

Article

Sustainable Smoke Extraction System for Atrium: A Numerical Study

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Abstract: Smoke extraction systems, either static with natural ventilation, or dynamic with mechanical ventilation are required to keep smoke layer at high levels in many tall atria. It is observed that a design fire with high heat release rate (HRR) is commonly used for designing natural vents, but a low HRR is used for mechanical ventilation system. This will not produce a sustainable environment. There are no internationally agreed on design guides to determine the HRR in the design fire for different extraction systems and scenarios. This issue will be studied using a Computational Fluid Dynamics (CFD)-based software, the Fire Dynamics Simulator (FDS) version 6.7.1. Simulations on natural smoke filling, static and dynamic smoke extractions were carried out in a big example atrium. CFD-FDS predictions were compared with previous full-scale burning tests. Results confirmed that static smoke extraction is a good option for big fires, and a dynamic system is best for small fires. A sustainable new hybrid design combining the advantages of static and dynamic systems is proposed, which could result in a lower smoke temperature and higher smoke layer interface height, indicating a better extraction design.

Keywords: numerical simulation; smoke extraction system; atrium; natural vent; mechanical ventilation



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1. Introduction

There are many construction projects with big atria while developing sustainable urban areas in the Asia-Oceania regions since 1980s [1–7]. The Greater Bay Area, which includes nine cities in Guangdong, Hong Kong and Macao, is a very good example [8]. Many of buildings with large atria [1–7,9] have been constructed or planned [8]. It is required that fire safety provisions have to be installed for crowded atria. The smoke layer in a fire has to be kept high to allow people to see through so that smoke will not give adverse effects to people staying inside, occupants trapped inside can locate the exits, firemen can identify the fire sources, and high thermal radiation will not act on the firemen fighting against the fire [10–14]. A high smoke layer will also provide an environment free of toxic smoke for the occupants and reduce thermal radiative heat flux from the hot smoke. There are no design guides with applicable correlation expressions. Performance-based design (PBD) or Fire Engineering Approach (FEA) was thus applied to design smoke extraction systems in big atria to achieve a high smoke layer [15–19].

It is not feasible to study the fire aspects and performance of system design with full-size physical models for most projects. In a fire, there are many conditions which are difficult to be ‘repeated’ or ‘reproduced’ [20]. Prohibitive human and financial resources are required to do the tests. Only a few selected real fire tests are required in some areas to confirm that the smoke layer can be kept at a high level for a small fire. Computational Fluid Dynamics (CFD) fire models have therefore been used for designing the atrium smoke exhaust systems in PBD/FEA projects in recent decades [21–27]. However, there are

numerous problems in using CFD [28,29]. For example, two-dimensional simulations were conducted with only 100,000 cells in the 1990s due to the limited computer capability at that time, and only averaged flow parameters can be predicted using Reynolds-Averaged Navier–Stokes (RANS) equations. The CFD software Fire Dynamics Simulator (FDS) version 6.7.1 [30], developed by the National Institute of Standards and Technology in the USA, became a common design tool 15 years ago. The newer version of FDS is based on the Large Eddy Simulation (LES) and is able to handle up to 5 million computing domains and can be readily executed now even on a personal computer. The simulation results can be applied to predict the parameters of fire hazards, such as estimating the Available Safe Egress Time (ASET) [18,31,32] based on CFD predictions.

Most of the CFD fire models are not properly validated due to high cost and other difficulties in conducting large-scale fire tests. PBD-FEA projects using the CFD models are always challenged by different parties [19,22]. Consequently, hot smoke tests are required to demonstrate the system performance in the atrium with a small fire up to two MW. In Hong Kong, hot smoke tests are required to evaluate the performance of smoke extraction systems in halls of irregular shape or halls taller than 12 m [13,33].

There are two types of smoke extraction systems in large halls: natural venting or static smoke extraction systems, and mechanical ventilation systems or dynamic smoke extraction [11,12,33,34]. HRR is important in designing a smoke extraction system [35]. It is observed that large design fires up to 20 MW are commonly used for designing natural vents, while a small design fire (e.g., five MW) is commonly used for mechanical ventilation [13,19,36]. It is obvious that fire size should not be a function of the venting method that depends on how the combustibles are burnt. Fire size should be selected based on hazard assessment. The performance of the two smoke extraction systems under fires of different HRR will be further explored in this paper using CFD simulation. As an important purpose of the paper, a new proposed sustainable hybrid design combining static and dynamic extraction system will be explored and compared with the traditional ones, which could result in a lower smoke temperature and higher smoke layer interface height, indicating a better extraction design.

For example, a big atrium with 150 m length, 40 m width and 30 m height as shown in Figure 1 was used to study the performance of the two traditional extraction systems under big, medium and small fires as well as the proposed hybrid extraction system in the same conditions. The CFD-FDS predictions on smoke extraction were validated using full-scale burning tests data in two example buildings with atria [22,29,37,38]. This included smoke filling and extraction data in the PolyU/USTC Atrium in Mainland China [39,40] and hot smoke test data in an irregular atrium in Hong Kong [41,42].

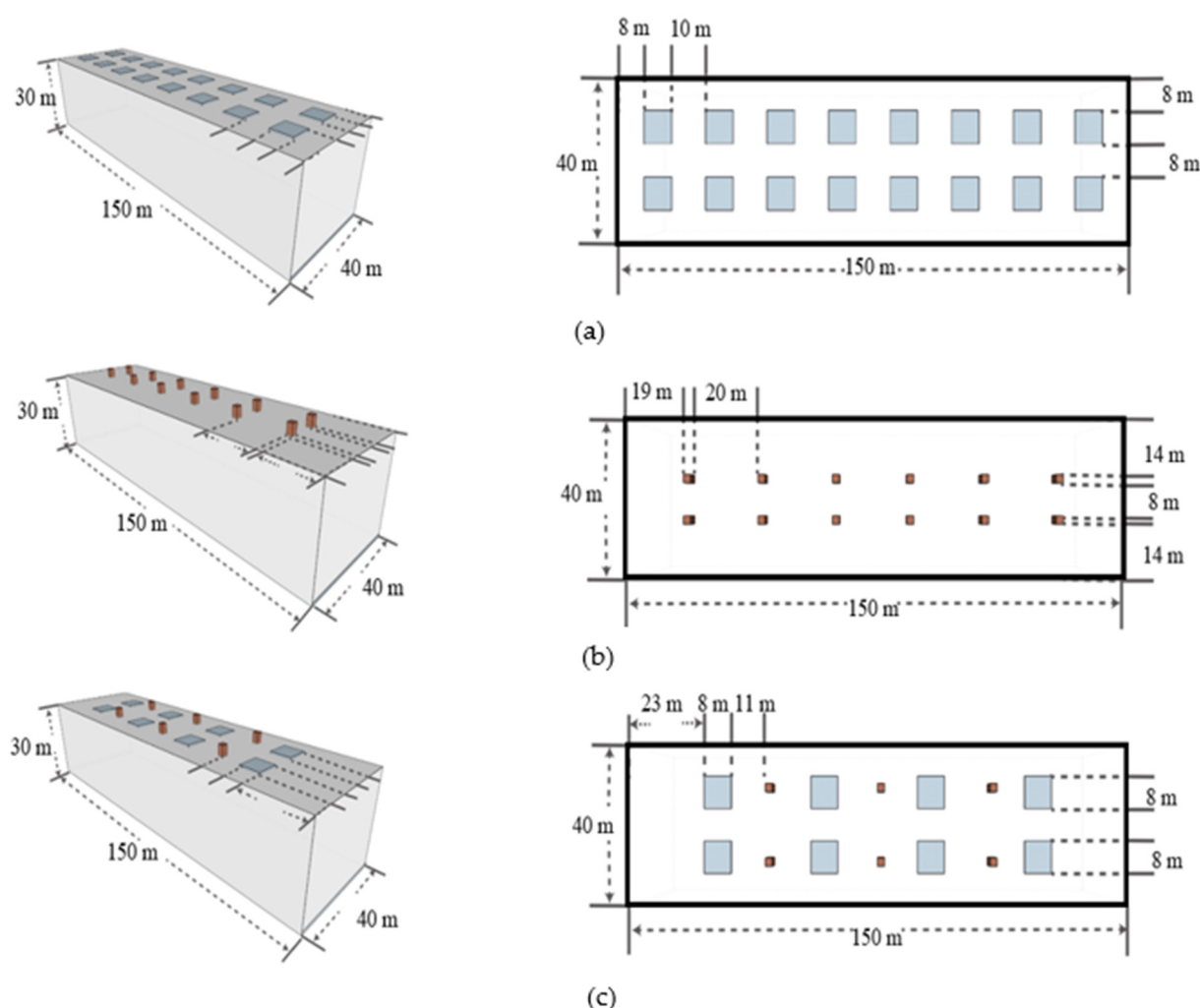


Figure 1. Smoke extraction designs in the example atrium. (a) Natural vents; (b) Mechanical ventilation; (c) Proposed hybrid system.

2. Atrium Smoke Movement

The smoke filling the atrium, the natural ventilation in case of fire, the mechanical ventilation in case of fire, the sprinklers in atria and their effectiveness, the spill plume dynamics and others were studied by physical experiments and numerical simulations worldwide.

As a beginning of the foreign literature review on the topic, the first full-scale testing of smoke behavior in large scale spaces was reported in Japan in 1985. This was a full-scale experiment in a seven-story atrium with dimensions of 30 m × 24 m × 26.4 m with methanol as a fuel and with an HRR of 1.3 MW. In this research, Yamata et al. studied the smoke filling process and ventilation methods [43]. Earlier in the same year, they examined the same occurrences theoretically [44]. The comparison demonstrated a good match with the real experiment. That same year in Sweden, Hagglund et al. made a full-scale test in the Cubic Hall with the dimensions of 5.62 m × 5.62 m × 6.15 m and with sources with an HRR from 70 to 630 kW [45]. In the 1990s, smoke filling experiments became popular in Australia [46] as well as big full-scale experiment in a stadium, reported by Dillon [47].

Later in 1997, fire experiments in the field of the mechanical ventilation of smoke in the case of a fire were carried out in Canada by Loughheed and Hadjisophocleous in a compartment with the dimensions of 9 m × 6 m × 5.5 m with a square propane sand burner an an HRR from 15 kW to 1 MW [48]. Later experiments by Loughheed et al. were carried out in a facility with dimensions of 13.11 m × 17.22 m × 12.2 m, which is approximately

two times bigger than the previous one with possibilities of fires with HRRs from 250 kW to 5 MW [49].

The beginning of the current millennium can be regarded as a cornerstone in the burning experiments in atria. There are currently two bigger globally known facilities where full-scale burning experiments in atria can be conducted.

The first model atrium is located in at the campus of the University of Science and Technology of China at Hefei. It is a big concrete construction jointly built by The Hong Kong Polytechnic University and the University of Science and Technology of China with internal dimensions of $22.4\text{ m} \times 11.9\text{ m} \times 27\text{ m}$. In 2000, experimental studies on natural smoke filling in this atrium with liquid pool fires up to 1.6 MW were conducted [39].

The second model atrium is in the Technological Metal Centre in Murcia, Spain. Researchers from the University of Jaén, The University of Comillas, with the collaboration of the Imperial College of London, MAPFRE and the fire consultancy firm JVVA Fire and Risk have undertaken several full-scale fire tests in the facility. The dimensions of the Murcia Fire Atrium are $19.5\text{ m} \times 19.5\text{ m} \times 17.5\text{ m}$ and a pyramidal roof raised 2.5 m at the center. It is made of aluminum construction with several vents in the walls and four exhaust fans at the roof [50].

As a beginning of this project, Gutierrez-Montes et al. made an overview of smoke and fire dynamics in atria and enclosures including: atrium typology and definitions; atrium fire dynamics and safety problems; fire strategies and atrium design; computer modeling of atria; sprinklers and smoke detectors in atria; the process of smoke filling in atria; natural venting of smoke in atria; mechanical venting of smoke in atria; spill plume dynamics in atria; and design trends in fire safety in atria [51]. The first reports from the practical research in Murcia Fire Atrium are from 2008 with a numerical model and validation experiments of atrium enclosure fire [50]. Later, several articles were reported, including low and medium full-scale atrium fire tests and numerical validations of FDS followed by an experimental and numerical study of the smoke ventilation in atrium fires under dynamic ventilation performance where the validation of two fire experiments of 2.3 MW and 5.3 MW was made [52–55]. The latest full-scale fire experiments and simulations in Murcia under transient and asymmetric venting conditions were reported in 2016 with sources with different HRRs (1.7 MW, 2.3 MW, 3.9 MW, 5.3 MW) with a good agreement between the fire tests and the numerical set-up [56]. The latest computational research was reported by Zidek et al. in 2019 where fire tests in the Murcia Fire Atrium were used to validate FDS results. Three FDS cases were presented with the same HRR of 2.34 MW but different dimensions of the computational domain (mesh cell sizes). The results showed a good match with the real fire experiment in the atrium, except the plume region near the fire pool [57].

The current research aims not only to add some additional value to the atrium extraction system knowledge by comparing and validating traditional full-scale burning test with the FDS computational capacities but also to present a hybrid extraction system for atriums combining the advantages of traditional methods.

3. Smoke Extraction in the Example Atrium

The calculations for static and dynamic smoke extraction systems are presented in Appendix A and can be summarized as follows.

Following the Hong Kong guides [33], the minimum number of natural vents N_s in this example atrium, without having plugholing, is 12. Smoke must reach the nearest exhaust system without travelling more than 30 m [33]. A minimum of one smoke outlet must be provided in every 500 m^2 floor area. In order to satisfy both the distance to the smoke exhaust and the area served by one smoke outlet, a total of 16 natural vents were used. The area served by each smoke exhaust outlet was 50 m^2 . The positions of the vents are shown in Figure 1a.

For a dynamic smoke extraction system, the minimum number of mechanical vents N_D is also 12. The minimum volume flow rate for each mechanical vent V_{MV} is $33.33\text{ m}^3/\text{s}$.

According to the code [33], the minimum mechanical ventilation area S_{MV} is 4.17 m². Therefore, 12 mechanical vents were used in this example atrium to extract smoke. The mechanical ventilation vents are shown in Figure 1b.

A thick enough smoke layer has to be formed below the ceiling to have adequate buoyancy before opening the vents. The activation time can be determined by referring to the smoke layer interface height [29]. When the smoke layer interface height reaches about 30% to 50% of the ceiling height, the smoke extraction system should be activated [58]. The activation time of the system is determined from the HRR of the fire and other factors such as the sensitivity of the detector or activating mechanism and the threshold setting agreed on by the authorities.

The stability of the plume generated by a fire is affected by the make-up air inlets, even for a small make-up air inlet. This would affect the performance of natural venting systems. According to the suggestion by NFPA-92 [34], the average velocity of make-up air is 1 m/s in order to maintain satisfactory functioning of the whole system.

In general, the minimum supply or supplemental air rate must be 80% of the exhaust rate [33]. In using mechanical methods to supply air, the air must be extracted from the outside by a separate system or a general air conditioning system [33]. Therefore, the volume of supply air per unit time V_{SA} is 320 m³/s.

4. Numerical Simulation of Smoke Extraction in the Example Atrium

A steady fire with fixed size [18] has been used in designing smoke control systems in PBD-FEA projects for 30 years. The HRR is limited to a maximum value even for unsteady fire [34]. Smoke extraction systems can then be designed quickly but effectively by using a constant fire size. A 4 m by 4 m square pool fire was put at the center of the example atrium floor. Four groups of simulations, each with three cases of fire sizes commonly accepted in different applications (5 MW, 20 MW and 50 MW), for a specified scenario were considered in the example atrium with the model schematically shown in Figure 2.

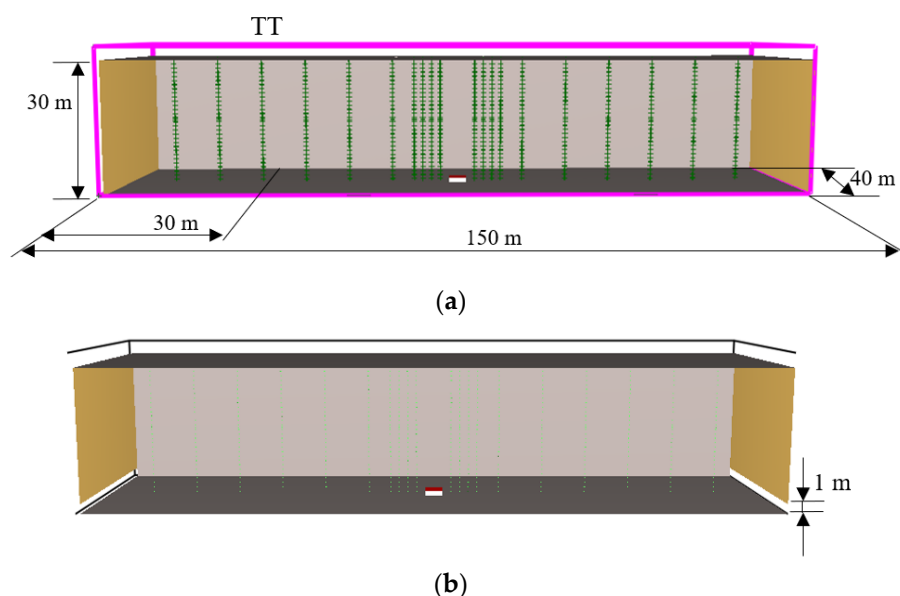


Figure 2. The example atrium. (a) Computational domain of the example atrium; (b) With make-up air inlets at the bottom.

The computational domain of the example atrium using FDS after preliminary grid size sensitivity study is shown in Figure 2a. The objective of the study is to compare the performance of different smoke exhaust systems. In this example design, two air inlets of 1 m height at the atrium floor were provided with the schematic diagram shown in Figure 2b.

The details of the four groups of simulations shown in Table 1 are:

- Group 1: Natural smoke filling without any ventilation
- Group 2: Static smoke extraction with natural vents
- Group 3: Dynamic smoke extraction with mechanical ventilation
- Group 4: Hybrid smoke extraction

Table 1. Summary of numerical simulation groups.

| Group | Smoke Extraction System | Case | Heat Release Rate/MW | Activation Time of Smoke Extraction System/s |
|-------|--|------|----------------------|--|
| 1 | Natural smoke filling | a | 5 | N/A |
| | | b | 20 | N/A |
| | | c | 50 | N/A |
| 2 | Static smoke extraction or natural ventilation | a | 5 | 150 |
| | | b | 20 | 50 |
| | | c | 50 | 30 |
| 3 | Dynamic smoke extraction or mechanical ventilation | a | 5 | 150 |
| | | b | 20 | 50 |
| | | c | 50 | 30 |
| 4 | Hybrid smoke extraction | a | 5 | 150 |
| | | b | 20 | 50 |
| | | c | 50 | 30 |

The simulation of the mechanical exhaust vent was made by setting a series of VENTs and designating a SURF_ID with specified volume flow rates in the FDS input file. This fan modeling method has also been adopted in much of the previous research. From the perspective of scientific issues, the change rule of the mechanical smoke extraction with the actual air volume at the smoke vent was studied. The simulation of the natural vent was performed by setting a series of OPENs in the FDS input file with a certain area, which meant pressure outlets from inside of the atrium to the atmosphere through the opening area.

The initial temperature of both inside and outside of the example atrium was 20 °C, and the initial pressure of both inside and outside was 101,325 Pa. No wind was set in the simulation.

The two main parameters used to validate the simulation results are the transient vertical temperature distributions and the descending time of the smoke layer. The smoke layer interface height is an indication of the performance of smoke extraction system and determined as described later. If the smoke layer interface height can be kept at a certain value, occupants are able to escape safely.

The values of the two main parameters can be directly obtained by CFD-FDS simulation. The smoke interface height at Position TT outside the plume for convenient determination of smoke layer interface and the temperature under the atrium ceiling at Position TT are selected as the typical value for analysis, which is 30 m away from the left side and about 40 m away from the fire source edge, as shown in Figure 2a.

Smoke layer interface height is obtained by simulation using a steady fire with a constant HRR. In order to get a certain smoke layer thickness without plugholing during smoke extraction, the activation times of vents are 150 s, 50 s and 30 s for 5 MW, 20 MW and 50 MW fires, respectively. The simulation time of each case is 500 s.

Group 1 simulations on natural smoke filling are taken as the reference. Smoke almost filled up the whole space after 300 s. The smoke layer interface height of the three fire sizes is shown in Figure 3a and the smoke temperature rise is presented in Figure 3b.

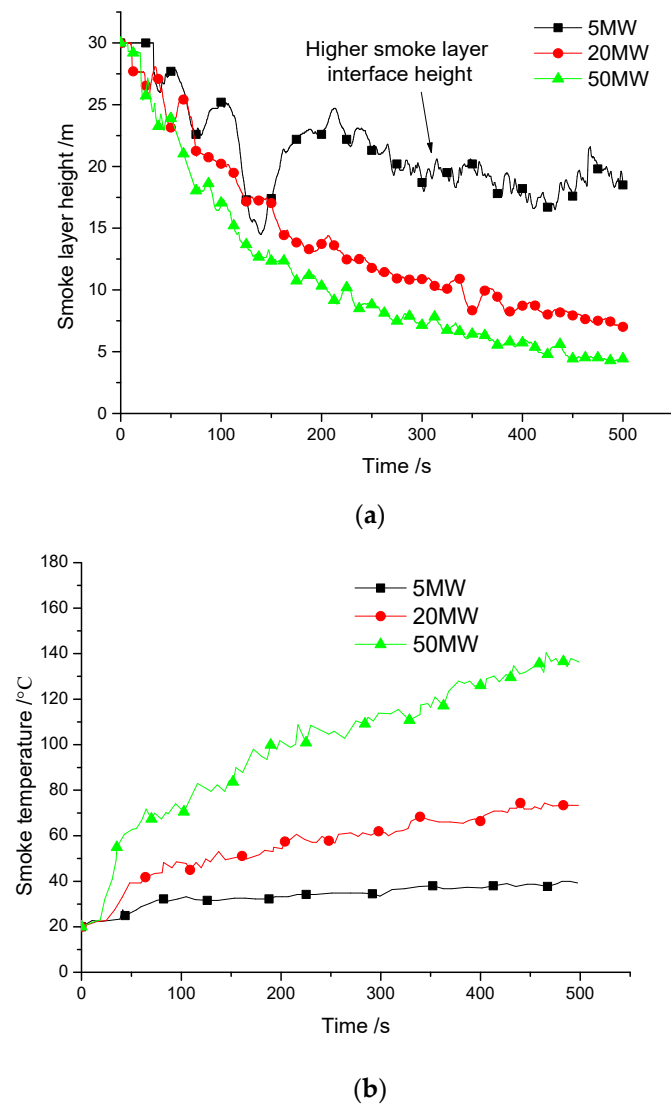


Figure 3. Natural smoke filling in the example atrium. (a) Smoke layer interface height at TT; (b) Smoke temperature at TT.

The N-percentage Rule [59,60] was applied to the vertical temperature profile obtained at different appropriate locations. The vertical temperature profile contains temperatures at heights y_1, y_2, \dots, y_m with y_1 at the lowest level, and y_m at the top. The temperatures predicted at time t are $T(y_1, t), T(y_2, t), \dots, T(y_m, t)$. A reference upper layer temperature $\Delta T_{Ref}(t)$ at t is determined within the burning duration t_b .

$$\Delta T_{Ref}(t) = \text{Max}[T(y_m, t)] - T(y_m, 0) \quad (1)$$

$\text{Max}[T(y_m, t)]$ is the maximum value of $T(y_m, t)$ in the interval from 0 to t and $T(y_m, 0)$ is the initial temperature at y_m with values very close to the ambient value T_0 .

The smoke layer interface y is determined by the N-percentage rule to be the value of height y_1, \dots, y_m satisfying the following:

$$T(y, t) - T(y, 0) = \frac{N}{100} \Delta T_{Ref}(t) \quad (2)$$

N was commonly taken to be 10, 15 and 20 [41,58]. In this study, the value of N was taken to be 20 from 100 s to 400 s and 25 for 500 s. The predicted vertical temperature

profile was used to determine the hot air layer height. Air temperatures at every 1 m interval height at the $x = 30$ m in the atrium are shown in Figure 4a–e.

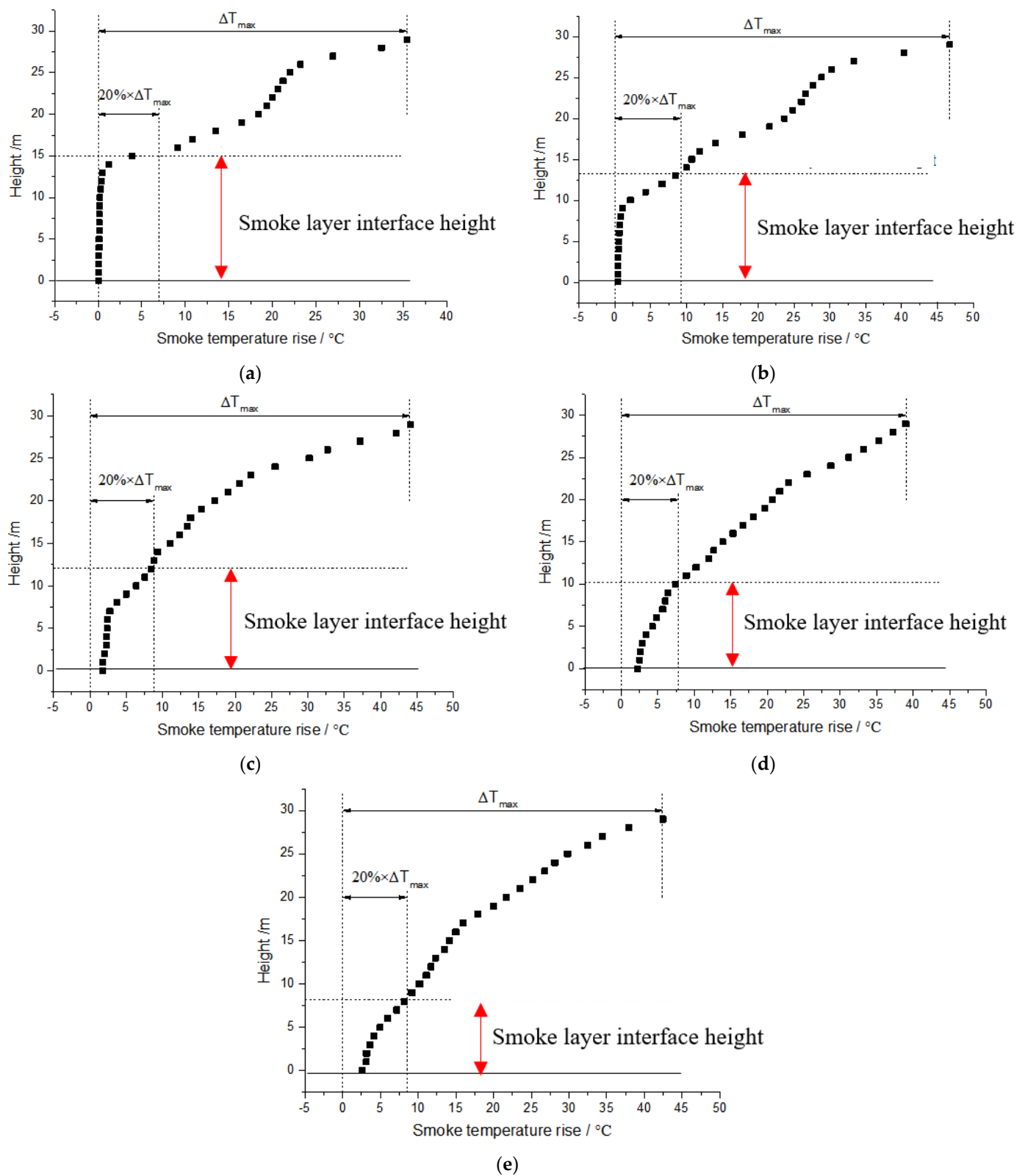


Figure 4. Vertical temperature profiles and N-percentage rule. (a) 100 s; (b) 200 s; (c) 300 s; (d) 400 s; (e) 500 s.

The performance of natural vents with and without make-up air is shown in Figure 5a–c. It is observed that a stable smoke layer was not formed with natural smoke vent under small fire without make-up air.

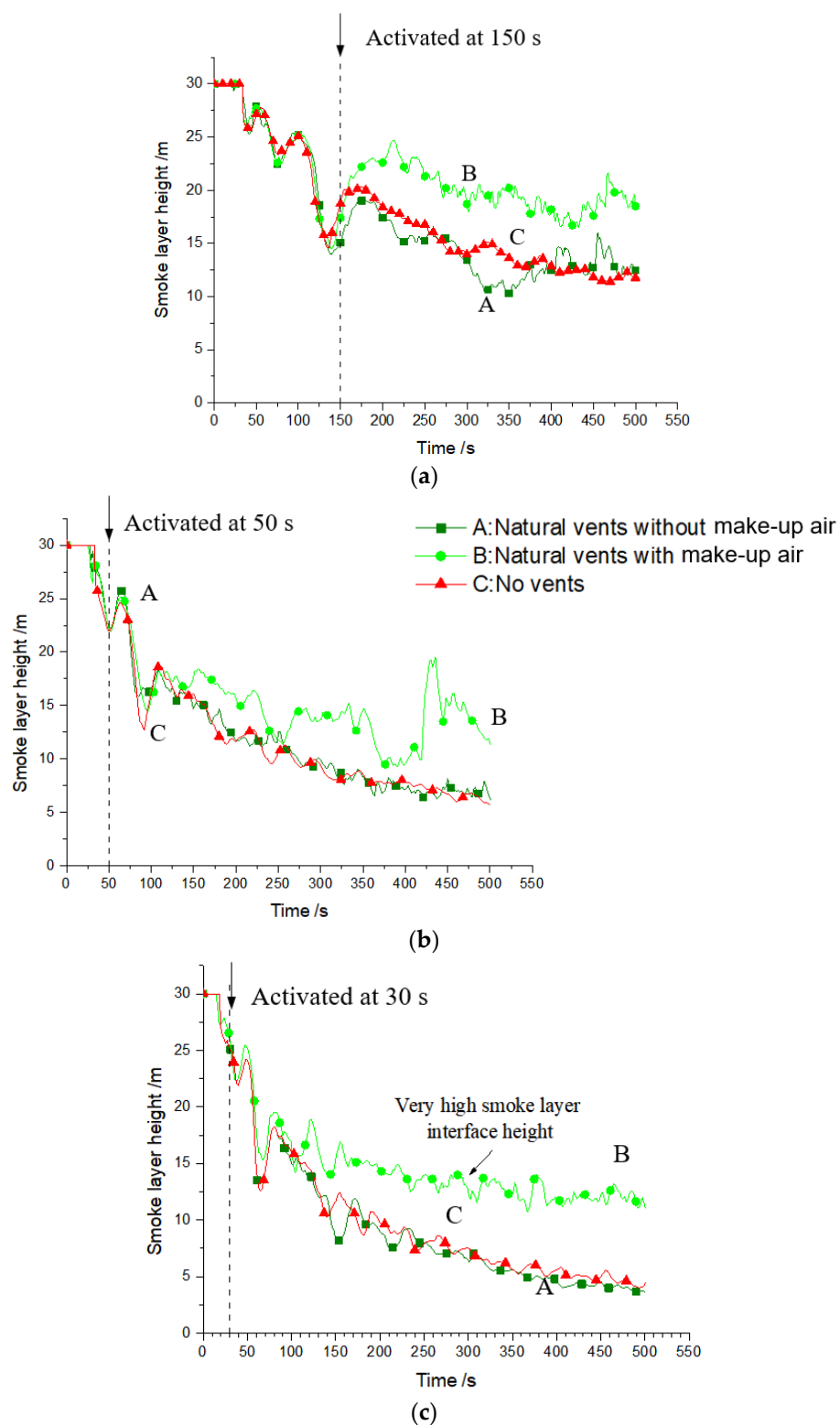


Figure 5. Smoke layer interface height at TT with and without make-up air. (a) Small 5 MW fire; (b) Medium 20 MW; (c) Big 50 MW fire.

Natural vent was not able to take smoke away from the atrium under medium and big fires when there was no make-up air. Air circulation was blocked. Without fresh air makeup from outside, the whole atrium cannot remain at a positive pressure relative to outside, so that smoke could not move out. Therefore, supplying adequate air below the smoke layer is very important to extract smoke. Otherwise, natural venting systems would not work effectively.

5. Simulation Results for the Example Atrium

The performance of the two types of smoke exhaust systems is evaluated by simulating the smoke environment inside the example hall. The performance of a ventilation system is determined by the stable smoke layer interface height. Simulation result based on a series of temperature thermocouples at a vertical plane labeled TT, which are 30 m away from the left side and about 40 m away from the fire source edge, as shown in Figure 2a.

5.1. Group 1 Simulation with Natural Smoke Filling

- Case 1a with 5 MW Fire

The smoke layer interface height dropped to about 10 m above the floor at around 460 s, and the whole space was filled up by smoke and the temperature increased to 50 °C under the situation of no openings.

- Case 1b with 20 MW fire

The smoke layer interface height dropped to about 5 m above the floor, and the whole space was filled up by smoke and the temperature increased to 80 °C under the situation of no openings.

- Case 1c with 50 MW fire

The smoke layer interface height dropped to about 2 m above the floor and the whole space was filled up by smoke and the temperature increased to 140 °C under the situation of no openings.

5.2. Group 2 with Static Smoke Extraction

- Case 2a with 5 MW Fire

The performance of natural vents was not satisfactory. The smoke layer interface height remained steady at 13 m at 460 s.

- Case 2b with 20 MW fire

The smoke layer interface height remained steady at about 12 m high. The natural venting system showed better performance than mechanical vents.

- Case 2c with 50 MW fire

The smoke layer interface height remained steady at about 12 m high. The natural venting system showed best capability of smoke control in the big fire than the other two smoke exhaust systems.

5.3. Group 3 with Dynamic Smoke Extraction

- Case 3a with 5 MW Fire

The smoke layer interface height remained at the 13 m level after 500 s by mechanical vents.

- Case 3b with 20 MW fire

The smoke layer interface height remained at 11 m high by mechanical venting system.

- Case 3c with 50 MW fire

The smoke layer interface height remained at 8 m high by mechanical venting system.

The curves of the smoke layer interface height by the two smoke exhaust systems compared with smoke natural filling are shown in Figure 6a–c for smoke layer interface

height at Position TT. As these four curves show, the smoke exhaust capacity of mechanical smoke exhaust is best when the fire is relatively small. This is because the buoyancy of the smoke generated by the small fire is not strong enough, which would lead to an insufficient pressure difference for the natural smoke exhaust to work well.

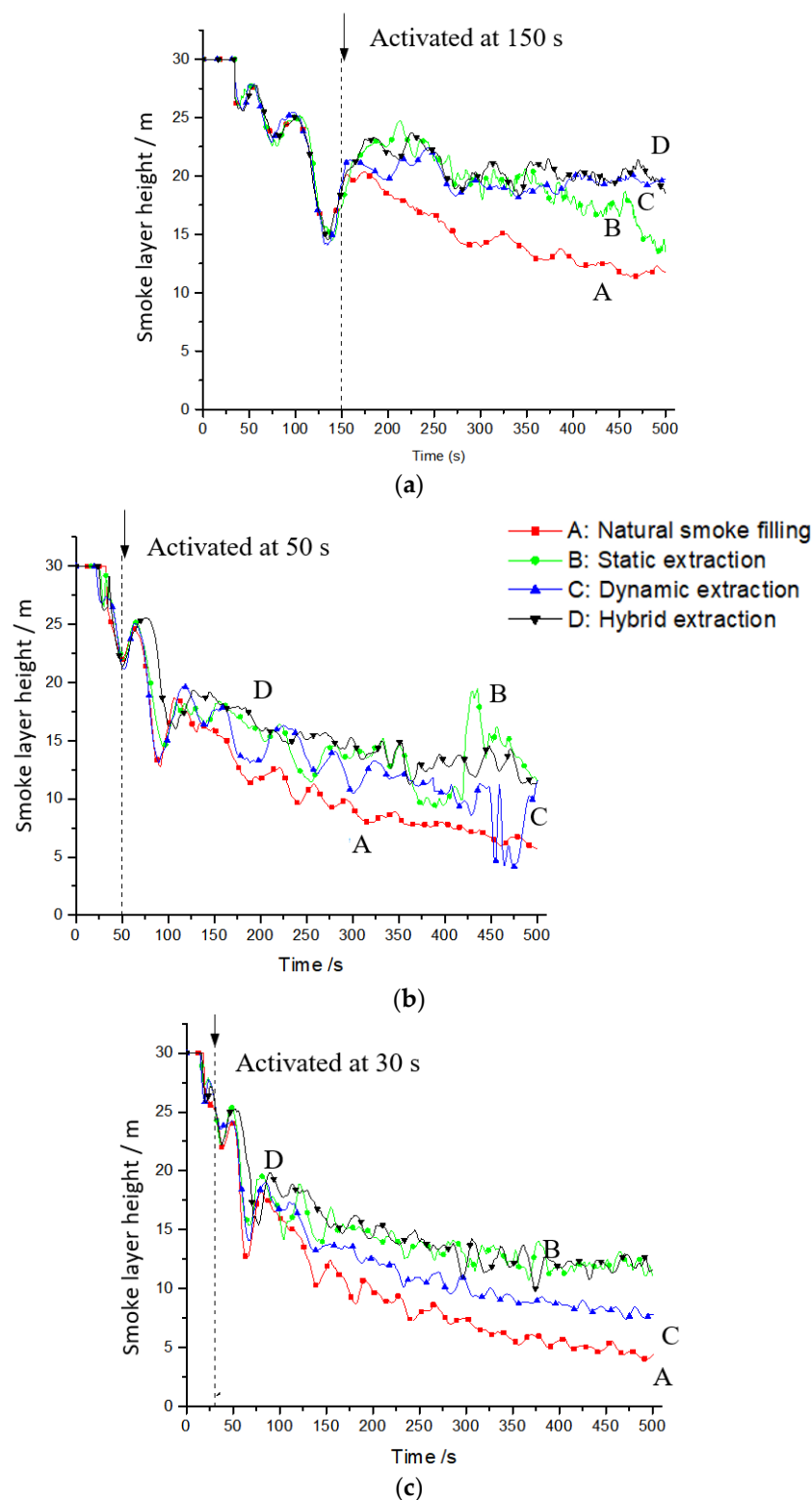


Figure 6. Smoke layer interface height at TT. (a) Small 5 MW fire; (b) Medium 20 MW fire; (c) Big 50 MW fire.

In medium fire, the performance of the three types of smoke exhaust systems is similar. Natural smoke exhaust is not good for small fires, and mechanical smoke exhaust is not good for big fires.

In the big fire cases, as seen by the results, the static and hybrid extraction systems have similar performance and are much better than the dynamic system. The results again show that the natural smoke exhaust systems are better for big fires.

6. New Hybrid Design for Sustainable Smoke Extraction

It is clear from the above results that a static smoke extraction system with natural vent is not able to take out smoke from small fires. Similarly, a dynamic smoke extraction system with mechanical ventilation is not good for big fires. A hybrid design that combines the two is proposed to produce a sustainable and safe atrium. Half of the volume of smoke is extracted by natural vents, and the other half is extracted by mechanical vents.

The calculations for the proposed hybrid system are presented in Appendix B and can be summarized as follows.

For natural vents, a smoke extraction rate V_{NV1} of 200 m³/s. The area of the vent S_{NV1} is 400 m². The number of natural vents N_1 in the new design is six. In order to satisfy both the distance to smoke exhaust and the area served by one smoke outlet, eight natural vents were used to extract smoke by static system. The area of each natural vent inlet is 50 m².

For dynamic smoke extraction, the volume flow rate V_{MV1} for the new design is 200 m³/s. The number of mechanical vents N_{MV1} is six. The mechanical vent area for each vent S_{MV1} is 25 m². The volume flow rate for each mechanical vent S_{MV1} is 4.17 m³/s.

Eight vents for the natural venting system and another six vents for the mechanical venting system are used with the schematic shown in Figure 1c.

The last group (Group 4) of simulations was carried out for this hybrid design under the fire sizes of 5 MW, 20 MW and 50 MW. The transient smoke layer interface heights are also plotted in Figure 6 together with the Group 2 and 3 predictions.

- Case 4a with 5 MW fire

The performance of the hybrid system is similar to the other two systems with the smoke layer interface heights lying between those two curves.

- Case 4b with 20 MW fire

The performance of the hybrid system is efficient to keep smoke layer at high level.

- Case 4c with 50 MW fire

The performance of the hybrid system can also keep smoke layer at high level.

The static smoke extraction system performs the best for a large fire. The driving force due to buoyancy is strong enough to give a sufficient pressure differential between the vents.

7. Validation of CFD-FDS Simulation Results

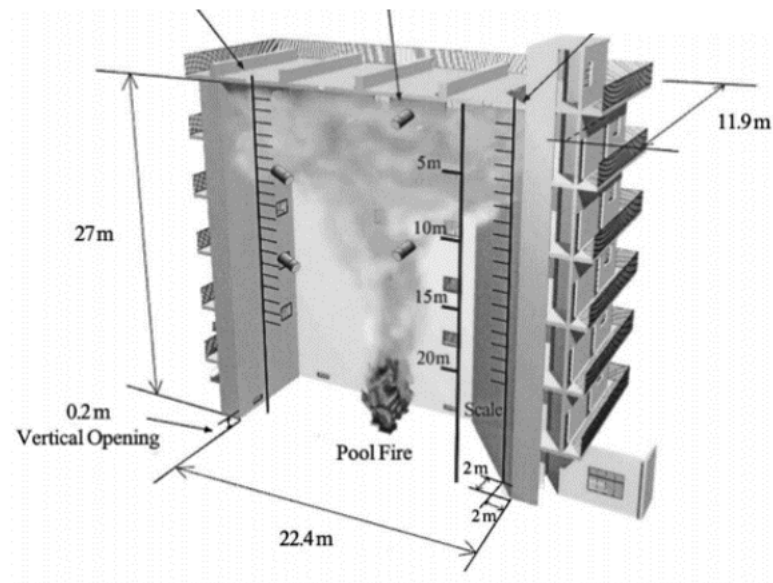
Smoke filling and extraction simulations by CFD-FDS [29,37] were validated by two cases.

7.1. PolyU/USTC Atrium

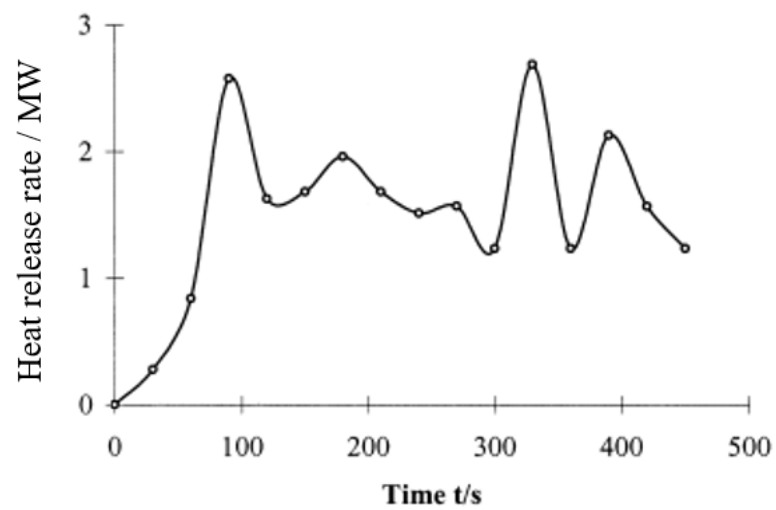
Experimental studies on natural smoke filling in the PolyU/USTC Atrium with liquid pool fires with HRR up to 1.6 MW were conducted in 2000 [39].

The geometry of the PolyU/USTC Atrium is shown in Figure 7a with details explained elsewhere [39,40]. The internal dimensions of the PolyU/USTC Atrium are 22.4 m long, 11.9 m wide and 27 m tall.

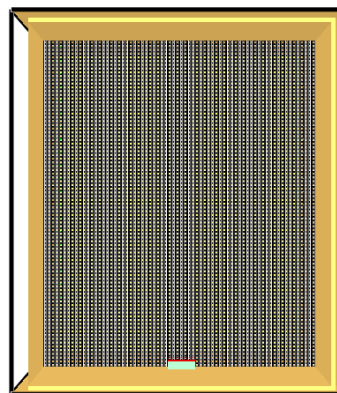
Thermocouple TA1



(a)



(b)



(c)

Figure 7. The PolyU/USTC Atrium [39]. (a) Schematic; (b) HRR of pool fires [2]; (c) Schematic of FDS input.

The experimental conditions were as follows: closed ceiling vents and side windows of the atrium with a gap of 20 cm high at the bottom of the wall. The HRR was an input parameter from the experimental results. The curve of HRR is shown in Figure 7b. From the curve, the pool fires were similar to an unsteady t^2 -fire at the beginning and then became a steady fire. Two grid systems were used: coarse at $0.4\text{ m} \times 0.4\text{ m} \times 0.4\text{ m}$ and fine at $0.2\text{ m} \times 0.2\text{ m} \times 0.2\text{ m}$. The modeling schematic is shown in Figure 7c with a grid system of $0.4\text{ m} \times 0.4\text{ m} \times 0.4\text{ m}$.

A lot of data obtained using CFD-FDS is shown in the following charts. The maximum smoke temperature rise ΔT_{max} was up to $45\text{ }^{\circ}\text{C}$, which is over twice that in the experimental results (see Figure 8). Values of smoke layer interface height by simulation are close to experimental results, which are clearly shown in the curves in Figure 9. In order to save computer time, the simulation duration of ‘fine grid’ is made shorter.

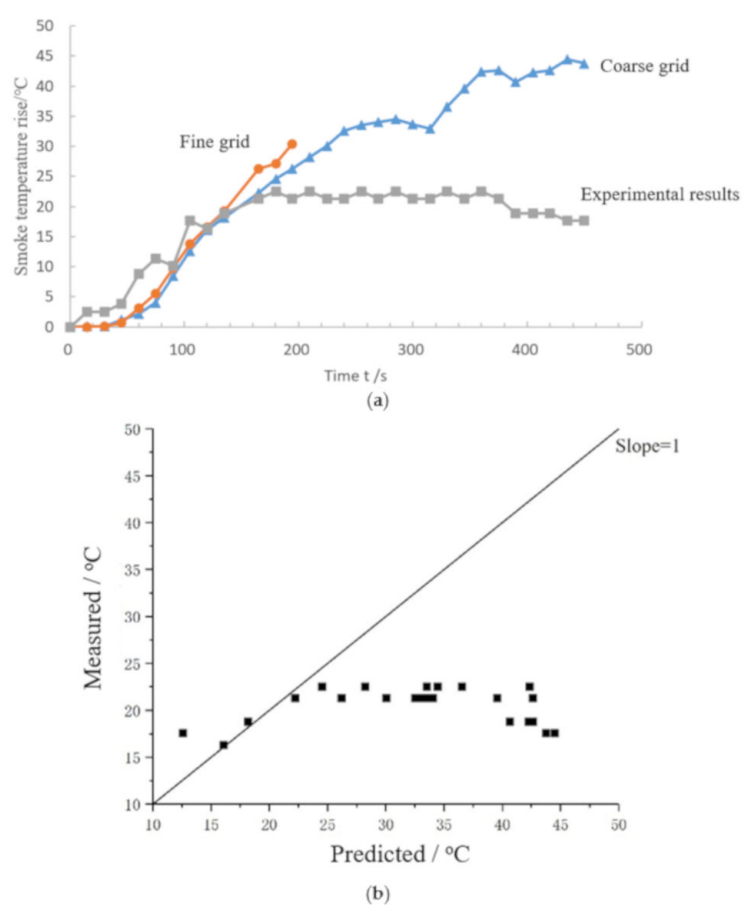


Figure 8. Results for maximum smoke temperature rise for PolyU/USTC Atrium. (a) Transient smoke temperature rise; (b) Predicted smoke temperatures rise for coarse grid.

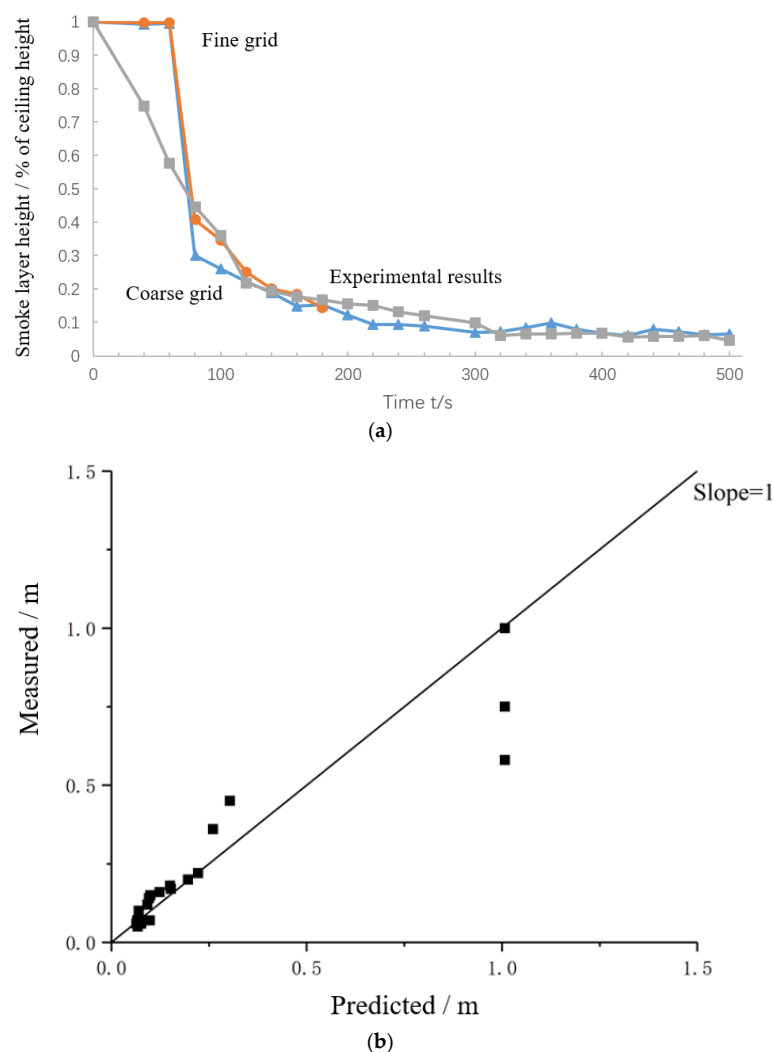


Figure 9. Smoke layer interface heights for PolyU/USTC Atrium. (a) Transient smoke layer interface height; (b) Predicted smoke layer interface heights for coarse grid.

As the simulation results of the PolyU/USTC atrium show, the predicted smoke temperature rise is a little bit higher than the experimental results, but the smoke layer interface height agrees very well with the measured data. By comparison with experimental results, the results of coarse grid are good enough for prediction.

7.2. Irregular Atrium in Hong Kong

Another validation case is an atrium in Hong Kong reported earlier in 2007 [40,57].

An irregular atrium geometry shown in Figure 10a is considered. Its floor area is 35 m by 9 m, with a ceiling height of 28 m. The atrium is equipped with a mechanical smoke exhaust system with a designed volumetric flow rate of 47.8 m³/s. Hot smoke tests were carried out [41,58].

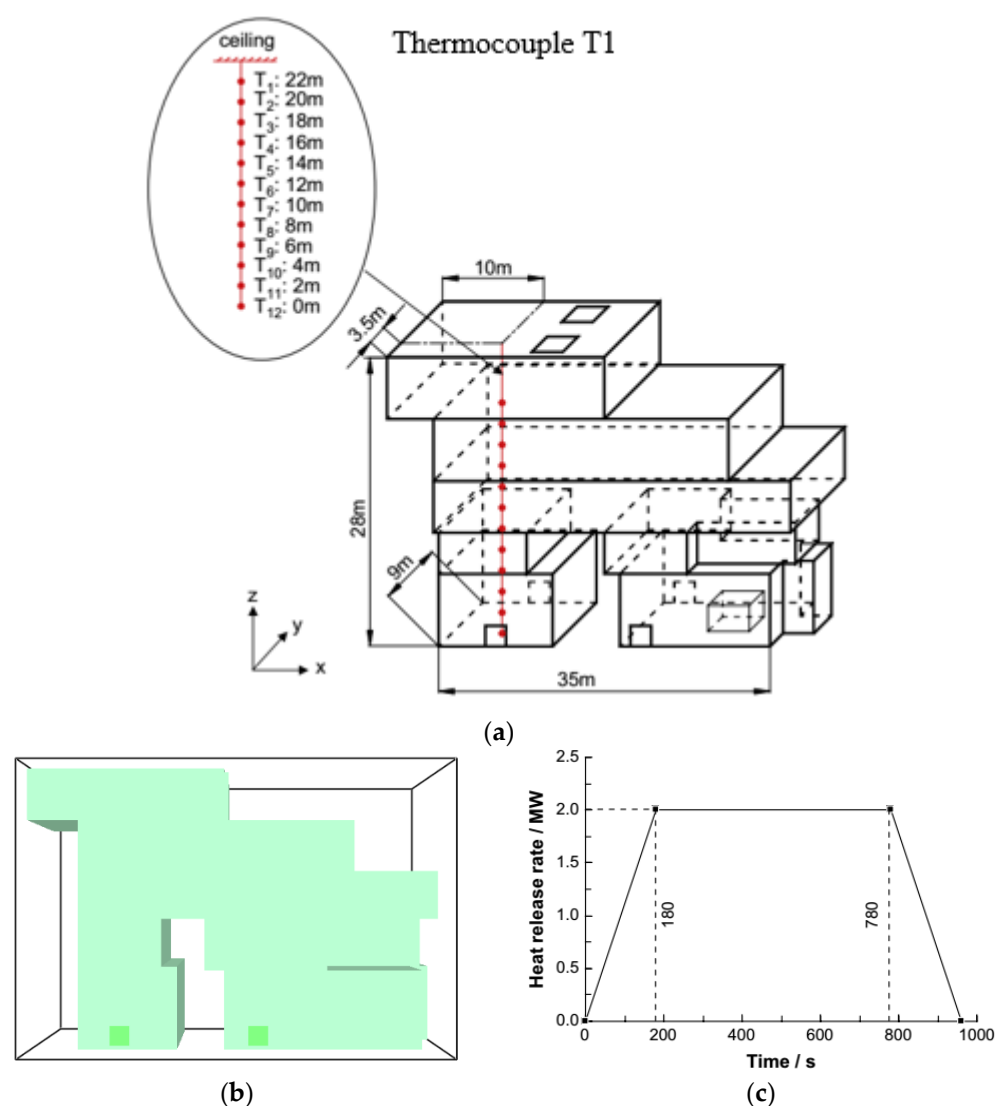


Figure 10. Irregular atrium in Hong Kong [58]. (a) Geometry; (b) Input schematic drawing; (c) Input HRR.

The schematic of CFD-FDS input and HRR are shown in Figure 10c. Figure 10b shows the grid size $0.5\text{ m} \times 0.5\text{ m} \times 0.5\text{ m}$. The smoke temperature was recorded at 12 positions as in the experiment. From the vertical temperature profile, the smoke layer interface height was determined. The predicted and measured results are plotted in Figure 11a,b.

As shown in the plots of smoke temperature and smoke layer interface height, most points are near to the line with slope = 1, meaning that the simulation results agree well with the experimental results.

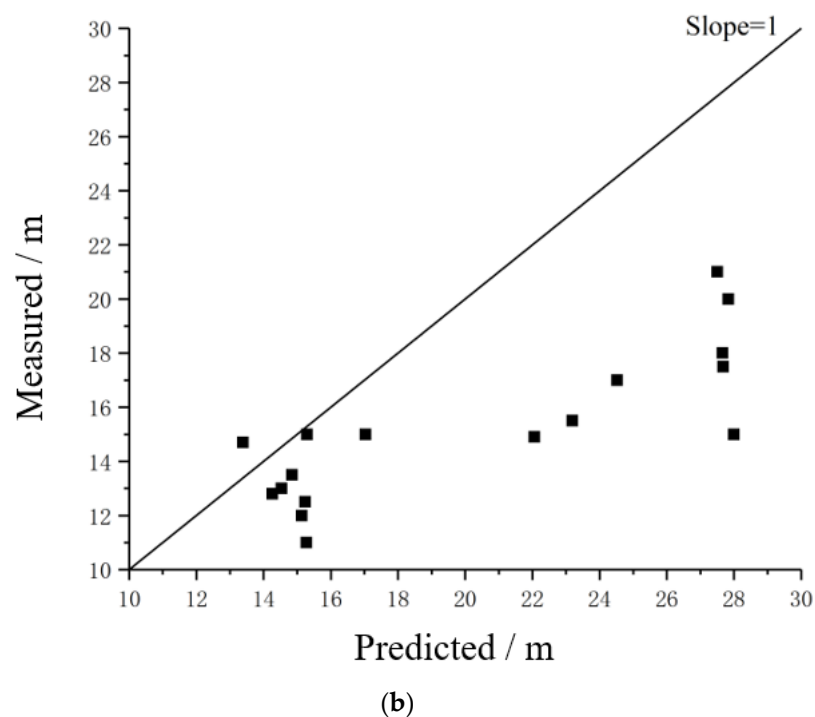
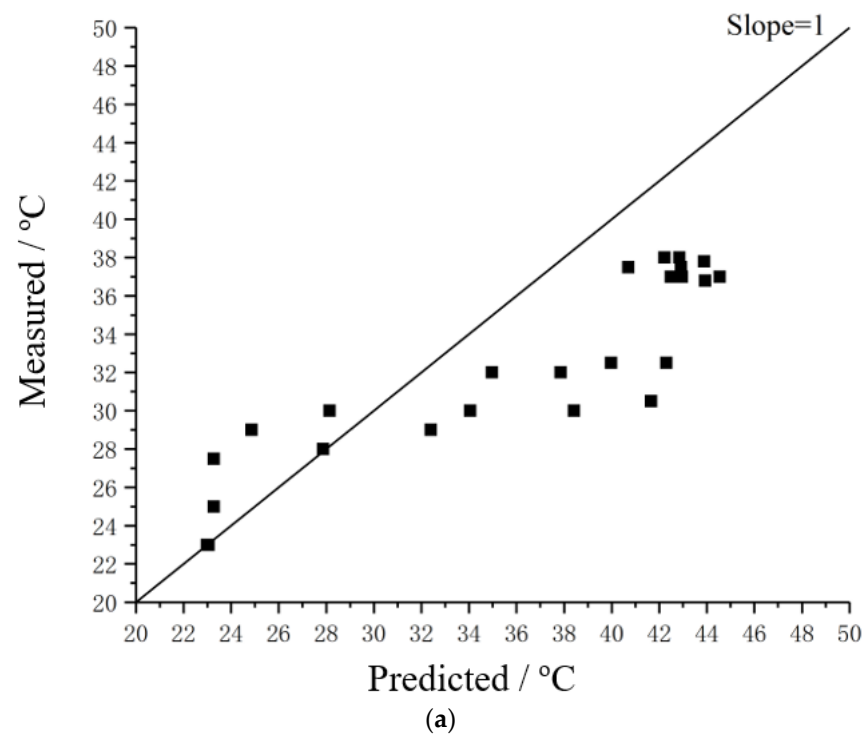


Figure 11. Predicted results for the irregular atrium in Hong Kong. (a) Predicted temperatures at T1; (b) Predicted smoke layer interface heights.

It is observed that CFD-FDS can give fairly reasonable predictions in the two atria when compared with the experimental data. The input parameters and geometry are very important to obtain accurate results using CFD-FDS.

8. Discussion

Buoyancy is demonstrated by the above CFD-FDS predictions to be important in removing smoke by static smoke extraction systems. The static extraction system is thus more appropriate for a fire with a large HRR. A large fire can produce enough buoyancy to form a thick enough hot smoke layer. For a small fire with low HRR at the initial stage, a dynamic smoke extraction system performs better. The fan exhaust rate is a key part in designing a dynamic smoke extraction system.

For natural smoke filling without any smoke exhaust systems, smoke spreads in the whole space rapidly, producing a threat to the people inside. A static smoke exhaust system can extract smoke for bigger fires with higher HRR due to strong buoyancy. The static smoke extraction system performs well in removing smoke away from the hall with large volume space, as indicated in the third case of Group 3 with a 50 MW fire. The bigger the fire size, the bigger the air entrainment rate to the plume and hence the higher the exhaust rate required for a dynamic smoke extraction system. However, for smaller fires with lower HRR, buoyancy is insufficient to give a stable smoke layer. A static smoke extraction system with natural vents cannot perform well in keeping a stable smoke layer height. Therefore, dynamic smoke extraction with mechanical ventilation is needed to remove smoke, as shown in the CFD-FDS predictions for a 5 MW fire.

For the hybrid system combining the two extraction systems as proposed by numerical simulation, the operation times are important. The volume of smoke extracted by the mechanical fan might not be able to meet the requirement in removing smoke from the entire hall space. However, combining the static smoke vent will be effective. This will not generate solid waste in replacing the active system.

In summary, the proposed hybrid design will produce a sustainable smoke extraction system that uses less electrical power in operating the fan and is capable of keeping the smoke layer high under both small and big fires.

9. Conclusions

The performance of static and dynamic smoke extraction systems in an example atrium were studied using CFD-FDS in this paper. The objective was to compare the efficacy of mechanical ventilation and natural ventilation for small, medium and large fires. The CFD-FDS simulations on natural smoke filling, static and dynamic smoke extractions with three design fires of 5 MW, 20 MW and 50 MW confirmed that static smoke extraction is good for big fires but not good for taking smoke out under the initial stage of small fires. A dynamic smoke extraction system performs well for small fires, but not for big fires. A new hybrid design combining static and dynamic systems is proposed to produce a sustainable and safe atrium and is expected to perform well in both big and small fires.

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Appendix A. Calculations for Static and Dynamic Smoke Extraction Systems

A value of 8 air changes per hour is specified in Hong Kong. The smoke extraction rate V_{smoke} in the example atrium is given by:

$$V_{smoke} = 150 \times 40 \times 30 \times 8 \text{ m}^3/\text{h} = 1,440,000 \text{ m}^3/\text{h} = 400 \text{ m}^3/\text{s} \quad (\text{A1})$$

For static smoke extraction system with natural vents, taking the average velocity at natural vents v_{NV} on the ceiling to be 0.5 m/s, the minimum venting area S_{NV} is:

$$S_{NV} = \frac{V_{smoke}}{v_{NV}} = \frac{400}{0.5} \text{ m}^2 = 800 \text{ m}^2 \quad (\text{A2})$$

The minimum number of natural vents N_s in this example atrium, without plugholing, is:

$$N_s = \frac{150 \times 40}{500} = 12 \quad (\text{A3})$$

For dynamic smoke extraction system, the minimum number of mechanical vents N_D is as follows:

$$N_D = \frac{150 \times 40}{500} = 12 \quad (\text{A4})$$

The minimum volume flow rate for each mechanical vent V_{MV} is:

$$V_{MV} = \frac{V_{smoke}}{N_D} = \frac{400}{12} \text{ m}^3/\text{s} = 33.33 \text{ m}^3/\text{s} \quad (\text{A5})$$

According to the codes, for efficient extraction, the maximum velocity of mechanical vents on the ceiling is above 8 m/s. Thus, the minimum mechanical ventilation area S_{MV} is:

$$S_{MV} = \frac{V_{MV}}{V_D} = \frac{33.33}{8} \text{ m}^2 = 4.17 \text{ m}^2 \quad (\text{A6})$$

The volume of supply air per unit time V_{SA} is as follows:

$$V_{SA} \geq 1,440,000 \times 0.8 \text{ m}^3/\text{h} = 1,152,000 \text{ m}^3/\text{h} = 320 \text{ m}^3/\text{s} \quad (\text{A7})$$

Appendix B. Calculations for the Proposed Hybrid System

For natural vents, the smoke extraction rate V_{NV1} is given by:

$$V_{NV1} = V_{smoke}/2 = 400/2 \text{ m}^3/\text{s} = 200 \text{ m}^3/\text{s} \quad (\text{A8})$$

The area of vent S_{NV1} is:

$$S_{NV1} = \frac{V_{smoke}}{v_{NV1}} \geq \frac{200}{0.5} = 400 \text{ m}^2 \quad (\text{A9})$$

The number of natural vents N_1 in the new design is:

$$N_1 \geq \frac{75 \times 40}{500} = 6 \quad (\text{A10})$$

For dynamic smoke extraction, the volume flow rate V_{MV1} for the new design is:

$$V_{MV1} = V_{smoke}/2 = 400/2 \text{ m}^3/\text{s} = 200 \text{ m}^3/\text{s} \quad (\text{A11})$$

The number of mechanical vents N_{MV1} is as follows:

$$N_{MV1} \geq \frac{75 \times 40}{500} = 6 \quad (\text{A12})$$

The mechanical vent area for each vent S_{MV1} is:

$$S_{MV1} = \frac{V_{smoke}}{V_{MV1}} \geq \frac{200}{8} \text{ m}^2 = 25 \text{ m}^2 \quad (\text{A13})$$

The volume flow rate for each mechanical vent S_{MV1} is:

$$S_{MV1} = \frac{S_{MV1}}{N_{MV1}} = \frac{25}{6} \text{ m}^3/\text{s} = 4.17 \text{ m}^3/\text{s} \quad (\text{A14})$$

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