissipation, and human
 459 body heat dissipation. Therefore, the energy consumption of occupants can be calculated as

$$Q_G = W_z = \sum_{N_O} W_O + 1000 \cdot n_1 \cdot n_2 \sum_{N_C} W_C + \sum_{N_L} W_L \quad (11)$$

460

461 The values of W_C , W_L could be different based on the device type and size. W_O can also be
 462 different according to occupant type (men, women, or children), activity, and garments worn. To
 463 maintain a consistent room temperature setting (T_{SET}), VAV boxes in the HVAC system need to
 464 provide conditioned air with supply temperature T_{IN} to compensate for the total heat flux
 465 illustrated in all heat gain equations. The total heat flux, then, can be calculated as

$$Q = \sum Q_z = C_{Air} \cdot \sum_k M_k \cdot (T_{IN} - T_S). \quad (12)$$

466

467 M_k is the volume of air provided by VAV box k at unit time, which is the control object in this
 468 study. The volume of supply air and the ventilation of VAV boxes can be used to calculate the
 469 energy consumption of the ventilation system in the test space.

470

471 **3.5 Infrastructure Coupling for the Simulated Space**

472 Current building service systems in most commercial buildings assume the building occupants
 473 have fixed occupancy schedules. The facility managers operate the building facilities based on
 474 fixed occupancy schedules or maximum occupancy. Partially occupied and unoccupied conditions
 475 are not considered during the day's standard operating periods (e.g., 9am to 6pm) [10]. The chart
 476 in Figure 7 illustrates the integrated detection-control system and the interrelationships between
 477 the indoor positioning system and the HVAC control system. Through a positioning algorithm and
 478 received signal strength of all Wi-Fi access points (APs) and BLE broadcasters (iBeacons), the
 479 indoor positioning system can locate all occupants inside the room. The identity of occupants can
 480 be acquired through each device's unique MAC address. Location coordination and timestamps

481 can calibrate and synchronize the two systems. Then, the number of occupants and their
482 distribution inside the room is calculated through occupant location coordination. The flowrate of
483 each VAV box is determined by the occupancy status (fully occupied, partially occupied, or
484 unoccupied) of patches in each thermal zone. Under the thermal comfort requirement, we could
485 adjust the status of VAV boxes based on feedback control mechanisms, and the mapping of
486 occupancy and thermal comfort requirements in this integration can provide further insight into
487 the relationship between occupancy distribution and air supply amount at one acceptable thermal
488 comfort requirement. The fluctuations caused by occupant pass-by and short stays will be
489 eliminated to avoid too-frequent adjustments [10].

490 -----
491 Insert Figure 7 about here
492 -----

493
494 Figure 7 shows the fundamental infrastructures of the VAV system and the indoor positioning
495 system. Wireless temperature sensors are also installed at the breathing level to measure
496 temperature in the space. The APs and iBeacons generate two radio frequency networks covering
497 the whole experimental space. There are four thermal zones inside the room, and each zone is
498 independently served by a VAV box. The proposed HVAC control mechanism is also able to
499 differentiate the occupied, partially occupied, or fully occupied spatial conditions by coupling with
500 an indoor positioning system. The experiment space is divided into four thermal zones (Zone A,
501 Zone B, Zone C, and Zone D) based on the locations of four VAV boxes. Each zone has wireless
502 temperature sensors installed, and each zone is further divided into smaller patches for higher error
503 tolerance in positioning. The distribution of occupancy in each zone can be reflected in the
504 occupied patches. For example, in Zone A of Figure 8, there are five work stations, but only three
505 are occupied. The temperature of each zone is collected from the wireless temperature sensors to
506 represent the thermal comfort level of occupants. The occupancy pattern and residents' routes in
507 the space are randomly simulated based on the signal samples collected from the preliminary field
508 experiment. All occupants perform as the regular building energy end-users by requiring thermal
509 comfort and controlling the occupancy-schedule related appliances. With occupancy distribution,
510 the building energy load at zone level or building level can be formatted in detailed in occupant-

511 related loads, such as human thermal gains as well as appliances' energy usages in real-time.

512 -----

513 Insert Figure 8 about here

514 -----

515

516 **4. RESULTS AND ANALYSIS**

517 **4.1 Types of Occupancy Distribution**

518 To validate the efficacy of the proposed control algorithm, 12 scenarios with four zone-level
519 occupancy types (fully occupied, half-occupied, partly-occupied, and unoccupied) were tested in
520 the preliminary experiment. These occupancy types are defined as follows: (1) fully-occupied
521 zones are zones where residents occupy all patches, (2) half-occupied zones have half or more than
522 half of patches occupied, (3) partly-occupied zones have fewer than half of all patches occupied,
523 and (4) unoccupied zones are zones without any occupants. Two types of room-level occupancy
524 distributions are also defined in the preliminary test: (1) even distribution, in which all four zones
525 are occupied at some level (or no zones in the room are unoccupied), and (2) uneven distribution,
526 in which at least one unoccupied zone is observed. Figure 9 illustrates different occupancy
527 distributions.

528 -----

529 Insert Figure 9 about here

530 -----

531

532 To investigate the performance of the HVAC system control algorithm developed by this research,
533 CFD Airpak was applied to simulate airflow patterns to assure the required comfort level is
534 achieved. In the CFD simulation, temperature is selected as the thermal comfort level indicator. In
535 the preliminary test, we assumed a thermally comfortable space should have a temperature around
536 25°C. As a demand-driven control system, the proposed algorithm only guarantees the occupied
537 patches are thermally comfortable. If the temperature of the zone is higher than the threshold, the
538 VAV boxes will switch on and adjust the temperature. Once the target temperature is reached, the

539 VAV boxes will turn off. The supply air amount for each zone varies with different numbers of
540 occupants in that zone. Table 3 lists the 12 scenarios with different occupancy patterns and supply
541 air amounts, determining the temperature variation. Each zone is conditioned independently with
542 its VAV box at full capacity while no cooling air is provided to an unoccupied zone.

543 -----
544 Insert Table 3 about here
545 -----

546

547 **4.2 Comparison with Conventional Methods**

548 The most widely selected indicators to estimate the number of occupants are CO₂ concentration
549 and room temperature distribution. In practice, facility management systems use the temperature
550 at return air vents as the control variable of thermal comfort in most commercial buildings. Yet, in
551 this paper, we utilize IPS as a new way to gather more precise occupancy information to estimate
552 actual demands. Therefore, to form a comparison, two conventional control methods, the return
553 air temperature (RAT) control system and the breathing level temperature (BLT) control system,
554 have also been examined in the CFD simulation. In the RAT control system, the temperature sensor
555 is installed inside the return air vent to collect return air temperatures. The VAV boxes operate
556 based on the temperature difference between the setting point and the return air. However, the
557 temperature around a return duct cannot efficiently represent the thermal comfort need of
558 occupants due to thermal stratification and uneven distribution. The BLT control system uses
559 temperatures collected by sensors at the human breathing level to determine when to turn the VAV
560 boxes on or off to maintain a thermal comfort temperature. However, temperature fluctuations at
561 breathing level are not necessarily caused by occupants, so that cannot accurately reflect the
562 occupancy. In the proposed IDC system, VAV boxes are operated based on the zone-level
563 occupancy of the space meshes/patches with the help of IPS tags and feedback adjustments based
564 on temperature sensor nodes at breathing level. To keep the temperature (T_z represents breathing
565 level temperature) at the proper level, cooling air is supplied to the zone in proximity to occupied
566 zones, intentionally avoiding VAV-box operation in unoccupied zones.

567 Figure 10 shows the comparison between temperature distributions around occupancy level under
568 all three control mechanisms (RAT, BLT, and IDC) in three typical scenarios. In those cases, we

569 assumed the occupancy distributions as illustrated in the figure and that equipment is turned on to
570 distinguish the differences among the three methods.

571 -----

572 Insert Figure 10 about here

573 -----

574

575 Temperature contours under the three control systems depicted in Figure 10 show different
576 temperature distributions around occupied patches. In the RAT, the VAV boxes condition all zones
577 with temperature feedback from return ducts without detecting the room occupancy condition.
578 Therefore, all VAV boxes must be adjusted simultaneously with the same setting. In the BLT, the
579 heat radiation of the occupants and computers skews the temperature contour. VAV boxes are
580 adjusted accordingly using the feedback from the breathing-level temperature sensors where
581 occupants are located. Such improvement avoids unnecessary over-cooling and assures all zones
582 are sufficiently conditioned. However, unoccupied zones with a higher temperature are also
583 supplied with cooling air, as in Zone B. Therefore, cooling air supplied by a BLT system in such
584 areas results in energy waste. In the IDC, only occupied zones are conditioned, so the temperatures
585 in unoccupied zones would not be considered. The temperatures of unoccupied zones are normally
586 higher than the typical thermal comfort level. As shown in Figure 10, only occupied zones are
587 conditioned. Comparing the RAT, BLE, and IDC systems, the IDC system conserves the most
588 energy by avoiding interference from unoccupied zones. The temperature-based thermal comfort
589 level is sufficiently satisfied in occupied zones for all three systems, while the IDC system leaves
590 unoccupied zones at a higher temperature. Therefore, the rationale behind the proposed IDC
591 system is mainly based on the temperature distribution of the occupied zones and leaves the
592 unoccupied zones. Therefore, energy can be saved by intentionally avoiding cooling unoccupied
593 zones and leaving their temperatures high.

594 -----

595 Insert Figure 11 about here

596 -----

597

598 Figure 11 compares the average temperatures of occupied and unoccupied zones for the above
599 three scenarios. Using Scenario 4 as an example, the proposed IDC system only needs to maintain
600 the temperature of occupied Zone C and Zone D at 25.5°C and intentionally leave Zone A and
601 Zone B uncooled to be more efficient with the air supply. Since the RAT does not need information
602 on occupancy distribution and merely adjusts VAV boxes based on the temperature at return air
603 ducts, the VAV boxes must be uniformly adjusted for the whole room. For the BLT system, where
604 occupancy detection is also based on temperature, a fluctuation in temperature caused by
605 equipment can be easily misinterpreted as occupancy. With the help of the CFD simulation, we
606 can examine the necessary supply air amount for all three systems. Table 4 compares the amount
607 of conditioned air supplied in the above three different scenarios. Under conventional control
608 systems, the required supply airflow rate is 1350 m³/h for the RAT system and 1087 m³/h for the
609 BLT system. Compared to the RAT system, the IDC system will save 30.4% of air supply,
610 dramatically reducing the energy consumed to power the electrical fans and cooling system. It can
611 also be observed that more detection of unoccupied zones by the IDC means more energy can be
612 saved using the proposed IDC systems.

613 -----
614 Insert Table 4 about here
615 -----

616
617 Figure 12 shows the required air supply amount of all 11 scenarios (S12 is the unoccupied scenario).
618 The IDC system is the most energy-efficient mechanism that needs the least air supply to maintain
619 the occupants' thermal comfort. It is also important to note that in the condition of even distribution
620 of room-level occupancy, the BLT and IDC systems require the same amount of cooling air.

621 -----
622 Insert Figure 12 about here
623 -----

624

625 **4.3 A Case Study on the Daily Energy Saving of the IDC System**

626 This case study adopts a sample occupancy schedule to examine the energy-saving potential of the
627 proposed control system. An occupancy of 21 total occupants in the preliminary experiment is
628 scaled to an ASHRAE-recommended occupancy schedule. According to ASHRAE standard 90.1
629 [31], the hourly schedule has been mapped into the three occupancy scenarios we previously
630 discussed: (1) from 09:00 to 10:00 and 18:00 to 19:00 in Scenario 6, (2) from 10:00 to 12:00 and
631 from 14:00 to 18:00 in Scenario 1, and (3) from 12:00 to 14:00 in Scenario 4. We assume the
632 HVAC system for the whole building is turned off from 20:00 to 08:00. Therefore, in this study,
633 we divided the occupancy schedule into three scenarios. Based on the results in the previous
634 section, occupancy distributions of those scenarios were acquired and air supply amounts were
635 determined at a certain comfort level. To illustrate energy consumption, Figure 13 shows the
636 proposed occupancy schedules with specifications for the four thermal zones. We also assumed
637 computers are still running when occupants leave the room during working hours or lunchtime,
638 and all three control systems update the VAV box settings every hour.

639 -----
640 Insert Figure 13 about here
641 -----

642
643 Figure 14 shows the cooling load estimated for all three systems and IDC’s energy-saving potential
644 during a typical work day. The cooling load was calculated with the supply air amount in Table 3
645 and the duration of occupancy. Then, the energy consumption of the room was estimated based on
646 the calculated cooling load. As shown in Figure 14, a significant load reduction can be realized
647 when IDC control systems are substituted for the traditional RAT control systems. As observed in
648 Figure 14, an IDC system has a higher chance of saving more energy when more load variation
649 exists. For example, when occupants leave the office during lunchtime, some of the VAV boxes
650 can be turned off. Since we assume all occupants are present during the day and only leave for a
651 short period at lunch, there is a relatively small energy-saving potential with the IDC system
652 compared to the BLT system. Similarly, in some seldom-used rooms designed for special purposes
653 —such as the conference room, kitchen, or restrooms—the IDC system has a higher energy-saving
654 potential. Compared to the RAT system, the energy-saving potential of the IDC system during the

655 entire day can reach around 8.99 kWh, which is 22.77% of the total daily energy consumption.
656 Compared to the BLT system, the IDC system can achieve a 0.74kWh (2.36%) reduction in energy
657 use during a two-hour lunchtime. The difference between the IDC and BLT is not large since the
658 room was almost fully occupied by the residents throughout the whole day. It also can be observed
659 from Figure 14, the majority of energy saving comes from the discrepancy between the IDC and
660 BLT during lunch time. Given the IDC and RAT evenly provide conditioned air to all zones due
661 to the lack of occupancy distribution information, the BLT has the advantage in providing
662 unbalanced service to different zones. Therefore, the more unevenly distributed occupants are
663 within the large space, the more energy can be saved with the BLT.

664 -----
665 Insert Figure 14 about here
666 -----

667

668 **5. DISCUSSION**

669 **5.1 Implication of the Findings**

670 In this study, a cooling load estimation model is employed to assess the primary cooling air amount.
671 Such a temperature-based estimation system is the most widely used control method in most
672 buildings. However, the two major drawbacks of temperature-based systems are (1) all conditioned
673 air supplies must be uniformly controlled, and (2) there is no feasible solution for unoccupied zone
674 detection. Demand-driven control can resolve both limitations by adjusting the supply airflow rate
675 based on accurate occupancy [19] and leave high temperatures in unoccupied areas un-ameliorated
676 [53]. Therefore, the proposed IDC control system provides significant energy-saving potential by
677 avoiding unnecessary zone-level energy consumption. As shown in Figure 6, through intentionally
678 avoiding interfering with the temperature in unoccupied zones and leaving them as hot spots, the
679 total conditioned air supply can be reduced. This principle can also be extended to other building
680 service systems, such as lighting and ventilation systems. Building upon occupant distribution
681 detection, engineers will be able to design and construct more sustainable and smarter systems that
682 can be automatically controlled.

683 The IDC system also provides a low-cost pervasive sensing network solution for building
684 monitoring and indoor occupancy acquisition. If the complete installation of an indoor positioning

685 system is not feasible for some buildings, a portable system could be used to detect the occupancy
686 pattern from room to room. Since many existing buildings are manually and periodically controlled,
687 using occupancy patterns would be helpful to guide the system settings in different thermal zones
688 [54]. Long-term occupancy pattern recognition can assess the utilization of space and identify
689 mostly unoccupied zones. Such information can not only help optimize the HVAC operation
690 schedule, but also enable unoccupied space reassignment [16]. It is also feasible to provide a better
691 indoor environment and thermal comfort for frequently used spaces and preferred spaces [55]. In
692 the IDC system, the integrated facility operating system and indoor positioning system record the
693 occupancy, timestamp, and system operation information of each space. Historical data logs of
694 these systems provide a rich source for further data mining and processing, especially for profiling
695 energy-consumption patterns through a machine-learning process by associating energy needs with
696 spatial locations [56]. A large number of local energy management strategies require such types of
697 information, including demand response (DR) and demand-side management (DSM) [57] [58].

698

699 **5.2 Limitations and Future Work**

700 In this research, we would like to couple the Wi-Fi and BLE networks to obtain occupancy
701 distribution for demand-driven control with the purpose of improving building energy efficiency.
702 Although the proposed system advances current energy management in buildings, this study also
703 yields several limitations we intend to resolve in future research. First, the room-level cooling load
704 in this research is estimated by linear addition of all subzones. In practical operation, the supplied
705 flows from multiple air vents will interfere with each other through a nonlinear heat transfer
706 process. Therefore, future research should adopt more sophisticated fluid-dynamic models that
707 consider such superimposed effects. Second, obstacles in the positioning networks could result in
708 inaccuracy in the location detection. The current experiment test bed is a continuous space without
709 large separators, such as structural columns or walls. Also, the space boundaries in direct contact
710 with the building envelope and surface could significantly change the simulation results since the
711 outside environment is more dynamic and complicated. These building surfaces and components
712 could also potentially disturb the stability and accessibility of received signals [59]. Therefore,
713 more complicated indoor geometry needs to be examined to identify potential problems and
714 develop future improvements to positioning accuracy. Third, the indoor positioning system would
715 collect the MAC address or UUID information of occupants' devices, but for the occupants'

716 privacy, this tagging might be impermissible and should be improved and protected. This issue
717 could not be avoided during the research on location tasks. To protect privacy, future work must
718 enable new tagging techniques. Fourth, this study ignored the positions of those occupants who
719 occupied one room just for one short-term duration and could be defined as temporary occupants.
720 In this positioning work, we did not figure out a method to filter the temporary occupants from the
721 permanent occupants. In the next study, we would like to focus on this issue and illustrate a
722 schedule of permanent occupants to provide a more detailed control basis for HVAC systems. Fifth,
723 the frequency of adjustment must be determined for practical application. The positioning system
724 can stream real-time occupancy data to the building management system, but the response of the
725 facility control system is subject to lag and instability. A too-frequent adjustment could result in
726 error accumulation and system instability. Therefore, for automatic centrally-controlled systems,
727 it is essential to investigate the best system adjustment frequency to find a compromise between
728 system uncertainty and energy-saving potential. Sixth, another significant issue is related to
729 privacy: no matter which IPS is used, it always requires the occupants to carry sensing tags (in our
730 system these tags were personal cellphones). More studies are necessary to develop safer
731 technologies to provide occupant privacy. The last and most important limitation of our research
732 is the control system actuation is based on simulation. More validity must be provided through
733 field experimentation to verify how much energy can be saved by the proposed system. Current
734 energy consumption is derived from CFD temperature simulation. It is strongly suggested that
735 future studies create a complete control loop with physical actuators and carefully monitor their
736 energy consumption.

737 **5.3 Contributions**

738 Efficient HVAC system control depends on reliable occupancy information. The direct observation
739 of indoor positioning systems enables accurate load estimation and flexible control. In this study,
740 the proposed control mechanism relies on high-resolution occupancy information collected from
741 a dual Wi-Fi and BLE network. Both networks are convenient to install and apply, especially Wi-
742 Fi access points which are normally pre-installed in most buildings. In current popular applications,
743 BLE technology is convenient for broadcasting and integrating signals with location information.
744 To eliminate the weakness of Wi-Fi technology in providing more accurate location tags, this study
745 coupled BLE technology into the dual networks. Integrating the indoor positioning system with an
746 energy management system could allow for more efficient energy use in buildings. In conventional

747 control systems, occupancy information is approximated from historical or indirect data. Such
748 approximation not only lacks accuracy, but also sacrifices resolution [60]. The major improvement
749 in the proposed system is controlling the building facility by relying on zone-level occupant
750 distribution. Using current control mechanisms, the setting point of a building's facilities, such as
751 an HVAC system or lighting system, is determined by the rough estimation of room-level occupant
752 counts. With the help of the IPS, the proposed control system can not only detect the number of
753 occupants, but locate their accurate spatial distribution in the form of space meshes. With meshed
754 occupancy patches enabled by the dual layer positioning network, the VAV boxes of HVAC
755 systems can be adjusted accordingly with higher positioning accuracy tolerance and compatibility
756 with system design. In addition, the zone-level and room-level occupancy patterns can be
757 recognized and used to guide facility operation. For example, the occupancy pattern has been
758 categorized into unoccupied, partly-occupied, half-occupied, and fully-occupied to estimate the
759 necessary air supply amounts and adjust the system settings. Such differentiation avoids over-
760 cooling caused by uniform control of multiple zones with different occupancy levels. In summary,
761 the major contribution of our proposed system is it develops a specifically designed algorithm,
762 which combines both kNN and random walk model, for the dual layer positing system, and it
763 integrates the demand-driven HVAC control principle with meshed spatial occupancy distribution
764 detected by the proposed IPS. Given that the HVAC system is designed to control thermal zones,
765 the spatial occupancy distribution is ready for direct implementation. At the same time, given that
766 the IPS system we proposed is not intended to capture exact location coordinates, it has much
767 higher tolerance in positioning inaccuracy. Moreover, meshes/patches work like a 2D lattice, which
768 is much simpler compared to a continuous space. This allows simple control algorithm design as
769 well as higher processing and control speed.

770

771 **6. CONCLUSION**

772 Among the HVAC control modes, occupancy behaviors are the key to assessing whether the
773 occupants' thermal comfort has been sufficiently satisfied. The proposed demand-driven control
774 system implements an indoor positioning system to collect occupancy information with higher
775 resolution and accuracy. Wi-Fi and BLE technologies are utilized in the indoor positioning
776 equipment, and in this research, occupancy distribution data acquired by dual networks is
777 generated as a demand-driven signal. Based on such information, the service space can be

778 monitored and controlled by actual demand rather than rough estimations of temperature and the
779 number of occupants. In the zone temperature distribution simulation, different control methods
780 were compared and the results showed higher-accuracy occupancy acquisition can better conserve
781 cooling air amounts. Integrated with the accuracy of IPS, daily energy performance analysis was
782 conducted comparing IDC, BLT, and RAT systems. The results of this study show the proposed
783 system has significant energy-saving potential for demand-driven HVAC operation in large-scale
784 rooms by avoiding over-cooling and uniform cooling.

785 Although the core of this research focuses on determining occupancy distribution and then
786 controlling air supply modes in VAV systems, such demand-driven control systems can also be
787 extended to other building service systems and may enable more sophisticated control design,
788 making the potential applications far more extensive. In the coupled IPS and HVAC system, we
789 postulate the functional harmony in the integration of two systems and the ability to provide more
790 useful information for future improvements. As the technologies and algorithms in IPS evolve and
791 mature, the accuracy of IPS can be improved and provide greater benefits to occupancy
792 information schemes.

793

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1022 Table 1 Accuracy of Each Measurement Metric and Its Chronic Development

Timestamps	Route 1				Route 2			
	EU	CB	HM	M3	EU	CB	HM	M3
1-200	0.7605	0.7939	0.5842	0.7674	0.8062	0.8276	0.5240	0.8416
201-400	0.6707	0.7197	0.5552	0.6800	0.7017	0.7952	0.5217	0.7598
401-600	0.6400	0.7145	0.5481	0.7258	0.6922	0.7750	0.5123	0.7602
601-800	0.6378	0.7140	0.5512	0.7072	0.7022	0.7813	0.5109	0.7707
801-1000	0.6707	0.7197	0.5552	0.6800	0.7017	0.7952	0.5217	0.7598
Over all	0.6759	0.7323	0.5587	0.7120	0.7208	0.7948	0.5181	0.7784

1023 *Note: EU – Euclidean; CB – City Block; HM – Hamming; M3 – Minkowski 3;
 1024 The value in each cell is the positioning accuracy.

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1027 Table 2 Room Specifications and Model Settings

Ventilation type	Mixing ventilation
Geometry	10m x 10m x 3m
Diffuser type	Square ceiling diffuser
The dimension and quantity of fresh air diffusers	0.4m x 0.4m, 4
The dimension and quantity of exhaust air vents	0.4m x 0.4m, 4
Heat sources	Occupant 21x75W Computer 21x150W Lamp 16x35W
Total cooling load	5.285kW
Air supply temperature	18°C
Room temperature set	25 ±0.5°C
Turbulence model	Standard equations

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1030 Table 3 Supply Air Amount of Different Occupancy Distributions in a Sample Space

Scenario Index	Occupancy	M_A	M_B	M_C (m3/h)	M_D	M	Zone Occupancy	Room Occupancy
1	(5,4,6,6)	353	265	382	382	1382	(FO,FO,FO,FO)	Even Distribution
2	(5,0,6,6)	353	0	382	382	1117	(FO,UO,FO,FO)	Uneven Distribution
3	(5,4,6,0)	353	265	382	0	1000	(FO,HO,FO,UO)	Uneven Distribution
4	(0,0,6,6)	0	0	382	382	764	(UO,UO,FO,FO)	Uneven Distribution
5	(5,4,0,0)	294	206	0	0	500	(FO,FO,UO,UO)	Uneven Distribution
6	(0,0,6,0)	0	0	265	0	265	(UO,UO,FO,UO)	Uneven Distribution
7	(5,2,6,2)	353	118	382	118	971	(FO,PO,FO,PO)	Even Distribution
8	(2,3,2,5)	118	176	118	353	765	(PO,HO,PO,HO)	Even Distribution
9	(1,1,3,5)	59	59	206	353	667	(PO,PO,HO,HO)	Even Distribution
10	(4,3,3,5)	280	206	206	280	972	(HO,HO,HO,HO)	Even Distribution
11	(3,0,4,3)	206	0	265	206	667	(HO,UO,HO,HO)	Uneven Distribution
12	(0,0,0,0)	0	0	0	0	0	(UO,UO,UO,UO)	Even Distribution

1031
 1032 *Note: The column of “Occupancy” shows the number of occupants at Zone A, B, C, and D;
 1033 M_s' show the amount of conditioned air that needs to be supplied to that zone to meet comfort level:
 1034 FO – fully-occupied, HO – half-occupied, PO – partly-occupied, UO – unoccupied.

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1037 Table 4 Comparison of Supply Air Amount among Three Control Systems

Scenario Index	Occupancy	Room Occupancy	Supply Air Amount (m3/h)			Percentage Reduction	
			RAT	BLT	IDC	RAT-IDC	BLT-IDC
4	(0,0,6,6) (UO,UO,FO,FO)	Uneven Distribution	1350	1087	940	30.4%	13.5%
1	(5,4,6,6) (FO,FO,FO,FO)	Even Distribution	1704	1381	1381	19%	0%
11	(3,0,4,3) (HO,UO,HO,HO)	Uneven Distribution	1250	1116	1014	18.9%	9.1%
6	(0,0,6,0) (UO,UO,FO,UO)	Uneven Distribution	1071	881	650	39.3%	26.3%

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1039 *Note: The column of “Occupancy” shows the number of occupants at Zone A, B, C, and D;

1040 FO – fully-occupied, HO – half-occupied, PO – partly-occupied, UO – unoccupied;

1041 The column of “Percentage Reduction” compares the amount of supplied air between the proposed control
 1042 system and the conventional systems.

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